

Modeling Triple Friction Pendulum Bearings in Program OpenSees Including Frictional Heating Effects

by Hyun-Myung Kim and Michael C. Constantinou



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by

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Preface

MCEER is a national center of excellence dedicated to the discovery and development of new knowledge, tools and technologies that equip communities to become more disaster resilient in the face of earthquakes and other extreme events. MCEER accomplishes this through a system of multidisciplinary, multi-hazard research, in tandem with complimentary education and outreach initiatives.

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The Center derives support from several Federal agencies, including the National Science Foundation, Federal Highway Administration, Department of Energy, Nuclear Regulatory Commission, and the State of New York, foreign governments and private industry.

This report presents and verifies a revised model of the behavior of triple friction pendulum bearings that explicitly computes the sliding displacements, sliding velocities and the rise in temperature at its four sliding interfaces. This permits consideration of the effects of velocity and temperature on the coefficients of friction. The model is based on the earlier model of Fenz and Constantinou, which did not explicitly compute the conditions at each sliding interface but correctly predicted the global force-displacement relationship of the bearing. The model was implemented in program OpenSees and interested users may obtain the source code from the authors.

ABSTRACT

A Triple FP bearing element currently available in program OpenSees is based on the series model, which consists of properly combined hysteretic/frictional and gap elements. The model can account for the dependency of the friction coefficient on axial pressure and on sliding velocity using modified parameters without explicitly calculating the velocity at each sliding surface. The model cannot be directly used to account for frictional heating effects since for those calculations, the histories of velocity and displacement on each sliding surface are needed.

This report presents a new triple FP element in OpenSees that is based on the existing element in program OpenSees but modified to account for the effect of frictional heating on the friction coefficient. To accomplish this, the displacement and velocity of the top of the bearing with respect to its bottom computed by the element are partitioned into components at each sliding surface. This partitioning is based on two different procedures: (a) modification of the histories of displacements and velocities computed in the three individual FP elements in the series model of the bearing, and (b) retracing the histories of displacements and velocities based on the force-displacement relationship of the theory of Fenz and Constantinou.

Verification of the modified Triple FP element is performed by computing force-displacement loops and histories of displacement, velocity, and temperature at each sliding surface for two configurations of the bearing under various imposed sinusoidal displacements of the top of the bearing and ground motions and comparing it with results obtained by the program 3pleANI.

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SECTION 1 INTRODUCTION

The behavior of Triple Friction PendulumTM (FP) bearings has been previously described by many of which the works of Fenz and Constantinou (2008a to d, 2009), Dao et al. (2013), and Sarlis and Constantinou (2013, 2016) are most relevant to this report. Figure 1-1 shows the geometry of the Triple FP bearing and its parameters. R_1, R_2, R_3 and R_4 are radii of curvature; h_1, h_2, h_3 and h_4 are distances of the slide plate to the pivot point(O); d_1, d_2, d_3 and d_4 are the nominal displacement capacities; b_1, b_2 , and b_3 are diameters of the rigid slider and the two inner slide plates; and μ_1, μ_2, μ_3 and μ_4 are the friction coefficients of the four sliding interfaces. Figure 1-1 shows the Triple FP isolator with an inner ring (or displacement restrainer) which is not needed, and modern versions of the isolator do not have it. In those cases, the displacement capacities d_2 and d_3 are larger than the nominal capacities shown in Figure 1-1.



Figure 1-1 Geometry of the Triple FP Bearing (Sarlis and Constantinou, 2013)

The behavior of Triple FP bearing has been described in Fenz and Constantinou (2008a to 2008d, 2009) and summarized in Constantinou et al. (2011). The following parameters determine the response of Triple FP bearing: friction coefficients at its four sliding interfaces, μ_i , nominal displacement capacities, d_i , radii of curvature R_i , and heights h_i , where i = 1 to 4. The effective radii of curvature are defined as $R_{effi} = R_i - h_i$, and the actual displacement capacities are given by $d_i^* = \frac{R_{effi}}{R_i} d_i$. When the inner ring is absent, the actual total displacement is increased by $b_2/2$, at which point the isolator becomes unstable (Sarlis and Constantinou, 2013)

In the Fenz and Constantinou (2008a, c) model, the following conditions need to apply: 1) $R_{eff1} = R_{eff2} \gg R_{eff2} = R_{eff3}$, 2) $\mu_2 = \mu_3 < \mu_1 < \mu_4$, 3) $d_1^* > (\mu_4 - \mu_1)R_{eff1}$, 4) $d_2^* > (\mu_1 - \mu_2)R_{eff2}$, 5) $d_3^* > (\mu_4 - \mu_3)R_{eff3}$. These conditions typically apply for practical Triple FP bearings. The more complex theory of Sarlis and Constantinou (2013, 2016) is more general and is not restricted by these conditions. It can also explicitly calculate the state of motion at each of the four sliding surfaces of the bearing.

A computational model for the Triple FP element was developed initially by Fenz and Constantinou (2008d, 2009), and later Sarlis and Constantinou (2010) presented details of modeling the bearing in program SAP2000 (CSI, 2011) in a report to the engineering community. Dao et al. (2013) implemented the same model in the program OpenSees (McKenna et al., 2010), having improved on the use of multi-directional gap elements, whereas the model in SAP2000 required the use of several one-directional gap elements. Sarlis and Constantinou (2013) then developed many advanced theories for the behavior of the Triple FP bearing, which were not bound by the conditions of the Fenz and Constantinou (2008a) theory. They implemented these theories in computer program 3pleANI (Sarlis and Constantinou, 2013). These theories can produce histories of displacement, velocity and temperature at each of the sliding interfaces of the bearing, can account for pressure, velocity and temperature dependency of the friction coefficient of each sliding interface, and can model uplift and collapse of the bearing. Program 3pleANI was developed for motion in one horizontal and the vertical directions. It cannot be used for simulation of behavior or response history analysis under triaxial motion.

There is no element in any commercial or open-source software capable of simulation of the triaxial behavior of the Triple FP bearing with due consideration for the dependency of friction on the temperature at each sliding interface. Such a model is needed when performing simulations with significant heating effects, as in the case of long-duration ground motions. The only available element that can do is for the simpler Single FP bearing element FPBearingPTV in OpenSees (Kumar et al., 2015).

This work reports on a modification of the current Triple FP element in OpenSees (Dao et al., 2013) to account for the triaxial behavior of the Triple FP bearing with due consideration for the dependency of friction on the temperature, velocity, and pressure at each sliding interface. Program 3pleANI is used in this work for verification of the model implemented in program OpenSees.

SECTION 2

MODELING THE BEHAVIOR OF TRIPLE FRICTION PENDULUM BEARING

2.1 Model of Triple FP Bearing in Program OpenSees by Dao et al. (2013)

Figure 2-1 depicts the series model for the Triple FP bearing that was implemented by Dao et al. (2013) in the program OpenSees (McKenna et al., 2010). In the numerical model, k_{21} , k_{23} and k_{25} are normalized stiffnesses due to the curvature (inverse of pendulum length). Quantities k_{11} , k_{13} and k_{15} are high initial stiffnesses with small yield displacement to account for frictional behavior as shown in Figure 2-2. Also, k_2 , k_4 and k_6 are stiffnesses, with large values, of circular gap elements which are needed to represent the behavior of the element when there is contact with any of restrainers of the bearing. The resulting backbone curve is shown in Figure 2-3. The model is identical to the series model of Fenz and Constantinou (2008d, 2009) but for the nomenclature used: L_1 for effective radii $R_{eff2}=R_{eff3}$, L_2 for effective radius R_{eff4} , and L_3 for effective radius R_{eff4} . This may be realized by comparing Figure 3-1 in Fenz and Constantinou (2008d) to Figure 2-1 below.



Figure 2-1 Series model and Numerical model for Triple FP bearing (Dao et al., 2013)



Figure 2-2 Modeling Friction Behavior (Dao et al., 2013)

(a) Theoretical "rigid-plastic" friction behavior

(b) "Elastoplastic" frictional behavior with very large initial stiffness in OpenSees



Figure 2-3 Normalized Backbone Curve (Dao et al., 2013)

However, in the implementation of the element in OpenSees by Dao et al. (2013), multidirectional circular gap elements were used, whereas the implementation of the element in program SAP2000 by Fenz and Constantinou (2009), several one-directional gap elements were used as they were the only gap elements available in the program.

Based on the nomenclature used in Figure 2-1, Table 2-1 presents the parameters in the series model in OpenSees. The parameters are identical to the Fenz and Constantinou (2009) model. Accordingly, the theory presented in Fenz and Constantinou, 2008c, d) is applicable to the Dao element in OpenSees.

Bidirectional Plasticity	Friction Model	Actual Displacement limit
Element		(Gap Element)
$\bar{L}_1 = 2L_1$	$\bar{\mu_1} = \mu_2 = \mu_3$	$\bar{u}_2 = u_{limit} - \bar{u}_2 - \bar{u}_3$
$\bar{L}_2 = L_2 - L_1$	$\bar{\mu}_2 = \mu_1$	$\bar{u}_2 = \left(1 - \frac{L_1}{L_2}\right) d_1^*$
$\bar{L}_3 = L_3 - L_1$	$ar{\mu}_3=\mu_4$	$\bar{u}_3 = \left(1 - \frac{L_1}{L_3}\right) d_4^*$
$u_{limit} = d_1^* + d_2^* + d_3^* + d_4^* + \frac{b_2}{2},$	where $b_2 = diameter \ of \ rigid \ slides b_2$	ler

Table 2-1 Parameters in Series Model

The series model captures well the behavior of Triple FP bearings that satisfy the restrictions listed in the Introduction (Fenz and Constantinou, 2008a, c and Constantinou et al., 2011). It also captures acceptably well the velocity dependence of the coefficient of friction by use of adjusted friction model parameters as demonstrated in Fenz and Constantinou (2009). However, the model does not compute the actual histories of displacements and velocities at each of the four sliding interfaces. This topic will be addressed in Section 3.

2.2 Dependence of Friction Coefficient on Velocity, Pressure and Temperature

Constantinou et al. (2007) presented information on the dependency of the coefficient of friction at interfaces of sliding bearings on the velocity of sliding, apparent bearing pressure, and temperature. They also presented and validated by experimentation a theory on computing the change in temperature at a sliding interface as the result of frictional heating. In general, the coefficient of friction increases with increasing velocity, drops with increasing pressure and drops with increasing temperature, and reaches a somehow stable value at some large velocity, pressure and temperature. The behavior depends on the materials of the sliding interface and needs to be determined by testing, although some general rules apply and are discussed in Constantinou et al. (2007). Always one of the two sliding surfaces is stainless steel, and the other is a much softer material (plastic or fabric in woven form or several layers of these materials).

The computation of the temperature at the sliding interfaces requires knowledge of the instantaneous values of the coefficient of friction, the apparent bearing pressure (load divided by apparent contact area), and sliding velocity at each interface. The total temperature $T = T_0 + \Delta T$ at time t > 0, consists of the starting value T_0 at time zero and the rise ΔT at the sliding interface. The temperature rise ΔT at time t is given by (Constantinou et al., 2007):

$$\Delta T(t) = \frac{\sqrt{D}}{k\sqrt{\pi}} \int_0^t \frac{q(t-\tau)d\tau}{\sqrt{\tau}}$$
(2-1)

where, *D* and *k* are the thermal diffusivity and conductivity of stainless steel, respectively, τ is a time parameter that varies between 0 and time *t*, and *q(t)* is the heat flux, which is calculated in accordance with the equation (2-2).

$$q(t) = \begin{cases} \mu(t)p(t)v(t), & d \le rContact \\ 0, & otherwhise \end{cases}$$
(2-2)

In equation (2-2), $\mu(t)$ is a friction coefficient, p(t) is the apparent bearing pressure at the sliding surface, and v(t) is the absolute value of the resultant velocity at the sliding surface. Also, *d* is the absolute value of the resultant displacement of the slider and *rContact* is the radius of the circular apparent contact area, which can be obtained by $b_i/2$ (see Figure 1-1). Equation (2-2) computes the temperature at the center of the bearing. Note that the heat flux may be intermittent depending on the motion of the slider over the contact area (Constantinou et al., 2007). Typical values of the thermal properties of stainless are presented in Table 2-2.

Table 2-2 Thermal Properties of Stainless Steel

Parameter	Unit	Value
Thermal Diffusivity	m²/sec	$0.444 * 10^{-5}$
Thermal Conductivity	<i>W</i> /(<i>m</i> °C)	18

A generally accepted relation for the velocity dependence of the friction coefficient is (Constantinou et al., 2007):

$$\mu(v) = \mu_{max}(1 - (1 - \widetilde{\mu_v})e^{-av})$$
(2-3)

In this equation, $\mu(v)$ is the friction coefficient at sliding velocity v, μ_{max} is friction coefficient at large sliding velocity, $\tilde{\mu_v}$ is the ratio of the friction coefficient at very small velocity to μ_{max} , and a is the rate parameter, which has a value of about 100s/m for some of the materials used in sliding bearings (Constantinou et al., 2007). When $\tilde{\mu_v}$ is assumed as 0.5, which is consistent with past studies (Constantinou et al., 2007), equation (2-3) reduces to the following form when the velocity is in units of meter/second:

$$\mu(v) = \mu_{max}(1 - 0.5e^{-100v}) \tag{2-4}$$

Sarlis and Constantinou (2013) used the following relation to model the coefficient of friction as function of temperature:

$$\mu(T) = \mu_{min} + (\mu_{max} - \mu_{min})e^{-hT}$$
(2-5)

In equation (2-5), *h* is a heating rate parameter, μ_{max} is the value of the high velocity coefficient of friction at the start (t=0), and μ_{min} is the value of the high velocity coefficient of friction at some large temperature (T $\approx 1/h$) where it is assumed to be stable.

Tsopelas et al. (2005) implemented in program 3D-BASIS-ME-MB pressure dependency for the coefficient of friction based on the following relationship:

$$\mu_{max} = \mu_{max0} - \left(\mu_{max0-}\mu_{maxp}\right) \tanh\left(\varepsilon p\right)$$
(2-6)

In this equation, μ_{max} is the value of the high velocity coefficient of friction at pressure *p* (axial load divided by apparent contact area), μ_{max0} is the value of the high velocity coefficient of friction at almost zero pressure, μ_{maxp} is the value of the high velocity coefficient of friction at some high value of pressure, tanh is the tangent hyperbolic function, and ε is a constant that controls the rate of change of the coefficient of friction with pressure. In this model it is assumed that the coefficient of friction at very low velocity is not dependent on pressure.

Kumar et al. (2015) combined and simplified these models into one composite model for the dependency of friction on velocity, pressure and temperature, and had the model implemented in program OpenSees for an element of the Single FP bearing (FPBearingPTV). The coefficient of friction is given by equations (2-7) to (2-10) in which μ_{ref} is the reference high speed coefficient of friction at the initial (time *t*=0) temperature $T_0 = 20$ °C and initial pressure p_0 , *a* is velocity rate parameter, *p* is the apparent pressure, and *v* is the amplitude of the velocity.

$$\mu(p, v, T) = \mu_{ref} k_p k_v k_T \tag{2-7}$$

$$k_{\rm p} = 0.7^{0.02(p-p_0)} \tag{2-8}$$

$$k_v = (1 - 0.5e^{-av}) \tag{2-9}$$

$$k_T = 0.79 * (0.7^{0.02T} + 0.40) \tag{2-10}$$

Note that function k_T for the dependency of the coefficient of friction on the temperature and function k_p for the dependency of the coefficient of friction on instantaneous pressure have been calibrated using general data that apply for FP bearings. Also, function k_v for the dependency of the coefficient of friction on velocity is based on the assumption that the ratio of the very low speed to high-speed coefficient of friction equals to 0.5. The model of equations (2-7) to (2-10) has been used in the developed new Triple FP bearing in OpenSees.

SECTION 3

NEW TRIPLE FP ELEMENT TO ACCOUNT FOR HEATING EFFECTS

The new Triple FP element to be used in OpenSees is named "TripleFrictionPendulumX" to complement the "TripleFrictionPendulum" element of Dao et al. (2013) already implemented in OpneSees. The new element determines the state of motion (displacement and velocity) at each of the four sliding interfaces based on two different procedures. The user has the choice of selecting one of these for the execution. These procedures are: (1) Modification of the element displacement histories computed in the three elements of the series model (based on Fenz and Constantinou, 2008d, 2009) and (2) Retracing through the force-displacement relationship, which again is based on Fenz and Constantinou (2008a-d). The sliding velocities are computed from the displacement histories using numerical differentiation (Burden and Faires, 1989).

3.1 Approach 1: Modification of the displacement histories in the series elements

The series model of the Triple FP bearing employs three pendula with friction in order to represent the stiffnesses and strength that the bearing exhibits in its five regimes of operation. Figures 3-1 and 3-2 show the series model and its parameters and the force-displacement loop as determined by Fenz and Constantinou (2008d).



Figure 3-1 Series model for Triple FP bearing per Fenz and Constantinou (2008d)



Figure 3-2 Force-displacement loop of Triple FP bearing as computed in the series model

Tables 3-1 and 3-2 depict the active ranges of motion in the four sliding interfaces and three pendula elements of the series model, in its five regimes of operation. Herein, u_i and v_i are the displacement and velocity on the four sliding interfaces (*i*=1 to 4), which comprise the bearing's top to bottom motion. From analyzing the information in Tables 3-1 and 3-2, some characteristics of the series model can be described: (a) the second and third pendulum elements in the series model match their active ranges well with actual motions in all five regimes, (b) the first pendulum element describes the actual motion of the inner slide plates u_2 and u_3 only within specific regimes but can represent the motion properly in an overall sense.(i. e., $u_2 + u_3$) From these characteristics, the motion of sliding interfaces can be approximately obtained using compatibility conditions between the actual motion of the pendula and the three representative motions in the series elements, and the methodology presented in Fenz and Constantinou (2008d, 2009) which has been used for accounting for the velocity dependence of friction by modification of the rate parameters in the three frictional elements of the series model.

	Regime I	Regime II	Regime III	Regime IV	Regime V
u_{1}, v_{1}					
u_2, v_2					
u_{3}, v_{3}					
u_4 , v_4					

Table 3-1 Actual motion in each of four sliding interfaces in the five regimes of operation

	Regime I	Regime II	Regime III	Regime IV	Regime V
Element 1					
Element 2					
Element 3					

Table 3-2 Activated elements in the series model in the five regimes of operation

In the sequel, there is a detailed presentation of the theory that leads to calculating the histories of displacement on the four sliding interfaces from the histories of displacement of three pendula. Velocity histories are then calculated numerically from the displacement histories.

1) Derivation for displacement history modification in Regime I

Initiation of motion occurs when the horizontal force exceeds the friction force on surfaces 2 and 3 (they are equal and less than the friction forces at surfaces 1 and 4). This is regime I per Fenz and Constantinou (2008a). During this regime, the force-displacement relationship for surfaces 2 and 3 can be expressed, respectively, as follows:

$$F = \frac{W}{R_{eff2}} u_2 + F_{f2} \tag{3-1}$$

$$F = \frac{W}{R_{eff3}} u_3 + F_{f3}$$
(3-2)

In these equations, F is the horizontal force, W is vertical load, R_{eff2} and R_{eff3} are effective radius of curvature, F_{f2} and F_{f3} are friction forces on surfaces 2 and 3, and u_2 and u_3 are displacement histories on surfaces 2 and 3, respectively.

In the series model, there is motion in element 1, and the corresponding force-displacement relationship can be written as follows:

$$F = \frac{W}{\overline{R_{eff1}}}\overline{u_1} + \overline{F_{f1}}$$
(3-3)

Where $\overline{R_{eff1}} = R_{eff2} + R_{eff3}$, $\overline{F_{f1}} = F_{f2} = F_{f3}$ and $\overline{u_1}$ is displacement in first series element. As $R_{eff2} = R_{eff3}$, the relationship between u_2, u_3 and $\overline{u_1}$ is expressed by combining equations (3-1) to (3-3).

$$u_2 = u_3 = 0.5 \,\overline{u_1} \tag{3-4}$$

In this regime, the displacements on the outer sliding surfaces are zero, as there is no motion on surfaces 1 and 4.

$$u_1 = u_4 = 0 \tag{3-5}$$

2) Derivation for displacement history modification in Regime II

In regime II, motion occurs on surfaces 1 and 3, leading to the following relationship governing motion on the surface:

$$F = \frac{W}{R_{eff1}} u_1 + F_{f1}$$
(3-6)

Here, F_{f1} is friction force at surface 1. The equivalent motion in the second FP element in the series model initiates, and the relationship can be expressed as:

$$F = \frac{W}{\overline{R_{eff2}}}\overline{u_2} + \overline{F_{f2}}$$
(3-7)

Where $\overline{R_{eff2}} = R_{eff1} - R_{eff2}$, $\overline{F_{f2}} = F_{f1}$ and $\overline{u_2}$ is displacement in the second series element. By combining equation (3-6) and (3-7), the relation between u_1 and $\overline{u_2}$ is obtained as:

$$u_1 = \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_2}$$
(3-8)

The displacement history on surface 3 is still governed by the relation 3.4, with motion on surface 2 stopped and obtained by the following compatibility equation:

$$u_{total} = u_1 + u_2 + u_3 + u_4 = \overline{u_1} + \overline{u_2} + \overline{u_3}$$
(3-9)

This leads to

$$u_2 = u_{total} - (u_1 + u_3 + u_4) = u_{total} - (\frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_2} + 0.5\,\overline{u_1})$$
(3 - 10)

Note that u_{total} is total displacement of the bearing, that is, the top to bottom relative displacement. Also, u_3 and u_1 are obtained by use of equations (3-4) and (3-8) with u_4 being zero. Note that \overline{u}_i can be obtained directly from the series elements.

3) Derivation for displacement history modification in Regime III

In Regime III, motion occurs on surfaces, 1 and 4. Using equation (3-6) for surface 1, the governing relationship for motion on surface 4 can be expressed as:

$$F = \frac{W}{R_{eff4}} u_4 + F_{f4} \tag{3-11}$$

Here, F_{f4} is friction force on surface 4. The onset of motion in the third FP element in Series occurs when applied horizontal force F exceeds $\overline{F_{f3}} = F_{f4}$ and corresponding force-displacement relationship is:

$$F = \frac{W}{\overline{R_{eff3}}}\overline{u_3} + \overline{F_{f3}}$$
(3 - 12)

Where $\overline{R_{eff3}} = R_{eff4} - R_{eff3}$ and $\overline{u_3}$ is displacement in the third series element. Combining equations (3-11) and (3-12), results in:

$$u_4 = \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_3}$$
(3 - 13)

Equation (3-8) still describes the motion on surface 1. Motion in inner two surfaces stops in Regime III, but the series model cannot capture these phenomena for surfaces 2 and 3 individually. Instead, the motion can be approximately captured based on the following compatibility equation:

$$u_{2} + u_{3} = u_{total} - (u_{1} + u_{4}) = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_{2}} + \frac{R_{eff4}}{R_{eff4} - R_{eff3}}\overline{u_{3}}\right)$$
(3 - 14)

Note that displacements u_2 and u_3 in Regime III have values equal to the last values recorded at the last time step in Regime I and II. Note that \bar{u}_1 varies during this regime, but it is very small and considered negligible.

4) Derivation for displacement history modification in Regime IV

Stiffening behavior occurs in Regime IV as motion stops on surface 1 and re-starts on surface 2, which has a smaller effective radius of curvature than surface 1. In the series model (Figures 2-1 and 3-1), the gap element in the second series element is activated with very high stiffness so that motion stops on surface 1. Displacements u_2 and u_4 are governed by equations (3-4) and (3-13), whereas u_3 has a constant value that is obtained by use of the following compatibility equation:

$$u_{3} = u_{total} - (u_{1} + u_{2} + u_{4}) = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_{2}} + 0.5\overline{u_{1}} + \frac{R_{eff4}}{R_{eff4} - R_{eff3}}\overline{u_{3}}\right) \quad (3-15)$$

5) Derivation for displacement history modification in Regime V

Further stiffening behavior is achieved as motion on surface 4 stops and motion of surface 3 re-starts. The gap element in the third series element is activated and generates additional stiffness during this regime. Displacements = u_2 and u_3 are governed by equation (3-4).

6) Summary and considerations to account for heating effects

Table 3-3 summarizes the state of motion (displacement) at each of the four sliding interfaces.

RegimeEquations for Displacement $u_1 = u_4 = 0$ I $u_2 = u_3 = 0.5\overline{u_1}$ (based on $u_2 + u_3 = \overline{u_1}$) $u_1 = \frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_2}$ II $u_2 = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_2} + 0.5\overline{u_1}\right) = u_{2_regime_I}$ $u_3 = 0.5\overline{u_1}$ $u_4 = 0$		
I $u_{1} = u_{4} = 0$ $u_{2} = u_{3} = 0.5\overline{u_{1}}$ (based on $u_{2} + u_{3} = \overline{u_{1}}$) $u_{1} = \frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_{2}}$ II $u_{2} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_{2}} + 0.5\overline{u_{1}}\right) = u_{2_regime_I}$ $u_{3} = 0.5\overline{u_{1}}$ $u_{4} = 0$	Regime	Equations for Displacement
I $u_{2} = u_{3} = 0.5\overline{u_{1}}$ (based on $u_{2} + u_{3} = \overline{u_{1}}$) $u_{1} = \frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_{2}}$ II $u_{2} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_{2}} + 0.5\overline{u_{1}}\right) = u_{2_regime_I}$ $u_{3} = 0.5\overline{u_{1}}$ $u_{4} = 0$		$u_1 = u_4 = 0$
$(based on u_{2} + u_{3} = \overline{u_{1}})$ $u_{1} = \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}}$ II $u_{2} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} + 0.5 \overline{u_{1}}\right) = u_{2_regime_I}$ $u_{3} = 0.5 \overline{u_{1}}$ $u_{4} = 0$	Ι	$u_2 = u_3 = 0.5\overline{u_1}$
$u_{1} = \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}}$ II $u_{2} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} + 0.5 \overline{u_{1}}\right) = u_{2_regime_I}$ $u_{3} = 0.5 \overline{u_{1}}$ $u_{4} = 0$		(based on $u_2 + u_3 = \overline{u_1}$)
$u_{1} = \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}}$ II $u_{2} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} + 0.5 \overline{u_{1}}\right) = u_{2_regime_I}$ $u_{3} = 0.5 \overline{u_{1}}$ $u_{4} = 0$		Raffa
II $u_{2} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_{2}} + 0.5 \overline{u_{1}}\right) = u_{2_regime_I}$ $u_{3} = 0.5\overline{u_{1}}$ $u_{4} = 0$		$u_1 = \frac{u_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_2}$
$u_3 = 0.5\overline{u_1}$ $u_4 = 0$	II	$u_{2} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}}\overline{u_{2}} + 0.5 \overline{u_{1}}\right) = u_{2_regime_I}$
$u_4 = 0$		$u_3=0.5\overline{u_1}$
		$u_4 = 0$

Table 3-3 Displacement at each sliding interfaces (Approach 1)
$$(based on u_{2} + u_{3} = u_{total} - \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}})$$

$$u_{1} = \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}}$$

$$u_{2} + u_{3} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} + \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_{3}}\right) = u_{2_regime_I} + u_{3_regime_II}$$

$$u_{4} = \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_{3}}$$

No explicit expressions for u_2 and u_3 (assumed to be constant)

$$u_{1} = \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} = u_{1_regime_III} (Gap \ element \ activated)$$

$$u_{2} = 0.5\overline{u_{1}} (starting \ at \ u_{2_regime_I})$$

$$u_{3} = u_{total} - \left(0.5\overline{u_{1}} + \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} + \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_{3}}\right) = u_{3_regime_III}$$

$$u_{4} = \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_{3}}$$
(based on $u_{2} + u_{3} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} + \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_{3}}\right)$

V

$$\begin{split} u_{1} &= \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} = u_{1_regime_III} (Gap \ element \ activated) \\ u_{2} &= 0.5 \overline{u_{1}} \\ u_{3} &= 0.5 \overline{u_{1}} \ (starting \ at \ u_{3_regime_II}) \\ u_{4} &= \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_{3}} = u_{4_regime_IV} \ (Gap \ element \ activated) \\ (based \ on \ u_{2} + u_{3} = u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_{2}} + \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_{3}}\right) \end{split}$$

In Table 3-3, $u_{2_regime_I}$ is the value of displacement on surface 2 recorded at the last time step of Regime II, $u_{3_regime_II}$ is the value of displacement on surface 3 recorded at the last time step of Regime III, $u_{1_regime_III}$ is the value of displacement on surface 1 recorded at the last time step of Regime III, and $u_{4_regime_IV}$ is the value of displacement on surface 4 recorded at the last time step of Regime IV. The calculated histories of motion are then used for calculating the heat flux and updating values of the friction coefficients during analysis. The average value of u_2 and u_3 is used for updating the value of the friction coefficients $\mu_2 = \mu_3 = \overline{\mu_1}$.

Table 3-4 summarizes important equations used to calculate the displacement histories of individual surfaces in OpenSees element "TripleFrictionPendulumX".

Element	Equations
Element 1	$\overline{\overline{u_1}} = \frac{u_2 + u_3}{2} = 0.5 \left\{ u_{total} - \left(\frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_2} + \frac{R_{eff4}}{R_{eff4} - R_{eff3}} \overline{u_3} \right) \right\}$
Element 2	$\overline{\overline{u_2}} = u_1 = \frac{R_{eff1}}{R_{eff1} - R_{eff2}} \overline{u_2}$
Element 3	$\overline{\overline{u_3}} = u_4 = \frac{R_{eff4}}{R_{eff4} - R_{eff3}}\overline{u_3}$

Table 3-4 Equations to calculate surface displacements in element TripleFrictionPendulumX

In Table 3-4, $\overline{u_1}$, $\overline{u_2}$, and $\overline{u_3}$ are modified element displacements (modifications of displacements $\overline{u_1}$, $\overline{u_2}$ and $\overline{u_3}$) in order to obtain the actual displacement at each sliding surface. Note that $\overline{u_2} \cong 0$ in Regime I and $\overline{u_3} \cong 0$ during Regimes I and II. These displacements have very small but non-zero values. They occur due to the elasto-plastic representation of friction in the model (they are less than the "yield displacement").

3.2 Approach 2: Retracing Histories based on Force-displacement relationship

Approach 1 is incapable of capturing the start-stop-start behavior at all sliding surfaces so that it cannot provide the exact displacements of the inner surfaces 2 and 3 during the entire motion. However, with a small time increment in the analysis, the displacement and velocity histories on individual surfaces can be obtained based on the force obtained in the previous time step. Fenz and Constantinou (2008a) presented the force-displacement relationships for each sliding regime of the Triple FP bearing. In the modified Triple FP OpenSees element, forces and displacements are stored in each analysis step, and these are used for retracing displacement and velocity histories on the four sliding surfaces. The main purpose for developing Approach 2 is to verify the results of the simpler Approach 1.

In the sequel we present the steps needed for the calculations in Approach 2. Table 3-5 presents the forcedisplacement relationships in each regime and details of the computations for the initial (starting) loading phase.

1) Starting Loading Phase

Regime	Force-Displacement Relationship				
0	$u_1 = u_2 = u_3 = u_4 = 0$				
	Valid until: $f < F_{f2}$, Store sign of loading (say "sign")				
	$u_1 = u_4 = 0, u_2 = u_3 = \frac{f - \text{sign} * F_{f_2}}{W} L_1$				
Ι	Store u_1, u_2, u_3, u_4, f				
	Valid until: sign $* f \leq F_{f1}$				
	$u_1 = \frac{f - \text{sign} * F_{f_1}}{W} * L_2, u_3 = \frac{f - \text{sign} * F_{f_2}}{W} L_1$				
II	$u_2 = u_{2_stored}$, $u_4 = u_{4_stored}$				
	Store u_1, u_2, u_3, u_4, f				
	Valid until: $F_{f1} < \text{sign} * f \le F_{f4}$				
	$u_1 = \frac{f - \operatorname{sign} * F_{f_1}}{W} * L_2, u_4 = \frac{f - \operatorname{sign} * F_{f_4}}{W} * L_3$				
III	$u_2 = u_{2_stored}, u_3 = u_{3_stored}$				
	Store u_1, u_2, u_3, u_4, f				
	Valid until: $F_{f4} < \text{sign} * f \le F_{dr1}$				
	$u_{2} = \left(\frac{f - \operatorname{sign} * F_{f_{2}}}{W} - \frac{\operatorname{sign} * d_{2}}{L_{2}}\right) L_{1}, \ u_{4} = \frac{f - \operatorname{sign} * F_{f_{4}}}{W} * L_{3}$				
IV	$u_1 = u_{1_stored}, u_3 = u_{3_stored}$				
	Store u_1, u_2, u_3, u_4, f				
	Valid until: $F_{dr1} < \text{sign} * f \le F_{dr4}$				
	$u_{2} = \left(\frac{f - \operatorname{sign} * F_{f_{2}}}{W} - \frac{\operatorname{sign} * d_{2}}{L_{2}}\right) L_{1}, u_{3} = \left(\frac{f - \operatorname{sign} * F_{f_{2}}}{W} - \frac{\operatorname{sign} * d_{3}}{L_{3}}\right) L_{1}$				
V	$u_1 = u_{1_stored}, u_4 = u_{4_stored}$				
	Store u_1, u_2, u_3, u_4, f				
	Valid until: $F_{dr4} < \text{sign} * f$				

Table 3-5 Force-displacement	Relationships	during	Loading	Phase
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• Note: See Tables 2-1 and 3-7 for nomenclature

In this table, L_1 is the effective radius $R_{eff2} = R_{eff3}$, L_2 is the effective radius R_{eff1} , L_3 is the effective radius R_{eff4} , F_{fi} are friction forces on surfaces i, F_{dri} are forces when the inner slide plates contact the restrainers on the outer surface i.

During the numerical analysis it is assumed that the bearing is in the loading phase until a change in the sign of the derivative of the force with respect to the displacement (slope of force-displacement curve) changes, in which case unloading occurs. The behavior during unloading depends on the regime in which the bearing is at the instant of change of sign of the slope. As an example, Table 3-6 presents details of one of the possible cases of behavior when the force is in Regime V during the unloading phase.

Preparation	Call stored u_1, u_2, u_3, u_4 and f from previous loading phase and assign				
	as reference points				
Regime	Force-Displacement Relationship				
0	$u_1 = u_{1_stored}, u_2 = u_{2_stored}, u_3 = u_{3_stored}, u_4 = u_{4_stored}$				
	Store u_1, u_2, u_3, u_4, f				
	Valid until: $f_{ref} - 2F_{f2} < f$				
	$u_2 = \frac{f - f_{stored}}{W} L_1 + u_{2_{stored}} \le sign(u_2) * d_1^*$				
	$u_3 = \frac{f - f_{stored}}{W} L_1 + u_{3_{stored}} \leq sign(u_3) * d_1^*$				
Ι	$u_1 = u_{1_stored}, u_4 = u_{4_stored}$				
	Store u_1, u_2, u_3, u_4, f				
	Valid until: $F_{dr1} - 2F_{f1} < f \le f_{ref} - 2F_{f2}$				
	$u_1 = \frac{f - f_stored}{W} L_2 + u_{1_{stored}} \le sign(u_1) * d_2^*$				
	$u_3 = \frac{f - f_{stored}}{W} L_1 + u_{3_{stored}} \leq sign(u_3) * d_1^*$				
II	$u_2 = u_{2_stored}, u_4 = u_{4_stored}$				
	Store u_1, u_2, u_3, u_4, f				
	Valid until: $F_{dr4} - 2F_{f4} < f \le F_{dr1} - 2F_{f1}$				
	$u_1 = \frac{f - f_stored}{W} L_2 + u_{1_{stored}} \le sign(u_1) * d_2^*$				
	$u_4 = \frac{f - f_{stored}}{W} L_3 + u_{4_{stored}} \leq sign(u_4) * d_3^*$				
III	$u_2 = u_{2_stored}, u_3 = u_{3_stored}$				
	Store u_1, u_2, u_3, u_4, f				
	Valid until: $-F_{dr1} < f \le F_{dr4} - 2F_{f4}$				
IV	$u_2 = \frac{f - f_{stored}}{W} L_1 + u_{2_{stored}} \le sign(u_2) * d_1^*$				

2) Unloading Phase, Force-displacement loop starts in Regime V

Table 3-6 Triple FP Element Behavior in Unloading Phase Starting in Regime V

	f-fatored
	$u_4 = \frac{\int Stored}{W} L_3 + u_{4stored} \leq sign(u_4) * d_3^*$
	$u_1 = u_{1_stored}, u_3 = u_{3_stored}$
	Store u_1, u_2, u_3, u_4, f
	Valid until: $-F_{dr4} < f \leq -F_{dr1}$
	$u_2 = \frac{f - f_{stored}}{W} L_1 + u_{2_{stored}} \le sign(u_2) * d_1^*$
	$u_3 = \frac{f - f_{stored}}{W} L_1 + u_{3_{stored}} \leq sign(u_3) * d_1^*$
V	$u_1 = u_{1_stored}, u_4 = u_{4_stored}$
	Store u_1, u_2, u_3, u_4, f
	Valid until: $f \leq -F_{dr4}$

Note: See Tables 2-1 and 3-7 for nomenclature

In Table 3-6, d_i^* are the actual displacement capacities. Expressions for the force and corresponding displacement limits that define the boundaries of each regime are presented in Tables 3-7 and 3-8.

Force Limit		Displacement Limit	
F_{f1}	$\mu_1 W$	<i>u</i> *	$2(\mu_1 - \mu_2)L_1$
F_{f4}	$\mu_4 W$	<i>u</i> **	$u^* + (\mu_4 - \mu_1)(L_2 - L_1)$
F _{dr1}	$W\left(\frac{d_1^*}{L_2} + \mu_1\right)$	u_{dr1}	$u^{**} + d_1^* \left(1 + \frac{L_3}{L_2} \right) - (\mu_4 - \mu_1)(L_2 + L_3)$
F _{dr4}	$W\left(\frac{d_4^*}{L_3} + \mu_4\right)$	u _{dr4}	$u_{dr1} + \left(\left(\frac{d_4^*}{L_3} + \mu_4 \right) - \left(\frac{d_1^*}{L_2} + \mu_1 \right) \right) (L_1 + L_3)$

Table 3-7 Force and Displacement Limits in Each Regime

Table 3-8 Boundary Conditions in Each Regime (In Unloading Phase)

Regime	f_{ref} starts at regime 5	f_{ref} starts at regime 4	f_{ref} starts at regime 3,2,1
0 (START)	$f_{ref} - 2F_{f2} < f$	$f_{ref} - 2F_{f2} < f$	$f_{ref} - 2F_{f2} < f$
Ι	$F_{dr1} - 2F_{f1} < f \le f_{ref}$ $- 2F_{f2}$	$F_{dr1} - 2F_{f1} < f \le f_{ref} - 2F_{f2}$	$f_{ref} - 2F_{f1} < f \le f_{ref} - 2F_{f2}$
II	$F_{dr4} - 2F_{f4} < f \le F_{dr1}$ $- 2F_{f1}$	$f_{ref} - 2F_{f4} < f \le F_{dr1} - 2F_{f1}$	$f_{ref} - 2F_{f4} < f \le f_{ref} - 2F_{f1}$
III	$-F_{dr1} < f \le F_{dr4} - 2F_{f4}$	$-F_{dr1} < f \le f_{ref} - 2F_{f4}$	$-F_{dr1} < f \le f_{ref} - 2F_{f4}$
IV	$-F_{dr4} < f \le -F_{dr1}$	$-F_{dr4} < f \le -F_{dr1}$	$-F_{dr4} < f \le -F_{dr1}$
V	$f \le -F_{dr4}$	$f \leq -F_{dr4}$	$f \leq -F_{dr4}$

Approach 2, like in Approach 1, makes use of the average value of u_2 and u_3 in accounting for the dependency of the friction coefficient on temperature in inner sliding interfaces.

3.3 Calculation of Velocities on Sliding Surfaces

The sliding velocities at each sliding surface are needed for the calculation of the coefficient of friction when velocity-dependent and for the calculation of the heat flux to be used in computing the temperature at each sliding surface. Following the calculation of the displacements at each sliding surface, the sliding velocities are calculated by numerical differentiation. The numerical scheme used is based on the three-point formula (Burden and Faires, 1989):

$$f'(x_0) = \frac{1}{2h} \left[\left(f(x_0 + h) - f(x_0 - h) \right] - \frac{h^2}{6} f'''(\xi)$$
(3-16)

In this equation the first derivative of function f at point x_0 is computed as the slope of the function between points $x_0 + h$ and $x_0 - h$, where h is the step (or time-step). The error is order $O(h^2)$, which together with a small time-step results in an estimate of velocity with small error. Table 3-9 presents the equations used for each surface and for the two approaches used. In this table, subscript *i* denotes the current time.

Surface	Approach 1	Approach 2
1	$\overline{\overline{v_2}}_i = \frac{1}{2h} (\overline{\overline{u_2}}_{,i} - \overline{\overline{u_2}}_{,i-2})$	$v_{1_i} = \frac{1}{2h}(u_{1_i} - u_{1_{i-2}})$
2	$\overline{\overline{v_1}}_i = \frac{1}{2h} (\overline{\overline{u_1}}_{,i} - \overline{\overline{u_1}}_{,i-2})$	$v_{2_i} = \frac{1}{2h}(u_{2_i} - u_{2_{i-2}})$
3	$\overline{\overline{v_1}}_i = \frac{1}{2h} (\overline{\overline{u_1}}_{,i} - \overline{\overline{u_1}}_{,i-2})$	$v_{3_i} = \frac{1}{2h}(u_{3_i} - u_{3_{i-2}})$
4	$\overline{\overline{v_3}}_i = \frac{1}{2h} (\overline{\overline{u_3}}_{,i} - \overline{\overline{u_3}}_{,i-2})$	$v_{4i} = \frac{1}{2h}(u_{4i} - u_{4i-2})$

Table 3-9 Calculation of Velocities

Equation (3-16) makes use of the displacement at a forward time, which is not known during analysis. Accordingly, the velocity is computed at one time-step backwards and used at the current time. This requires the use of a small time-step.

SECTION 4

VERIFICATION EXAMPLES FOR PRESCRIBED BEARING MOTION AND CONSTANT FRICTION

This section presents examples of verification of the modified Triple FP element (TripleFrictionPendulumX element) in OpenSees. Table 4-1 presents information on two configurations in a total of four different cases of Triple FP bearing used in the examples. The isolators have an inner displacement restrainer or ring. In this section the friction coefficient values are considered constant and independent of pressure, velocity and temperature. This assumption is relaxed in other examples presented in Section 5. Values of parameters used in the analysis in addition to those below are presented in Appendix A.

Geometric and Frictional Properties	Configuration A		Configuration B	
$R_1=R_4 (mm)$	3962.4 (156")		2235 (88")	
$R_2=R_3 (mm)$	558.8	(22")	305 (12")	
h ₁ =h ₄ (mm)	215.9	(8.5")	114.5 (4.5")	
h ₂ =h ₃ (mm)	165.1	(6.5")	76 (3")	
$R_{eff1}=R_{eff4} (mm)$	3746.5 (147.5")		2120.5 (83.5")	
R _{eff2} =R _{eff3} (mm)	393.7 (15.5")		229 (9")	
b ₁ =b ₄ (mm)	711.2 (28")		279 (11")	
b ₂ =b ₃ (mm)	508 (20")		196 (7.7")	
d ₁ =d ₄ (mm)	533.4 (21")		267 (10.5")	
d ₂ =d ₃ (mm)	101.6 (4")		40 (1.6")	
$d_1^* = d_4^* (mm)$	504.3 (19.9")		253.3 (10.0")	
$d_2^* = d_3^* (mm)$	71.6 (2.8")		30.0 (1.2")	
Vertical Stiffness (MN/mm)	8.0 (45720 kip/in)		8.0 (45720 kip/in)	
Friction Case	Case 1	Case 2	Case 1	Case 2
μ ₁	0.04	0.04	0.04	0.04
μ_4	0.08	0.04	0.08	0.04
$\mu_2 = \mu_3$	0.01	0.01	0.01	0.01
Load W (kN)	13345 (3000kip)		6672 (1500kip)	

Table 4-1 Geometric and Frictional Properties used in Analysis (see Figure 1-1 for nomenclature)

4.1 Examples of Displacement Control Analysis

This subsection presents the results of displacement control analysis in programs OpenSees and 3pleANI. In these analyses, a displacement history is imposed at the top of the bearing with respect to its bottom. The results include force-displacement loops, and displacement, velocity and temperature histories at the four sliding interfaces. In the figures that follow, results are presented with (a) color green and denoted as APP1 when Approach 1 is used in OpenSees, and (b) color blue and denoted as APP2 when Approach 2 is used.



4.1.1 Harmonic Motion: Amplitude of 20in and Period of 20sec

Figure 4-1 Imposed Harmonic Motion (Amplitude: 20in, Period: 20sec)

A) Configuration A







Figure 4-3 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-4 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-5 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-6 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-7 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-8 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-9 Comparison of Force-Displacement Loops



Figure 4-10 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-11 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-12 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-13 Comparison of Velocity Histories on Inner Surfaces (Surface 2 and 3)



Figure 4-14 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-15 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

B) Configuration B



Figure 4-16 Comparison of Force-Displacement Loops



Figure 4-17 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-18 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-19 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-20 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-21 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-22 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-23 Comparison of Force-Displacement Loops



Figure 4-24 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-25 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-26 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-27 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-28 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-29 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

4.1.2 Harmonic Motion: Amplitude of 22in and Period of 10sec



Figure 4-30 Imposed Harmonic Motion (Amplitude: 22in, Period: 10sec)

A) Configuration A



Figure 4-31 Comparison of Force-Displacement Loops



Figure 4-32 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-33 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-34 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-35 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-36 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-37 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-38 Comparison of Force-Displacement Loops



Figure 4-39 Comparison of Displacement Histories on Outer Surfaces (Surface 1 and 4)



Figure 4-40 Comparison of Displacement Histories on Inner Surfaces (Surface 2 and 3)



Figure 4-41 Comparison of Velocity Histories on Outer Surfaces (Surface 1 and 4)



Figure 4-42 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-43 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-44 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

B) Configuration B



Figure 4-45 Comparison of Force-Displacement Loops



Figure 4-46 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-47 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-48 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-49 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-50 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-51 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-52 Comparison of Force-Displacement Loops



Figure 4-53 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-54 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-55 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-56 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-57 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-58 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

4.1.3 Harmonic Motion: Amplitude of 20in and Period of 5sec



Figure 4-59 Imposed Harmonic Motion (Amplitude: 20in, Period: 5sec)

A) Configuration A



Figure 4-60 Comparison of Force-Displacement Loops



Figure 4-61 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-62 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-63 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-64 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-65 Comparison of Force-Displacement Loops



Figure 4-66 Comparison of Displacement Histories on Outer Surfaces (Surface 1 and 4)


Figure 4-67 Comparison of Displacement Histories on Inner Surfaces (Surface 2 and 3)



Figure 4-68 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-69 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

B) Configuration B



Figure 4-70 Comparison of Force-Displacement Loops



Figure 4-71 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-72 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-73 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-74 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

b. Case 2: Same Coefficient of Friction for Surface 1 and 4



Figure 4-75 Comparison of Force-Displacement Loops



Figure 4-76 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-77 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 4-78 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 4-79 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

4.2 **Observations and Comments**

The following observations are made in the comparison of results in Section 4.

- 1) The force-displacement loops obtained by the model in program OpenSees and the more advanced model in program 3pleANI are in good agreement.
- 2) In general, the two models (Approach 1 and Approach 2) in program OpenSees produce nearly identical results for all computed quantities.
- 3) In general, the two models in program OpenSees produce results on the displacement, velocity and temperature of the main surfaces 1 and 4 that are in good agreement with those produced by the more advanced model in program 3pleANI.
- 4) In general, the two models in program OpenSees produce results on the displacement, velocity and temperature of the inner surfaces 2 and 3 that differ from those produced by the more advanced model in program 3pleANI. The temperature is underpredicted by the models in program OpenSees. However, the motion and temperature in the inner surfaces are significantly smaller than those of the main surfaces 1 and 4 (e.g., the temperature rise in the outer surfaces is an order of magnitude larger than that in the inner surfaces) so that the effect on the behavior of the bearing is insignificant.

SECTION 5

VERIFICATION EXAMPLES FOR PRESCRIBED BEARING MOTION AND VARIABLE FRICTION

This section presents verification examples when the friction coefficients are temperature-dependent. Dependencies on velocity and pressure are not considered in this section. The geometric properties of the Triple FP bearings are same as those used in Section 4 (Table 4-1) with the exemption that friction is not constant. In the examples of this section, the values of friction coefficient in Table 4-1 represent the reference values, that is, the values at high speed of motion, and at the initiation of motion prior to any increase in temperature of the sliding interfaces. Values of parameters used in the analysis in addition to those below are presented in Appendix A. Moreover, the values of thermal diffusivity and conductivity listed in Table 2-2 are used.

5.1 Examples of Displacement Control Analysis with Temperature-Dependent Friction Coefficient

In program OpenSees, the dependency of the friction coefficients on velocity, pressure and temperature is described by equation (2-7). Parameters k_p and k_v are set equal to unity so that pressure and velocity effects are excluded. The remaining equation describes the dependency of friction on only temperature:

$$\mu_i(T) = \mu_{ref,i} * 0.79 * (0.7^{0.02T_i} + 0.40)$$
(5 - 1)

In this equation, μ_{ref} is the reference high speed coefficient of friction at the initial (time *t*=0) temperature $T_0 = 20^{\circ}$ C at surface i, and T_i is temperature rise.

In program 3pleANI, the dependency of the friction coefficients on temperature is given by the following exponential form:

$$\mu_i(T) = \mu_{\min,i} + \left(\mu_{\max,i} - \mu_{\min,i}\right)e^{-h(T_i - T_0)}$$
(5-2)

In this equation, μ_i is friction coefficient at surface i, *h* is the heating rate parameter, T_i is temperature rise, T_0 is initial temperature, $\mu_{max,i}$ is the friction coefficient at initial temperature, $\mu_{min,i}$ is minimum value of friction coefficient at a large temperature. For the examples of this section, $\mu_{min,i} = \mu_{max,i}/2$ and parameter $h=0.01/^{\circ}$ C are used. Moreover, the values of the coefficient of friction in the Sarlis and Constantinou (2013, 2016) model differ from those of the Fenz and Constantinou (2008a-d) model which is used in program OpenSees. In general, the two values are identical when the sliding interfaces have infinitely large radii of curvature. Otherwise, they are related through the geometry of the sliding surfaces by the following equations:

$$\mu_{ref,2} = \mu_{max,2} \frac{R_2}{R_{eff2}}$$
(5-3)

$$\mu_{ref,1} = \frac{\mu_{max,1}R_1 - \mu_{max,2}R_2}{R_{eff1} - R_{eff2}}$$
(5-4)

$$\mu_{ref,4} = \frac{\mu_{max,4}R_4 - \mu_{max,2}R_2}{R_{eff1} - R_{eff2}}$$
(5-5)

In these equations, $\mu_{ref,i}$ is the friction coefficient used in OpenSees, $\mu_{max,i}$ is the friction coefficient used in 3pleANI, R_i is the radius of curvature, R_{effi} is effective radius of curvature which is equal to $R_{effi} = R_i - h_i$, and subscript i indicates sliding surface *i*=1 to 4. Based on the geometric parameters of the Triple FP bearing configuration A listed in Table 4-1, the values of the friction coefficients, $\mu_{ref,i}$ and $\mu_{max,i}$, are listed in Table 5-1 and plotted in Figure 5-1. Note that for comparison of the two models of frictiontemperature dependency, the values of the Sarlis and Constantinou (2013, 2016) were adjusted per equations (5-3) to (5-5) and shown in Figure 5-1 for an initial temperature of 20°C.

Table 5-1 Friction Coefficients used in OpenSees and 3pleANI

Friction Coefficient	Value of $\mu_{ref,i}$ in OpenSees	Value of $\mu_{max,i}$ in 3pleANI
μ_{ref1}	0.04	0.03484
μ_{ref4}	0.08	0.06869
$\mu_{ref2} = \mu_{ref3}$	0.01	0.00705



Figure 5-1 Friction-Temperature Dependency in OpenSees and 3pleANI

The two friction models differ but the differences are small in the range of 20°C to about 100°C so results of the two programs in this range of temperatures should be comparable.

Results by the two programs are presented and compared for prescribed conditions of motion of the top of the bearing. Approach 1 has been used in program OpenSees. These results include force-displacement loops, and displacement, velocity and temperature histories at the four sliding interfaces. Also, results are presented in the form of histories of the coefficient of friction. In the figures that follow, results of program OpenSees are in color blue and results of program 3pleANI are in color red. Note that in the comparison of histories of the friction coefficient, the two programs produce comparable results but the values differ based on the interpretation provided above and demonstrated in the results of Table 5-1 and Figure 5-1.

5.1.1 Harmonic Motion: Amplitude of 20in and Period of 20sec



Figure 5-2 Imposed Harmonic Motion (Amplitude: 20in, Period: 20sec)

A) Configuration A



Figure 5-3 Comparison of Force-Displacement Loops



Figure 5-4 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-5 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-6 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-7 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-8 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-9 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-10 Histories of Friction Coefficients on Outer Surfaces (note that the friction coefficient values are not the same but related through equations 5-3 to 5-5)

b. Case 2: Same Coefficient of Friction for Surfaces 1 and 4



Figure 5-11 Comparison of Force-Displacement Loops



Figure 5-12 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-13 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-14 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-15 Comparison of Velocity Histories on Inner Surfaces (Surface 2 and 3)



Figure 5-16 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-17 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-18 Histories of Friction Coefficients on Outer Surfaces (note that the friction coefficient values are not the same but related through equations 5-3 to 5-5)

B) Configuration B



Figure 5-19 Comparison of Force-Displacement Loops



Figure 5-20 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-21 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-22 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-23 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-24 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-25 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

b. Case 2: Same Coefficient of Friction for Surfaces 1 and 4



Figure 5-26 Comparison of Force-Displacement Loops



Figure 5-27 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-28 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-29 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-30 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-31 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-32 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

5.1.2 Harmonic Motion: Amplitude of 22in and Period of 10sec



Figure 5-33 Imposed Harmonic Motion (Amplitude: 22in, Period: 10sec)

A) Configuration A



Figure 5-34 Comparison of Force-Displacement Loops



Figure 5-35 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-36 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-37 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-38 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-39 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-40 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

b. Case 2: Same Coefficient of Friction for Surfaces 1 and 4



Figure 5-41 Comparison of Force-Displacement Loops



Figure 5-42 Comparison of Displacement Histories on Outer Surfaces (Surface 1 and 4)



Figure 5-43 Comparison of Displacement Histories on Inner Surfaces (Surface 2 and 3)



Figure 5-44 Comparison of Velocity Histories on Outer Surfaces (Surface 1 and 4)



Figure 5-45 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-46 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-47 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

B) Configuration B



Figure 5-48 Comparison of Force-Displacement Loops



Figure 5-49 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-50 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-51 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-52 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-53 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-54 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

b. Case 2: Same Coefficient of Friction for Surface 1 and 4



Figure 5-55 Comparison of Force-Displacement Loops



Figure 5-56 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-57 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-58 Comparison of Velocity Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-59 Comparison of Velocity Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-60 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-61 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

5.1.3 Harmonic Motion: Amplitude of 20in and Period of 5sec



Figure 5-62 Imposed Harmonic Motion (Amplitude: 20in, Period: 5sec)

A) Configuration A



Figure 5-63 Comparison of Force-Displacement Loops



Figure 5-64 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-65 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-66 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)


Figure 5-67 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

b. Case 2: Same Coefficient of Friction for Surfaces 1 and 4



Figure 5-68 Comparison of Force-Displacement Loops



Figure 5-69 Comparison of Displacement Histories on Outer Surfaces (Surface 1 and 4)



Figure 5-70 Comparison of Displacement Histories on Inner Surfaces (Surface 2 and 3)



Figure 5-71 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-72 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

B) Configuration B



a. Case 1: Different Coefficient of Friction for Surface 1 and 4

Figure 5-73 Comparison of Force-Displacement Loops



Figure 5-74 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-75 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-76 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-77 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

b. Case 2: Same Coefficient of Friction for Surface 1 and 4



Figure 5-78 Comparison of Force-Displacement Loops



Figure 5-79 Comparison of Displacement Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-80 Comparison of Displacement Histories on Inner Surfaces (Surfaces 2 and 3)



Figure 5-81 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 5-82 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

5.2 Observations and Comments

The observations made in the comparison of results when friction is temperature independent in Section 4 (presented in subsection 4.1.3) are valid for the case of friction that is temperature dependent. Specifically:

- 1) The force-displacement loops obtained by the model in program OpenSees and the more advanced model in program 3pleANI are in good agreement.
- 2) In general, the two models in program OpenSees produce results on the displacement, velocity and temperature of the main surfaces 1 and 4 that are in good agreement with those produced by the more advanced model in program 3pleANI.
- 3) In general, the two models in program OpenSees produce results on the displacement, velocity and temperature of the inner surfaces 2 and 3 that differ from those produced by the more advanced model in program 3pleANI. Nevertheless, the motion and temperature in the inner surfaces are significantly smaller than those of the main surfaces 1 and 4 (e.g., the temperature rise in the outer surfaces is an order of magnitude larger than that in the inner surfaces) so that effect on the behavior of the bearing is insignificant.

It is also observed that in examples where there is large increase in the temperature at the sliding surfaces, the two models in OpenSees and 3pleANI have predictions of temperature histories that differ more than that observed in examples with smaller temperature increases. This is best seen in comparing the results of Figure 5-24 (temperatures reaching about 110°C) and Figure 5-81 (temperatures reaching about 180°C)-both are for the same configurations (B with same friction coefficients on surfaces 1 and 4) and subjected to the same amplitude motion but with period of 20sec in the case of Figure 5-24 and period of 5sec in the case of Figure 5-81. This is due to differences in the temperature-dependency of the friction coefficient in the models in programs OpenSees and 3pleANI, as explained in Section 5.1 (see also Figure 5-1).

SECTION 6

RESPONSE HISTORY ANALYSIS OF BUILDINGS WITH TRIPLE FP BEARINGS

6.1 Example: Response History Analysis of a Rigid Structure

A rigid structure supported on triple FP isolators is subjected to ground motion in bidirectional horizontal and in the vertical directions. Results produced by programs OpenSees and 3pleANI on isolation system force-displacement loops, and the histories of isolator displacement, structural acceleration and temperature at the sliding surfaces are compared. In the analysis, only one isolator of Configuration B and two cases of friction (different and equal at the two main sliding interfaces-see Table 4-1) is utilized. The weight of the structure carried by the single isolator is W=2225kN (W=500kip). Analyses were performed with only temperature-dependent friction for comparison to results of program 3pleANI, which is limited to only this option. Additional results of program OpenSees with temperature, velocity and pressure-dependent friction are presented and evaluated. The parameters used in the analysis other than those in Section 4 and below are presented in Appendix A.

Parameter	Unit	Value
Thermal Diffusivity	m²/sec	$0.444 * 10^{-5}$
Thermal Conductivity	$W/(m^{\circ}C)$	18
Initial Temperature	°C	20

Table 6-1 Parameters for accounting for dependency of friction coefficient on temperature

When analyzing with only temperature-dependent friction, equations (2-7) to (2-10) are used but with parameters k_v and k_p set equal to unity. When analyzing with pressure and velocity-dependent friction, equations (2-8) and (2-9) are used with the following values for the initial pressure p_o , and velocity parameter.

 Table 6-2 Values in equations 2-8 and 2-9 to account for dependency of friction on pressure and velocity

Parameter	Unit	Value
Axial Load on Isolator	kN	2225
Initial Pressure at Surfaces 2 and 3	МРа	73.7

Initial Pressure at Surface 1	МРа	36.4
Initial Pressure at Surface 4	МРа	36.4
Velocity Rate Parameter	sec/m	100 (2.54sec/inch)

The ground motion used in analysis consists of the fault-normal (FN), fault-parallel (FP), and vertical (V) components of the ground motion recorded at the Kaminoyama station in the 2011 Tohoku earthquake, scaled in amplitude by factor of 8.0. The ground motion data were obtained from the K-NET (Kyoshin Network) operated by the National Research Institute for Earth Science and Disaster Resilience (NIED) in Japan. This motion is of long duration having a D_{s5-75} duration (Chandramohan et al., 2016) equal to 85.9sec for the FN component, 81.0sec for the FP component, and 82.4sec for the vertical component. Figures 6-1 to 6-3 present histories of the ground acceleration of the three components, after scaling by factor of 8 as used in the analyses, and their normalized Arias intensity histories in order to identify the sections of the D_{s5-75} duration.



Figure 6-1 Scaled 2011 Tohoku Motion at Kaminoyama (FN) and Normalized Arias Intensity



Figure 6-2 Scaled 2011 Tohoku Motion at Kaminoyama (FP) and Normalized Arias Intensity



Figure 6-3 Scaled 2011 Tohoku Motion at Kaminoyama (V) and Normalized Arias Intensity

Results are presented for the following four cases:

- Analysis using only the fault-normal (FN) component of the ground motion and with only temperaturedependent-friction for the two cases of friction of isolator of configuration B. The results of program OpenSees are compared to those of program 3pleANI.
- 2) Analysis using the fault-normal (FN) and vertical components of ground motion and with only temperature-dependent-friction and then again with temperature and pressure-dependent friction and then again with temperature, velocity and pressure-dependent friction. Results produced by program OpenSees are presented and evaluated.

- Analysis using triaxial ground motions and with constant friction and with only temperaturedependent-friction, and then again with temperature, velocity and pressure-dependent friction. Results produced by program OpenSees are presented and evaluated.
- Analysis using triaxial ground motions and with constant friction. Results produced by the two different triple FP bearing elements "TripleFrictionPendulum" and "TripleFrictionPendulumX" in program OpenSees are presented and evaluated.

6.1.1 Analysis with Temperature-Dependent Friction and Horizontal Ground Motion

Results produced by program OpenSees and 3pleANI are compared for the case of only temperaturedependent friction. The scale factor for the FN component is 8.



A) Different Friction Coefficients ($\mu_1 = 0.04$, $\mu_4 = 0.08$, $\mu_2 = \mu_3 = 0.01$)

Figure 6-4 Comparison of Force-Displacement Loops (W=500kip)



Figure 6-5 Comparison of Horizontal Acceleration Histories



Figure 6-6 Comparison of Isolator Displacement Histories



Figure 6-7 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-8 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



B) Same Friction Coefficients ($\mu_1 = \mu_4 = 0.04$, $\mu_2 = \mu_3 = 0.01$)

Figure 6-9 Comparison of Force-Displacement Loops (W=500kip)



Figure 6-10 Comparison of Horizontal Acceleration Histories



Figure 6-11 Comparison of Isolator Displacement History



Figure 6-12 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-13 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

6.1.2 Observations and Comments

For the case of only temperature-dependent friction, the observations made in Section 5 are valid as programs OpenSees and 3pleANI produce results on force-displacement loops, acceleration and displacement histories that are in very good agreement. The histories of temperature on surfaces 1 and 4 differ a little when temperatures are large but that has been explained by the differences in the model of the differences in the friction-temperature relationships in the two programs (see Figure 5-1). Also, there are some differences in the computed histories of temperature on surfaces 2 and 3 which are of the same order as those observed in the analyses in Section 5. These differences have apparently insignificant effects on the important force-displacement loops and the histories of isolator displacement and acceleration.

6.1.3 Analysis with Pressure and Temperature-Dependent Friction and Combined Horizontal-Vertical Ground Motion

Results produced by program OpenSees are compared for the two cases of combined pressure and temperature-dependent friction and only temperature-dependent friction ($\mu(P, T)$ versus $\mu(T)$ in the graphs that follow). The scale factor for FN and V components is 8.



A) Different Friction Coefficients ($\mu_1 = 0.04$, $\mu_4 = 0.08$, $\mu_2 = \mu_3 = 0.01$)

Figure 6-14 Comparison of Force-Displacement Loops (W=500kip)



Figure 6-15 Comparison of Horizontal Acceleration Histories



Figure 6-16 Comparison of Isolator Displacement Histories



Figure 6-17 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-18 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



B) Same Friction Coefficients ($\mu_1 = \mu_4 = 0.04$, $\mu_2 = \mu_3 = 0.01$)

Figure 6-19 Comparison of Force-Displacement Loops (W=500kip)



Figure 6-20 Comparison of Horizontal Acceleration Histories



Figure 6-21 Comparison of Isolator Displacement Histories



Figure 6-22 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-23 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

6.1.4 Observations and Comments

There are insignificant differences in the computed response by program OpenSees when temperaturedependent only and when combined pressure and temperature-dependent friction are considered. The fluctuating vertical load, and thus fluctuating pressure, are not large enough to cause a significant change in behavior. Rather, temperature is important in modifying friction and has an effect on response.

6.1.5 Analysis with Pressure, Velocity and Temperature-Dependent Friction and Combined Horizontal-Vertical Ground Motion

Results produced by program OpenSees are compared for the two cases of combined pressure, velocity and temperature-dependent friction and only temperature-dependent friction ($\mu(P, T, V)$ versus $\mu(T)$ in the graphs that follow). The scale factor for FN and V components is 8.

- Force-Displacement Loop (FN, V) Normalized Force (Force/W) 0.2 0.1 1 ST AL. 0 201 111 -0.1 -0.2 -10 10 -20 0 20 Displacement (in) $OpenSees(\mu = f(T))$ — $OpenSees(\mu = f(P,T,V))$
- A) Different Friction Coefficients ($\mu_1 = 0.04$, $\mu_4 = 0.08$, $\mu_2 = \mu_3 = 0.01$)

Figure 6-24 Comparison of Force-Displacement Loops (W = 500kip)



Figure 6-25 Comparison of Horizontal Acceleration Histories



Figure 6-26 Comparison of Isolator Displacement Histories



Figure 6-27 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-28 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



B) Same Friction Coefficients ($\mu_1 = \mu_4 = 0.04$, $\mu_2 = \mu_3 = 0.01$)

Figure 6-29 Comparison of Force-Displacement Loops (W= 500kip)



Figure 6-30 Comparison of Horizontal Acceleration Histories



Figure 6-31 Comparison of Isolator Displacement Histories



Figure 6-32 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-33 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

6.1.6 Observations and Comments

Including the velocity dependence in the model of friction did not have any important change in the computed response for the analyzed rigid structure (it could have for the accelerations in a flexible multidegree-of-freedom structure). The only important observation has to do with permanent displacements which are less when velocity-dependent friction is utilized. This is illustrated for the case of $\mu_1 \neq \mu_4$ for which the force-displacement loop and displacement history of the isolator are shown in Figures 6-24 and 6-26, respectively. Figure 6-34 below compares the displacement histories for surfaces 1 and 4 (unlike Figure 6-26 which shows the displacement of the top of the bearing with respect to its bottom). For surface 1 of friction coefficient equal to 0.04, there is insignificant permanent offset computed with both models. However, for surface 4 of higher friction (=0.08) there is permanent offset when the velocity dependence of friction is ignored.





Figure 6-34 Displacement History on Surface 1 and 4 in case of unequal friction $\mu_1 \neq \mu_4$

It is interesting to observe the instantaneous values of friction computed during the analysis by the two models and shown in Figure 6-35. In the $\mu(P,T,V)$ model, there is a rapid fluctuation of the friction coefficient value between the lower and upper limits as the sliding velocity varies from zero to its maximum value in every cycle of motion. In the $\mu(T)$ model, the friction coefficient varies much less and smoothly with temperature. Evidently, consideration of the velocity dependence of friction is important when permanent offsets are assessed.



Figure 6-35 Friction Coefficient on Surface 1 and 4 in case of unequal friction $\mu_1 \neq \mu_4$

6.1.7 Analysis with Pressure, Velocity and Temperature-Dependent Friction and Triaxial Ground Motion

Results produced by program OpenSees are compared for the three cases of combined pressure, velocity and temperature-dependent friction, only temperature-dependent friction, and Coulomb friction ($\mu(P, T, V)$ versus $\mu(T)$ versus $\mu = Coulomb$ in the graphs that follow). In the analysis, the FN and FP components are used in X and Y directions, respectively. The scale factor for the FN, FP, and V components is 7. Moreover, the isolators are assumed to be without an interior ring or displacement restrainer so that the total displacement capacity is increased by $b_2/2$ and $u_{limit}=26.2$ in. (see Table 2-1).

A) Different Friction Coefficients ($\mu_1 = 0.04$, $\mu_4 = 0.08$, $\mu_2 = \mu_3 = 0.01$)



Figure 6-36 Comparison of Force-Displacement Loops in X direction for Three Cases of Friction





Figure 6-38 Comparison of Force-Displacement Loops in X direction (W = 500kip)



Figure 6-39 Comparison of Force-Displacement Loops in Y direction (W = 500kip)



Figure 6-40 Comparison of Isolator Displacement Orbits (Circle shows *u*_{limit}=26.2in)





Figure 6-41 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-42 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)





Figure 6-43 Comparison of Force-Displacement Loops in X direction for Three Cases of Friction (W = 500kip)



Figure 6-44 Comparison of Force-Displacement Loops in Y direction for Three Cases of Friction (W = 500kip)



Figure 6-45 Comparison of Force-Displacement Loops in X direction (W = 500kip)



Figure 6-46 Comparison of Force-Displacement Loops in Y direction (W = 500kip)



Figure 6-47 Comparison of Isolator Displacement Orbits (Circle shows *u*_{limit}=26.2in)


Figure 6-48 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-49 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

6.1.8 Observations and Comments

Overall, observations and comments made in subsection 6.1.6 are still valid. The most important parameter that affects the response is the temperature dependence of the coefficient of friction, whereas the pressure and velocity effects are minor. As expected, the triaxial ground motion results in increases in the isolator response in terms of displacements and temperature at the sliding interfaces. The increase in displacement demand results in the isolators entering regime V (high stiffness) so that the shear force is significantly increased.

6.1.9 Comparison of Results Produced by Elements TripleFrictionPendulum and **TripleFrictionPendulumX in OpenSees**

Results for triaxial ground motion produced by TripleFrictionPendulum and TripleFrictionPendulumX elements in program OpenSees are compared in the case of constant friction coefficients when the two elements should produce identical results. In the analysis, the FN and FP components are used in X and Y directions, respectively. The temperature is calculated only by element TripleFrictionPendulumX and its histories are presented. The scale factor for FN, FP, and V components is 7.



-5

0

Displacement (in) TripleFrictionPendulum (μ = Coulomb) ---- TripleFrictionPendulumX (μ = Coulomb)

5

10

15

20

A) Different Friction Coefficients ($\mu_1 = 0.04$, $\mu_4 = 0.08$, $\mu_2 = \mu_3 = 0.01$)

-15

-10



Figure 6-50 Comparison of Force-Displacement Loops (W = 500kip)



Figure 6-51 Comparison of Isolator Displacement Orbits (Circle shows ulimit=26.2in)



Figure 6-52 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-53 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)



A) Same Friction Coefficients ($\mu_1 = \mu_4 = 0.04$, $\mu_2 = \mu_3 = 0.01$)

Figure 6-54 Comparison of Force-Displacement Loops (W= 500kip)



Figure 6-55 Comparison of Isolator Displacement Orbits (Circle shows u_{limit} =26.2in)



Figure 6-56 Comparison of Temperature Histories on Outer Surfaces (Surfaces 1 and 4)



Figure 6-57 Comparison of Temperature Histories on Inner Surfaces (Surfaces 2 and 3)

6.1.10 Observations and Comments

Elements TripleFrictionPendulum and TripleFrictionPendulumX in program OpenSees produce identical results in the case of constant friction coefficients.

SECTION 7

IMPLEMENTING NEW TRIPLE FP BEARING ELEMENT IN PROGRAM OPENSEES

Element TripleFrictionPendulumX has not been implemented in program OpenSees for direct use. Rather, interested users need to obtain the source code of program OpenSees and attach to it the source code for element TripleFrictionPendulumX. Details on how to do this and the source code for element TripleFrictionPendulumX are provided in a digital appendix entitled IMPLEMENTING "TRIPLEFRICTIONPENDULUMX" ELEMENT IN OPENSEES which can be obtained from the authors.

SECTION 8 SUMMARY AND CONCLUSIONS

This report presented an enhancement of the theory for the Triple FP isolators presented by Fenz et al (2018a, b, c, d, 2019) which was used in the implementation of a Triple FP element in program OpenSees (Dao et al., 2013). The enhanced theory computes the displacement and velocity histories at each of the four sliding interfaces of the isolator which are then used to introduce dependence of the coefficient of friction on axial pressure, sliding velocity, and temperature. The Dao et al. (2013) element in program OpenSees has been modified to introduce dependencies on pressure, velocity and temperature. The dependency on temperature is a new development, whereas the dependency on velocity is direct and based on the actual sliding velocities rather than based on the partitioning of velocity of the Fenz et al. (2008a, b, c, d, 2019) model.

Several examples have been presented in which results obtained by this model in program OpenSees were compared with results obtained by a much more complex and advanced model implemented in program 3pleANI (Sarlis et al., 2013, 2016). There is good agreement in the results in terms of displacement, velocity and temperature histories at the four sliding interfaces, and of force-displacement loops. Furthermore, the enhanced Triple FP element in OpenSees has been verified.

SECTION 9

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APPENDIX A

PARAMETERS USED IN ANALYSIS IN PROGRAM OPENSEES

Table A-1 presents values of the parameters used in each example presented in Sections 4, 5 and 6. All parameters are defined in the report except for the following: Y is yield displacement in the friction model; dt_{load} is loading time step; dt is analysis time step; Tol is a convergence tolerance parameter utilized by the Newton-Raphson method that is used in the numerical solution; a is friction velocity rate parameter (see equation 2-3).

Section	Model	Motion	Isolator Configuration	Model Parameters	
			Configuration A $(\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec, \\ dt = 0.001sec, Tol = 10^{-5}$	
4.1.1	Constant <i>u</i>	Amplitude = 20in	Configuration A $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec,$ $dt = 0.001sec, Tol = 10^{-5}$	
		Period = 20sec	Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01$)	$Y = 0.04 \text{ in } dt_{\text{load}} = 0.001 \text{ sec},$ $dt = 0.001 \text{ sec}, \text{ Tol} = 10^{-5}$	
			Configuration B $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04 \text{ in } dt_{\text{load}} = 0.001 \text{ sec},$ $dt = 0.001 \text{ sec}, \text{ Tol} = 10^{-5}$	
			Configuration A $(\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec,$ $dt = 0.001sec, Tol = 10^{-5}$	
412	Constant u	Amplitude = 22in Period = 10sec	Constant <i>u</i> Amplitude = 22in	Configuration A $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec,$ $dt = 0.001sec, Tol = 10^{-5}$
7.1.2	Period = $10 \sec$		Configuration B $(\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec,$ $dt = 0.001sec, Tol = 10^{-5}$	
		Configuration B $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec, \\ dt = 0.001sec, Tol = 10^{-5}$		

Table A-1 Parameters Used in Analysis

			Configuration A $(\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec, \\ dt = 0.001sec, Tol = 10^{-5}$
413	Constant u	Amplitude = 20in	Configuration A $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec, \\ dt = 0.001sec, Tol = 10^{-5}$
		Period = 5sec	Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01$)	$Y = 0.04in, dt_{load} = 0.001sec,$ $dt = 0.001sec, Tol = 10^{-5}$
			Configuration B $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec, \\ dt = 0.001sec, Tol = 10^{-5}$
			Configuration A $(\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01)$	$\begin{split} Y &= 0.04 \text{in}, dt_{\text{load}} = 0.0167 \text{sec}, \\ dt &= 0.0167 \text{sec}, \text{Tol} = 10^{-5} \end{split}$
5.1.1	и(Т)	Amplitude = 20in	Configuration A $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$\begin{split} Y &= 0.04 \text{in}, dt_{\text{load}} = 0.0167 \text{sec}, \\ dt &= 0.0167 \text{sec}, \text{Tol} = 10^{-5} \end{split}$
5.1.1		Period = 20sec	Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08,$ $\mu_2 = \mu_3 = 0.01$)	$\begin{split} Y &= 0.04 \text{in}, dt_{\text{load}} = 0.0167 \text{sec}, \\ dt &= 0.0167 \text{sec}, \text{Tol} = 10^{-5} \end{split}$
			Configuration B $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.0167sec, \\ dt = 0.0167sec, Tol = 10^{-5}$
			Configuration A $(\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.0167sec, \\ dt = 0.0167sec, Tol = 10^{-5}$
5.1.2	μ(Τ)	Amplitude = 22in Period = 10sec	Configuration A $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$\begin{split} Y &= 0.04 \text{in}, dt_{\text{load}} = 0.0167 \text{sec}, \\ dt &= 0.0167 \text{sec}, \text{Tol} = 10^{-5} \end{split}$
			Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01$)	$Y = 0.04in, dt_{load} = 0.001sec, \\ dt = 0.0167sec, Tol = 10^{-5}$
			Configuration B $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in$, $dt_{load} = 0.001sec$, $dt = 0.0167sec$, $Tol = 10^{-5}$
5.1.3	μ(Τ)	Amplitude = 20in Period = 5sec	Configuration A $(\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec,$ $dt = 0.008sec, Tol = 10^{-5}$

			Configuration A $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec, \\ dt = 0.001sec, Tol = 10^{-5}$
			Configuration B $(\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec, \\ dt = 0.008sec, Tol = 10^{-5}$
			Configuration B $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	$Y = 0.04in, dt_{load} = 0.001sec,$ $dt = 0.001sec, Tol = 10^{-5}$
6.1.1	$\mu(T)$	2011 Tohoku Earthquake at Kaminoyama	Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01$)	Y = 0.01 in, $dt_{load} = 0.001$ sec, $dt = 0.0016$ sec, $Tol = 10^{-3}$
		station (FN, Scaled by 8)	Configuration B $(\mu_1 = \mu_4 = 0.04,$ $\mu_2 = \mu_3 = 0.01)$	Y = 0.01in, $dt_{load} = 0.001$ sec, $dt = 0.0016$ sec, $Tol = 10^{-3}$
6.1.3	μ(Τ)	2011 Tohoku Earthquake at Kaminoyama	Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01$)	Y = 0.01 in, $dt_{load} = 0.001 \text{ sec},$ $dt = 0.004 \text{ sec}, \text{ Tol} = 10^{-3}$
0.1.5	and $\mu(P,T)$	station (FN and V, Scaled by 8)	Configuration B $(\mu_1 = \mu_4 = 0.04, \mu_2 = \mu_3 = 0.01)$	Y = 0.01 in, $dt_{load} = 0.001$ sec, $dt = 0.004$ sec, $Tol = 10^{-3}$
615	$\mu(T)$ and	2011 Tohoku Earthquake at Kaminoyama	Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01$)	Y = 0.01in, $dt_{load} = 0.001$ sec, $a = 2.54$ sec/inch (100sec/m) $dt = 0.006$ sec, $Tol = 10^{-3}$
0.1.5	$\mu(P,T,V)$	station (FN and V, Scaled by 8)	Configuration B ($\mu_1 = \mu_4 = 0.04$, $\mu_2 = \mu_3 = 0.01$)	Y = 0.01in, $dt_{load} = 0.001$ sec, $a = 2.54$ sec/inch (100sec/m) $dt = 0.004$ sec, $Tol = 10^{-3}$
6.1.7	Constant μ and $\mu(T)$	2011 Tohoku Earthquake at Kaminoyama	Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01$)	Y = 0.01in, $dt_{load} = 0.001$ sec, $a = 2.54$ sec/inch (100sec/m) $dt = 0.006$ sec, $Tol = 10^{-3}$
0.1.7	and $\mu(P,T,V)$	station (FN, FP, and V, Scaled by 7)	Configuration B ($\mu_1 = \mu_4 = 0.04$, $\mu_2 = \mu_3 = 0.01$)	Y = 0.01in, $dt_{load} = 0.001$ sec, $a = 2.54$ sec/inch (100sec/m) $dt = 0.004$ sec, $Tol = 10^{-3}$

6.1.9	Constant μ	2011 Tohoku Earthquake at Kaminoyama	Configuration B ($\mu_1 = 0.04, \mu_4 = 0.08, \mu_2 = \mu_3 = 0.01$)	Y = 0.01 in, dt _{load} = 0.001 sec, a = 2.54 sec/inch (100 sec/m) dt = 0.001 sec, Tol = 10 ⁻³
		station (FN, FP, and V, Scaled by 7)	Configuration B ($\mu_1 = \mu_4 = 0.04$, $\mu_2 = \mu_3 = 0.01$)	Y = 0.01in, dt _{load} = 0.001sec, a = 2.54sec/inch (100sec/m) dt = 0.001sec, Tol = 10 ⁻³

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