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Scissor-Jack-Damper Energy Dissipation System

by

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Preface

The Multidisciplinary Center for Earthquake Engineering Research (MCEER) is a national center of excellence in advanced technology applications that is dedicated to the reduction of earthquake losses nationwide. Headquartered at the University at Buffalo, State University of New York, the Center was originally established by the National Science Foundation in 1986, as the National Center for Earthquake Engineering Research (NCEER).

Comprising a consortium of researchers from numerous disciplines and institutions throughout the United States, the Center's mission is to reduce earthquake losses through research and the application of advanced technologies that improve engineering, preearthquake planning and post-earthquake recovery strategies. Toward this end, the Center coordinates a nationwide program of multidisciplinary team research, education and outreach activities.

MCEER's research is conducted under the sponsorship of two major federal agencies: the National Science Foundation (NSF) and the Federal Highway Administration (FHWA), and the State of New York. Significant support is derived from the Federal Emergency Management Agency (FEMA), other state governments, academic institutions, foreign governments and private industry.

MCEER's NSF-sponsored research objectives are twofold: to increase resilience by developing seismic evaluation and rehabilitation strategies for the post-disaster facilities and systems (hospitals, electrical and water lifelines, and bridges and highways) that society expects to be operational following an earthquake; and to further enhance resilience by developing improved emergency management capabilities to ensure an effective response and recovery following the earthquake (see the figure below).



A cross-program activity focuses on the establishment of an effective experimental and analytical network to facilitate the exchange of information between researchers located in various institutions across the country. These are complemented by, and integrated with, other MCEER activities in education, outreach, technology transfer, and industry partnerships.

This report describes an energy dissipation system configuration that extends the utility of fluid viscous damping devices to structural systems that are characterized by small interstory drifts and velocities. The geometry of the brace and damper assembly is such that the system resembles a jacking mechanism, and thus the name "scissor-jack-damper energy dissipation system" is adopted. The system is a variant of the toggle-brace-damper system, and offers the advantage of a more compact configuration. A theoretical treatment of the scissor-jack-damper system is presented and its effectiveness is demonstrated through testing of a large-scale steel framed model structure under imposed harmonic displacement on the strong floor, as well as dynamic excitations on the earthquake simulator. Experiments demonstrate that despite its small size, the scissor-jack system provides a significant amount of damping while also substantially reducing the seismic response of the tested structure. Comparisons of response-history and simplified analyses with the experimental findings produce results that are consistent. Application of the scissor-jack-damper system in a new building structure in Cyprus is described.

ABSTRACT

Installation of damping devices has been limited to diagonal or chevron brace configurations until the recent development of the toggle-brace configurations. These configurations magnify the effect of damping devices, thus facilitating their use in stiff framing systems. Such systems are not good candidates for supplemental damping when conventional diagonal or chevron brace configurations are used, due to the high cost of the damping system. This report introduces the scissor-jack-damper system that was developed as a variant of the toggle-brace-damper systems. An additional advantage of the scissor-jack-damper to the toggle-brace-damper system is in the compactness of the configuration. A theoretical treatment of the scissor-jack-damper system is presented and the effectiveness is demonstrated through testing of a large-scale steel framed model structure under imposed harmonic displacement on the strong floor, as well as dynamic excitations on the earthquake simulator. Experiments demonstrate that despite the small size of the damping device considered, the scissor-jack system provided a considerably significant amount of damping while also substantially reducing the seismic response of the tested structure. Comparisons of response-history and simplified analyses with the experimental findings produce results that are consistent. Application of the scissor-jack-damper system in a new building structure in Cyprus is described.

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SECTION 1 INTRODUCTION

1.1 Passive Energy Dissipation Systems

Conventional methods of seismic design rely on ductile behavior of structural members for energy dissipation. Such design is based on the principle that inelastic or nonlinear behavior will take place in selected components of the framing system, in the form of localized and/or spread plastic hinges. Examples include hinging in beams adjacent to the beam-to-column connections in the widely used moment-resisting frame, buckling of braces in a concentrically braced frame, and yielding of shear links in an eccentrically braced frame. The main disadvantage associated with conventional earthquake-resistant design is that all inelastic action (thus energy dissipation) is provided by elements that form part of the gravity-load-resisting system. Any damage to the gravity-load-resisting system as a result of inelastic action is typically costly, or may be irreparable.

In the past decade, the use of supplemental damping devices in building (and bridge) structures has become an increasingly popular approach to remedy the deficiencies inherent in conventional seismic design. These devices, commonly known as 'dampers', dissipate earthquake-induced energy through either hysteretic action (e.g., yielding of metals, sliding friction) or viscoelastic/viscous action (e.g., fluid viscous dampers, solid and fluid viscoelastic dampers). In comparison with conventional earthquake-resistant design, the underlying objective of implementing energy dissipation devices in structural systems is to limit or eliminate damage to the structural frame by dissipating most of the earthquake-induced energy, which would otherwise be absorbed by the load-bearing-system through inelastic deformations. An additional advantage related to the use of energy dissipation devices is that they can be replaced relatively easily after a major seismic event. Besides earthquake protection, viscoelastic and viscous energy dissipation systems are eminently suitable for reducing wind-induced vibrations. The interested reader is referred to the following for a comprehensive review of this technology: Federal Emergency Management Agency (1997), Soong and Dargush (1997), Constantinou et al. (1998), and Hanson and Soong (2001).

Today, many countries utilize various types of damping devices as protective systems. The

advantages offered by supplemental energy dissipation systems and the increasing number of applications utilizing these systems in the design or retrofit of building structures have spurred the need for systematic, robust and validated guidelines for the modeling, analysis, design and testing of various damping devices. Extensive analytical and experimental studies have been conducted that have investigated various kinds of supplemental damping systems, leading to a better understanding of the characteristics of damping devices and their effects on the earthquake response of structures. Analysis and design tools towards implementing these systems in the construction of new earthquake-resistant structures, as well as in the retrofit of existing structures for improved seismic performance have subsequently been proposed. Furthermore, several code-oriented documents, provisions and guidelines, on the design, testing and incorporation of damping devices in building structures have been developed. The most up-to-date of these publications are those of the Federal Emergency Management Agency (FEMA 273 Guidelines and FEMA 274 Commentary 1997, FEMA 356 Prestandard and Commentary 2000, FEMA 368 Provisions and FEMA 369 Commentary 2000, and the upcoming FEMA 2003), which contain the latest analysis and design guidelines for buildings with energy dissipation systems, as well as with seismic isolation systems. An overview of the code-oriented procedures related to the implementation of passive energy dissipation devices in building structures developed since the 1990s can be found in the recent work of Ramirez (Ramirez et al. 2001). This work concentrates on analysis and design procedures for displacement- and velocity-dependent dampers in new buildings and was utilized in writing the section on Structures with Damping Systems of FEMA 368 (2000) and the upcoming FEMA (2003).

The application of seismic energy dissipation systems differs in various countries. In Japan, the majority of the applications utilize yielding steel devices and viscoelastic fluid or solid devices. In the United States, engineers have primarily used fluid viscous dampers. In all of these applications, damping devices have either been installed in-line with diagonal bracing or as horizontal elements atop chevron bracing (Soong and Dargush 1997, Constantinou et al. 1998). The popularity of these configurations is based on the engineers' familiarity with such bracing systems in steel construction and the fact that all experimental research studies have utilized only these two configurations for energy dissipation systems.

Stiff structural systems such as reinforced concrete shear wall or steel-braced dual systems undergo small interstory drifts and velocities when subjected to dynamic excitation. Since

significant drifts and velocities are required for effective energy dissipation, it may appear that such systems are not suitable for the addition of damping devices. This observation is valid in the case of conventional damper configurations involving diagonal or chevron installations, in which, the damper displacement is less than or equal to the interstory drift. For example, in a stiff code-compliant building, interstory displacements will likely not exceed 15 mm (0.6 in) in the design earthquake. Conventionally configured damping systems in such a building will require large damper forces for moderate levels of supplemental damping, leading to an increase in the cost of the damping system. In addition, small-stroke damping devices require special detailing, which further increases their size and therefore, their cost.

Given that the major shortcomings associated with conventional configurations of supplemental damping systems in stiff structures are due to small interstory drifts and velocities, one might reasonably ponder the possibility of using non-traditional configurations that can magnify the damper displacement for a given interstory drift. Such magnification allows for the use of dampers with smaller force outputs (smaller damper volume) and larger strokes, resulting in reduced cost. These configurations can be used for stiff and flexible framing systems, as well as for limiting vibrations caused by wind.

A variety of mechanisms that can magnify displacements can be inspired by experiences in other disciplines, especially mechanical engineering. One may find it helpful to review one of the many publications with illustrations of concepts and devices in this field (e.g., Chironis 1991). While magnifying mechanisms are widely utilized in the construction and operation of machinery, their use in applications of earthquake and wind vibration protection of structures, however, is a novel approach.

1.2 Scope and Organization of this Report

This report describes an energy dissipation system configuration that extends the utility of fluid viscous damping devices to structural systems that are characterized by small interstory drifts and velocities. The geometry of the brace and damper æsembly is such that the system resembles a jacking mechanism, and thus the name "scissor-jack-damper energy dissipation system" is adopted. The development of this configuration followed that of the toggle-brace-damper system, also developed and tested at the University at Buffalo (Hammel 1997,

Constantinou et al. 1997, and Constantinou et al. 2001). Both systems utilize innovative mechanisms to amplify displacements and accordingly lower force demands in the energy dissipation devices. The magnifying mechanism in turn amplifies the damper force through its shallow truss configuration and delivers it to the structural frame. In addition to overcoming the limitations related to small drifts, the scissor-jack-damper system allows for open space due to its compact geometry and is therefore desirable architecturally.

This report consists of eight sections, followed by a list of references and the appendices. The concept underlying the scissor-jack-damper system and discussions on various issues that affect its behavior are presented in Section 2. Section 3 describes the experimental program, which includes strong floor tests with imposed cyclic displacement and earthquake-simulator testing. Sample results from the experimental program are given in Section 4. Section 5 focuses on the analytical modeling of the tested structure and prediction of response using the program SAP2000. Section 6 presents an overview of simplified analysis methods for structures with added damping systems, and illustrates the application of these methods to the tested model to estimate its fundamental period and damping ratio. Prediction of peak dynamic response from displacement and acceleration response spectra of ground motions is also illustrated in this section. Section 7 introduces the first application of the scissor-jack-damper system in the design of a new building structure in Cyprus. Summary and conclusions are outlined in Section 8. References and four appendices (including manufacturing drawings, strong floor and earthquake simulator test results and an input file for response-history analysis) follow. Parts of the work described in this report have previously been presented or briefly described in several conferences and publications (Whittaker and Constantinou 1999a, 1999b and 2000, Constantinou et al. 2000, Constantinou 2000, Constantinou and Sigaher 2000, and Hanson and Soong 2001). The first detailed publication on the scissor-jack-damper system is Sigaher and Constantinou (2003), which essentially represents a summary of this report.

SECTION 2

SCISSOR-JACK-DAMPER ENERGY DISSIPATION SYSTEM

2.1 Energy Dissipation Systems with Magnifying Mechanisms

It is possible to encounter in the field of mechanics and engineering a number of configurations that magnify displacements and hence can be implemented in energy dissipation systems for stiff structures.

The DREAMY damping system, developed by the Taisei Corporation in Japan, utilizes such a magnifying mechanism (Hibino et al. 1989). As illustrated in Figure 2-1, this system makes use of the lever principle to magnify displacements (the deformed configuration in Figure 2-1 is valid provided that the bracing is rigid). Also shown in Figure 2-1 are the relationships between the lateral interstory drift u and the damper deformation u_D , and the damping shear force F and the damper force F_D , in which f denotes the displacement magnification factor. The DREAMY system is simple in concept and is functional. The drawbacks, however, are the sizeable dimensions, large sections (due to the presence of bending forces) and details involved in the pin connections, which make this system cumbersome to construct.

Kani et al. (1992) have designed a similar system comprised of an inverted T-shaped lever and a pair of fluid dampers, which amplify the damping effect.

Another energy dissipation system that makes use of a magnifying technique is the "coupled truss and damping system" that was utilized in the construction of the 57-story Torre Mayor building in Mexico City (Rahimian 2002, Taylor 2003). Developed in the United States by the Cantor Seinuk Group, this system includes a damping mechanism between a pair of vertical trusses, which primarily deflect in cantilever mode under an external load. To illustrate the concept, Figure 2-2 depicts a simplified model of a floor from a multi-story, high-rise building. The springs represent the effect of the axial flexibility of the stacked columns supporting the floor. Upon an external force, both trusses move in cantilever motion (angle θ) so that the truss columns *AC* and *BD* are displaced vertically in opposite directions (note the presence of vertical displacement *v* in addition to lateral drift *u*). Nodes *C* and *B* at each end of the damper element thus move through a longer relative distance, which results in a larger damper stroke in comparison to conventional diagonal configurations (i.e., without the truss systems). The desired



Figure 2-1 Illustration of DREAMY System of Taisei Corporation



 $F = \cos \alpha \cdot F_D$

Figure 2-2 Illustration of Coupled Truss Systems with Damping

cantilever action can also be achieved with the use of solid walls. In the Torre Mayor, a variation of the configuration shown in Figure 2-2 was adapted such that the dampers were placed in-line with diagonal braces, which were configured as diamonds along the height of the building. Since the efficiency of the coupled truss and damping system increases at the upper levels of a building (the effect of the axial flexibility of the supporting columns is more pronounced at upper levels), the system is suitable mostly for high-rise buildings.

In the United States, the recent construction of one 38-story and two 37-story buildings utilizing a "toggle-brace" configuration has been an exception to the custom of using diagonal or chevron brace configurations for damping systems. The toggle-brace-damper system was developed and tested at the University at Buffalo, and is illustrated in Figure 2-3.



Figure 2-3 Illustration of Toggle-Brace-Damper System

As the name implies, this configuration operates based on the toggle mechanism (toggles *ABC* in Figure 2-3), which amplifies the damper displacement for a given interstory drift. This amplification results in reductions in the required damping force and damper size, which may lead to cost savings. The damper force output is magnified through the toggle mechanism and delivered to the framing system by compression or tension in the braces. The toggle-brace configuration is suitable for applications of wind-response reduction and seismic risk mitigation for stiff structures. A theoretical treatment of the system's behavior, along with experimental results confirming the validity of the concept and the developed theory can be found in Hammel (1997), Constantinou et al. (1997), and Constantinou et al. (2001). The last of these references also provides a brief description of the applications in the United States.

An additional consideration related to the application of energy dissipation systems is that in many cases the energy dissipation assemblies occupy entire bays in frames and often violate architectural requirements such as open space and unobstructed view. With the intent of providing an architecturally attractive solution, the scissor-jack-damper system was developed as a variant of the toggle-brace-damper system. The scissor-jack configuration combines the displacement magnification feature with small size, which is achieved through compactness and a near vertical installation.

2.2 Scissor-Jack-Damper Theory

The scissor-jack-damper system is best explained by first reviewing the conventional diagonal and chevron brace configurations, in which the displacement of the energy dissipation devices is either less than (case of diagonal brace) or equal to (case of chevron brace) the drift of the story at which the devices are installed. Consistent with the previous notations of Figures 2-1 to 2-3 if u and u_D denote the interstory drift and the damper relative displacement, respectively, then

$$u_D = f \cdot u \tag{2-1}$$

where f = magnification factor. For the chevron brace configuration, f = 1.0; for the diagonal configuration $f = \cos \theta$, where $\theta = \text{angle of inclination of the damper with respect to the horizontal axis}. The force <math>F_D$ along the damper axis is similarly related to F, the horizontal component of the damper force exerted on the frame at the degree of freedom u through

$$F = f \cdot F_D \tag{2-2}$$

Figure 2-4 illustrates a single-story structure with diagonal and chevron brace configurations. Also shown in the figure are the force F, and the interstory drift u. Consider that this single-story structure has an effective weight W and a fundamental period under elastic conditions T, and that it is equipped with a linear fluid viscous damper for which

$$F_D = C_o \cdot \dot{u}_D \tag{2-3}$$

where C_o = damping coefficient and \dot{u}_D = relative velocity between the ends of the damper along its axis. The damping force *F*, exerted on the frame by the damper assembly is given by

$$F = C_o \cdot f^2 \cdot \dot{u} \tag{2-4}$$

in which \dot{u} = interstory velocity. It follows that the damping ratio of a single-story frame with a linear fluid viscous device can be written as

$$\beta = \frac{C_o \cdot f^2 \cdot g \cdot T}{4 \cdot \pi \cdot W} \tag{2-5}$$

It is essential to realize the effect of the magnification factor on the damping ratio. As (2-5) suggests, the damping ratio varies proportionally with the square of the magnification factor. In the two conventional configurations in Figure 2-4, a damper designed to provide a damping ratio of 5-percent of critical when installed horizontally (chevron brace), will provide a damping ratio of 3.2-percent of critical in the diagonal configuration.

In contrast to the familiar diagonal and chevron brace configurations, the scissor- jack configuration can achieve magnification factors substantially greater than unity. This is also true for the toggle-brace-damper systems. Figure 2-4 also illustrates the scissor-jack-damper and toggle-brace-damper systems as implemented in a single-story frame. These systems make use of shallow trusses that amplify the effect of the interstory drift on the damper displacement and also amplify the small damper force and deliver it to the structural frame. The expression for the magnification factor f under the assumption of small rotations, and its value for a typical geometry are also given in Figure 2-4. A substantial increase in the damping ratio with respect to that provided by conventional damper configurations demonstrates the efficacy of these systems.



Figure 2-4 Illustration of Diagonal, Chevron Brace, Scissor-Jack-Damper, and Toggle-Brace-Damper Configurations, Magnification Factors, and Damping Ratios of Single-Story Structure with Linear Fluid Viscous Devices

The presence of the magnifying mechanism in the scissor-jack system extends the utility of fluid viscous devices to cases of small interstory drifts and velocities, which are typical of stiff structural systems under seismic excitation and structures subjected to wind load. Also, the absence of bending in the system allows the use of small sections and standard connection details. This damper configuration, therefore, may lead to cost savings, provided that the cost of the scissor-jack support framing is not substantially greater than the cost of the framing that would be required to support the dampers in conventional configurations. In addition to the displacement magnification, the scissor-jack system may be configured to allow for open space, minimal obstruction of view and slender configuration, features that are often desired by architects. Figure 2-5 illustrates various possible installation configurations of a scissor-jack damping system.



Figure 2-5 Possible Installation Configurations of Scissor-Jack Damping System

2.3 Magnification Factor and Forces in Scissor-Jack System

The effectiveness of the scissor-jack configuration is based on the magnification factor f, defined as the ratio of damper displacement, u_D , to the interstory drift, u. Figure 2-6 presents an analysis of the movement of a single-story frame with a scissor-jack system. The magnification factor is

$$f = \frac{u_D}{u} = \frac{\left|\overline{A'B'} - \overline{AB}\right|}{u}$$
(2-6)

where \overline{AB} and $\overline{A'B'}$ denote the initial and the deformed lengths of the damper, respectively.

It should be noted that the deformed configuration of Figure 2-6 does not take into account any deformations in the frame and any reduction in height due to column extensibility. This is also true for the configurations presented in Figure 2-4 (and eqns. 2-1 to 2-6), in which the damper displacement u_D relates only to the lateral interstory drift u. The column extensibility has negligible effect on the magnification factor for typical values of interstory drift and for low-rise structures. For high-rise structures, column inextensibility cannot be assumed. Deformations due to column extensibility will cause a decrease in the magnification factor for the configuration factor for the supporting columns.

Deformations due to frame action may have notable effect on the magnification factor, regardless of the height of the structure. As an example, consider that the beam in Figure 2-6 is simply connected to the column on the left and rigidly connected to the column on the right. Upon an interstory drift towards the right, the beam will deflect upwards, causing a decrease in the damper deformation and thus, in the magnification factor. The opposite will occur when the beam-to-column connections are reversed. This type of amplification/deamplification of the magnification factor will depend on the relative stiffnesses of the beam and the column, and the position of the point of connection of the scissor-jack on the beam. The effect of the frame deformations will be observed in the test results presented in Section 4. It follows that vertical frame deformations will not affect the damper displacement when the damping system extends between the beam-to-column joints, such as the diagonal, chevron and reverse toggle



Figure 2-6 Analysis of Scissor-Jack Movement and Analysis of Forces

configurations shown in Figure 2-4.

Whether caused by frame deformations, column rotation and axial flexibility of the supporting columns, vertical deformations may alter the damper displacement and hence the magnification factor of the scissor-jack-damper system. Simple analytical expressions can be written to quantify the effect of vertical deformations on the damper displacement, which will be presented in Section 2.5. The following derivations however, concentrate only on the lateral interstory drift u.

In addition to vertical displacements, the displacements due to the forces in the damper and in the scissor-braces (i.e., displacements due to finite stiffness of the scissor-braces) will reduce the magnification factor, regardless of the structural system configuration. This issue will be further explained in the following sections (Sections 2.6 and 6), and its effects will be observed in the test results presented in Section 4.

Based on rigid body kinematics, the damper displacement may be expressed as

$$u_D = \left| \overline{A'B'} - \overline{AB} \right| = \pm 2 \cdot \ell_1 \cdot \left[\sin(\theta \pm \Delta \theta) - \sin \theta \right]$$
(2-7)

where $\Delta \theta$ = angle of rotation of the scissor-braces. Preservation of lengths between points *C* and *D* requires that

$$2 \cdot \ell_1 \cdot \cos(\theta \pm \Delta \theta) = 2 \cdot \ell_1 \cdot \cos \theta \mp u \cdot \cos \psi$$
(2-8)

It must be noted that in writing eq. (2-8), the change in the angle ψ , $\Delta \psi$, is not taken into account. From the deformed configuration of Figure 2-6, one can recognize that $\Delta \psi$ per unit ψ equals the drift ratio, u / h. The drift ratio is typically $u / h \le 0.01$. For example for $u / h \approx 0.01$, $\Delta \psi \approx 0.01$ rad for $\psi \approx 1$ rad, or $\Delta \psi \approx 0.01 \cdot \psi$. Therefore, $\Delta \psi$ is negligible with respect to $\Delta \theta$ and is not included in the analysis of the scissor-jack movement.

Utilizing eqns. (2-7) and (2-8), the damper displacement and the angle of rotation can be written as

$$u_D = \pm 2 \cdot \ell_1 \cdot \left[\sin \left(\cos^{-1} \left\{ \cos \theta \mp \frac{\cos \psi}{2 \cdot \ell_1} \cdot u \right\} \right) - \sin \theta \right]$$
(2-9)

$$\pm \Delta \theta = \cos^{-1} \left(\cos \theta \mp \frac{\cos \psi}{2 \cdot \ell_1} \cdot u \right) - \theta$$
(2-10)

In eqns. (2-7) to (2-10), positive signs hold for drift towards the right (*u* and $\Delta\theta$ as shown in Figure 2-6) and for damper extension ($u_D > 0$). For drift towards the left, these equations are valid with negative signs.

Equation (2-9) may be used to calculate the damper displacement given a value of drift (provided the latter is small), which is presented in the following subsection. However, the equation cannot

be solved for the ratio of two displacements, which is of much practical value. Realizing that for most applications $\Delta\theta$ is very small and *u* is small in comparison to the dimensions, eqns. (2-7) and (2-8) may be significantly simplified to yield the magnification factor

$$f = \frac{\cos\psi}{\tan\theta} \tag{2-11}$$

It can be shown that eqns. (2-1) and (2-11) provide a very good approximation to the damper deformation given by (2-9) for $\Delta \theta \leq 0.2 \cdot \theta$. Moreover, (2-11) provides insight into the major factors affecting the performance of the scissor-jack configuration.

The dependence of the magnification factor f on angles θ and ψ is illustrated in Figure 2-7. As the figure suggests, the magnification factor assumes very large values as θ approaches 0° ; but this has no meaning since the scissors tend to act as a single brace inclined at an angle ψ . Rather, when designing such systems, emphasis should be placed on the fact that the magnification factor should have minimal sensitivity to small changes in geometry. A typical geometry is shown in Figures 2-4 and 2-7, which is representative of the tested scissor-jack-damper system (Section 3). Practical values of the magnification factor lie in the range 2-5.

In the laboratory, it was possible to configure the scissor-braces within 0.2° of accuracy, and the inclination of the braces was measured frequently. Changes of 0.2° to 0.8° from the original geometry (see Figure 3-1) due to movement in the joints and supports of the model (slippage, distortion), and some inelastic action during repeated testing were observed. It is therefore appropriate to consider changes of $\pm 0.5^{\circ}$ in the angles θ and ψ , when designing scissor-brace systems and assessing their sensitivity.

The forces that act on the scissor-jack and on the single-story frame are also shown in Figure 2-6. It should be noted that the frame shown is a mechanism such that force F represents the component of the inertia force that is balanced by forces from the damping system. Considering equilibrium in the original, undeformed configuration reveals the forces that develop in the scissor-braces as

$$T = \frac{1}{2 \cdot \sin\theta} \cdot F_D \tag{2-12}$$

where T and F_D denote the forces in the brace and damper, respectively. Note that the forces T



Figure 2-7 Dependency of Magnification Factor on Scissor-Jack Geometry

are greater than the force F_D by a factor of $1/2 \cdot \sin\theta$ due to the shallow truss configuration of the scissor-braces. The resultant of the horizontal component of forces *T* equals force *F*, that is,

$$F = 2 \cdot T \cdot \cos\theta \cdot \cos\psi \tag{2-13}$$

Equation (2-13) together with (2-12), result in (2-2). That is, the magnification factor can be written as (see eqns. 2-6 and 2-11)

$$f = \frac{u_D}{u} = \frac{F}{F_D} \tag{2-14}$$

which proves the accuracy of the analysis presented.

2.4 Analysis of Motion for Large Rotations

The movement of the scissor-jack-damper system for large rotation of the braces can be described using eqns. (2-9) and (2-10). Figure 2-8 presents a comparison of the large rotation (eq. 2-9) and the simplified (eqns. 2-1 and 2-11) relations between the damper displacement and

the lateral displacement of the frame, for a geometry that is representative of the tested frame at prototype (full) scale. It must be noted that the large rotation relation requires only the brace length, \mathcal{I}_{I} , to be known. Additional dimensions (\mathcal{I} , h) are shown on Figure 2-8 to illustrate what geometry might be used in actual applications.

It is observed that the small rotation theory overpredicts the damper displacement for drift towards the right (positive) and underpredicts the damper displacement for drift towards the left. This phenomenon can be explained by the changes in the geometry of the scissor-jack: for movement towards the right, both angles ψ and θ increase, causing a decrease in the instantaneous magnification factor, which is captured by large rotation analysis. The opposite is true for movement towards the left. The lack of symmetry observed in the damper displacement for positive and negative directions of drift when the large rotation relation is used, is a result of this behavior (see Figures 2-7 and 2-8).



Figure 2-8 Relation between Damper Displacement and Lateral Displacement
2.5 Effect of Vertical Deformations

Previously, the movement of the scissor-jack-damper system was analyzed for lateral interstory drift u only. However, the vertical displacements in the frame can also affect the damper deformation in many cases (see Section 2.3). Herein, analysis of the scissor-jack movement (Section 2.3) is revisited, taking into account the effect of vertical deflections of the frame on the damper deformation, and eqns. (2-8) to (2-10) are modified accordingly.

Figure 2-9 presents the deformed configuration of the single-story frame previously depicted in Figure 2-6, inclusive of vertical deformations. Let v denote the vertical displacement of point C (point of attachment of the scissor-jack to the frame). The lateral interstory drift is denoted by u, consistent with the previous notation of Figure 2-6. Assuming all previous sign conventions remain unchanged (i.e., drift towards the right and damper extension are taken positive) and taking downward v as positive, the inclusion of v can easily be incorporated in (2-8) as

$$2 \cdot \ell_1 \cdot \cos(\mathbf{q} \pm \mathbf{D}\mathbf{q}) = 2 \cdot \ell_1 \cdot \cos\mathbf{q} \mp u \cdot \cos\mathbf{y} + v \cdot \sin\mathbf{y}$$
(2-15)



Figure 2-9 Analysis of Scissor-Jack Movement under Horizontal and Vertical Displacements

In light of eqns. (2-7) and (2-15), the damper displacement u_D and the angle of rotation of the scissor-braces *Dq* can be written as (see eqns. 2-9 and 2-10)

$$u_D = \pm 2 \cdot \ell_1 \cdot \left[\sin \left(\cos^{-1} \left\{ \cos \boldsymbol{q} + \frac{\boldsymbol{u} \cdot \cos \boldsymbol{y} + \boldsymbol{v} \cdot \sin \boldsymbol{y} \cdot \boldsymbol{y}}{2 \cdot \ell_1} \right\} \right) - \sin \boldsymbol{q} \right]$$
(2-16)

$$\pm Dq = \cos^{-1} \left(\cos q \stackrel{\checkmark}{\mp} \frac{u \cdot \cos y + v \cdot \sin y}{2 \cdot \ell_1} \right) - q$$
(2-17)

Since Dq is very small for most applications, eqns. (2-7) and (2-15) may be simplified to give the damper deformation as

$$u_D = u \cdot \frac{\cos \mathbf{y}}{\tan \mathbf{q}} + v \cdot \frac{\sin \mathbf{y}}{\tan \mathbf{q}}$$
(2-18)

It must be noted that in the single-story frame of Figure 2-9, the vertical displacement v is shown to be caused by the deformation of the beam only. However, the change of length of the columns may also contribute to the vertical displacement experienced by the damping system.

The vertical displacement v may further be written as a function of the lateral displacement u in the form

$$v = a \cdot u \tag{2-19}$$

where *a* denotes a constant. The magnification factor can then be written as

$$f = \frac{u_D}{u} = \frac{\cos \mathbf{y}}{\tan \mathbf{q}} \cdot (1 + a \cdot \tan \mathbf{y}) \tag{2-20}$$

Given the constant *a*, it is possible to calculate the effect of *v* on the magnification factor using eq. (2-20). Analysis of the tested frame exclusive of the scissor-jack system resulted in $a \approx \pm 0.1$, where the positive sign corresponds to the rigid-simple beam-to-column configuration (rigid connection on the left and simple connection on the right), which enhances the damper deformation for each direction of drift, and the negative sign represents the simple-rigid beam-to-column configuration. It must be noted that even when the damper force is zero (static conditions), forces develop in the scissor-braces due to friction at the damper-to-brace connections (Section 3), causing |a| to be less than the above quoted value. Figure 2-10 presents

the dependency of the magnification factor on angles θ and ψ (calculated using eq. 2-20) for $a = \pm 0.1$, superposed upon the results of Figure 2-7 (v = 0).

It is clear from eqns. (2-18) to (2-20), and from Figure (2-10) that calculation of u_D using (2-1), where *f* is given by (2-11), may overestimate or underestimate u_D (and the related forces and the damping ratio). It will be seen, however, that the use of the theory of the scissor-jack based on lateral drift *u* only (Sections 2.2 and 2.3) in the application of simplified analysis methods to the tested structure (Section 6) provides results that are in good agreement with the experimental results. This is because in the rigid-simple beam-to-column configuration (for which the analysis was performed), the increase in the magnification factor due to the vertical deformations is offset by the decrease in the magnification factor due to the forces in the damper and in the scissor-braces, so that *f* given by (2-11) provides satisfactory estimates.

2.6 Effect of Energy Dissipation Assembly Flexibility

The theory and analysis of the scissor-jack-damper system presented in Sections 2.2 and 2.3 are based on the assumption of an infinitely stiff energy dissipation assembly. In general however, the damping system exhibits viscoelastic behavior depending on the geometry, stiffness and damping of the elements comprising the assembly. In the case of a linear viscous damper with a damper coefficient C_o , the energy dissipation assembly can be represented by a spring element in series with the viscous damper. For a multi-story structure, the behavior is then best described by the Maxwell viscoelastic model for which, the horizontal force F_j exerted on the frame at story *j* by the energy dissipation system is described by

$$F_j + \tau_j \cdot \dot{F}_j = C_{oj} \cdot f_j^2 \cdot \dot{u}_j \tag{2-21}$$

where \dot{u}_j is the interstory drift velocity at story j, τ_j is the relaxation time which is the ratio of the damping coefficient to the stiffness of the damping assembly at story j, and f_j is the magnification factor at story j. It is of importance to emphasize that the energy dissipation assembly includes, in addition to the damper-bracing assembly, the frame to which all these are connected. Accordingly, the spring component represents the stiffness resulting from a combined action of the braces, damper, and the frame element. For example, significant



Figure 2-10 Dependency of Magnification Factor on Scissor-Jack Geometry with and without Effect of Vertical Deformations (a = 0.1, top, and a = -0.1, bottom)

flexibility will result from frame deformations in structures with scissor-jack-damper systems when installed as shown in Figure 2-4. This is also true for structures with toggle-brace-damper systems as demonstrated by Constantinou et al. (2001). In fact, the development of the reverse toggle-brace-damper system has been motivated by a desire to minimize the flexibility of the energy dissipation assembly.

The force F may be alternatively, in a further simplification, described using the Kelvin viscoelastic model as a function of relative displacement u, and relative velocity \dot{u} , as

$$F_j = k'_j(\omega) \cdot f_j^2 \cdot u_j + c'_j(\omega) \cdot f_j^2 \cdot \dot{u}_j$$
(2-22)

where k'_{j} and c'_{j} are, respectively, the storage stiffness and damping coefficient of the energy dissipation system at story *j*, which are given by

$$k'_{j}(\omega) = \frac{C_{o_{j}} \cdot \tau_{j} \cdot \omega^{2}}{1 + \tau_{j}^{2} \cdot \omega^{2}} \quad \text{and} \quad c'_{j}(\omega) = \frac{C_{o_{j}}}{1 + \tau_{j}^{2} \cdot \omega^{2}}$$
(2-23)

and ω is the frequency of free vibration of the damped structure. The ramifications of the change in the damping force *F*, from (2-4) to (2-22) are:

- a. Introduction of additional lateral stiffness to the frame given by k'
- b. Modification (decrease) of the damping coefficient of the frame from C_o to c'
- c. Change of the phase angle between the frame damping force, *F*, and the lateral frame displacement, *u*, from 90° to Φ , where

$$\tan \Phi_j = \frac{1}{\tau_j \cdot \omega} \tag{2-24}$$

It should be noted that for infinitely stiff bracing ($\tau_j = 0$), the energy dissipation system behaves as a pure viscous system, and for a single-story structure, eqns. (2-21) to (2-23) reduce to (2-4).

The effect of the flexibility of the energy dissipation assembly on the behavior of the frame will be apparent in the test results presented in Section 4, and will be further studied in the analysis of the tested structure with simplified (approximate) methods (Section 6). The interested reader is also referred to Hanson and Soong (2001) for a comprehensive treatment of this issue.

SECTION 3

EXPERIMENTAL PROGRAM

3.1 Description of Tested Structure

The scissor-jack-damper system was first tested in a frame under an imposed displacement history on the strong floor, and then in a model structure on the earthquake simulator. The model structure was a half-length scale steel frame, which was previously designed and utilized for testing of the toggle-brace-damper system (Hammel 1997, Constantinou et al. 1997, and Constantinou et al. 2001). It consisted of two identical frames that could be tested individually on the strong floor, or together on the earthquake simulator with an added mass on top of the frames. Figure 3-1 illustrates one of the two tested frames with the scissor-jack-damper system. A view of the structure on the earthquake simulator is presented in Figure 3-2. The model features the following characteristics:

- Beam-to-column connections of the model frames were easily convertible from simple to rigid. This enabled testing with one rigid and one simple connection per frame (referred to as rigid-simple or simple-rigid configurations) and with two rigid configurations per frame (rigid-rigid configuration). The rigid-simple, simple-rigid and rigid-rigid configurations were tested in the strong floor experiments, whereas the earthquake-simulator testing included only rigid-simple and simple-rigid configurations.
- 2. The scissor-braces were connected to the frame (scissors-to-beam and scissors-to-column connections) utilizing plates, which were designed to undergo mainly rotation. As shown in the detail of Figure 3-1 and in Figure 3-3, these plates were designed with sufficient length to prevent inelastic action. The damper-to-brace connections were designed as true pins to avoid transfer of bending forces to the damper. However, this was not fully accomplished because of the tight pin configuration that exhibited considerable friction. This could be avoided by the use of spherical bushings at both ends of the damper. Figure 3-4 illustrates a close-up of the damper-to-brace connection during installation.
- 3. The concrete weight used for earthquake-simulator testing comprised of two blocks weighing a total of 142.3 kN, and was secured atop the columns by way of simple



Figure 3-1 Tested Scissor-Jack-Damper Configuration



Figure 3-2 Model with Scissor-Jack Damping System on Buffalo Earthquake Simulator





Figure 3-3 Connection Details of Scissor-Braces to Frame, Scissors-to-Beam (*top*), and Scissors-to-Column (*bottom*) Connection Details



Figure 3-4 Close-up of Damper-to-Brace Connection Detail

connections (see Figures 3-1 and 3-2). The center of mass of these blocks was 1,113 mm above the centerline of the beam.

Further details on the test model, including manufacturing drawings, are presented in Appendix A.

It should be noted that for lateral loading (seismic), the model tested on the earthquake simulator with one rigid and one simple beam-to-column connection (see Figure 3-2) is equivalent to a portal frame with two rigid beam-to-column connections and double bay length. That is, the frame is equivalent to a portal frame of 1,927 mm height and 5,080 mm bay length. It was tested at half-length scale, so the prototype has a height of 3,854 mm a bay length of 10,160 mm. This is illustrated in the sketches of Figure 3-5.

3.2 Fluid Viscous Dampers

A total of three fluid viscous dampers were utilized for the strong floor and for the earthquakesimulator testing. All three dampers were of the run-through piston rod construction (without an accumulator), which prevents changes in the fluid volume upon movement of the piston in either direction, thus the damper operates symmetrically in tension and in compression. In addition, through-rod dampers, unlike dampers with accumulators, are capable of functioning over a very wide frequency range without exhibiting stiffness.



Figure 3-5 Simplified Sketch of Tested Model and Equivalent Portal Frame of Double Bay Length

The geometry of the dampers utilized in the testing of the scissor-jack system is illustrated in Figure 3-6. These dampers were first tested under harmonic motion to extract their characteristics. Figure 3-7 illustrates the testing arrangement in which, sinusoidal motion with specified frequency and amplitude was imposed at the damper piston rod via the actuator, and the resulting reaction force was measured through the load cell. This load cell was also used in the calibration of a strain gage load cell built directly on the body of the damper, for use in the earthquake-simulator testing. Typical recorded force-displacement loops for various frequencies and amplitudes are presented in Figure 3-8. The dampers exhibit purely viscous behavior. From these loops, the peak force - peak velocity characteristics of the dampers were established by extracting the peak force at the instant of zero displacement (peak velocity), as shown in Figure 3-9 (with triangular symbols). It follows that for damper 1 (used in strong floor testing), the behavior is practically linear, which can be described by eq. (2-3) with $C_o = 25.8$ N-sec/mm for velocities up to 500 mm/sec. Dampers 2 and 3 (used in earthquake-simulator testing) could be described as practically having linear behavior for velocities up to 250 mm/sec (with $C_o = 40.0$ N-sec/mm in eq. 2-3). Over a wider range of velocities, these dampers had nonlinear behavior which can be described via $F_D = C_{No} \cdot \dot{u}_D^{\alpha}$, where $C_{No} = 137.3 \text{ N-(sec/mm)}^{\alpha}$ and $\alpha = 0.76$.

3.3 Testing of Frame with Scissor-Jack System

One frame with the scissor-jack system (as shown in Figure 3-1) was subjected to sinusoidal displacement of various frequencies and amplitudes at its beam-to-column connection. The





Figure 3-6 View of Fluid Viscous Damper (*top*), and Illustration of Its Geometry (*bottom*)



Figure 3-7 View of Damper Test Setup



Figure 3-8 Recorded Force-Displacement Loops of Fluid Viscous Damper



Figure 3-8 (Continued) Recorded Force-Displacement Loops of Fluid Viscous Damper



Figure 3-8 (Continued) Recorded Force-Displacement Loops of Fluid Viscous Damper



Figure 3-9 Peak Force versus Peak Velocity Relations of Tested Fluid Viscous Dampers

purpose of this testing was to confirm the predictions of the scissor-jack theory described in Section 2. Alternate configurations of beam-to-column connections (rigid-simple, simple-rigid, and rigid-rigid) were tested to observe the effect of frame deformations on the magnification factor f (see Sections 2.3 and 2.5).

As illustrated in Figure 3-10, the tested frame was simply supported on a W21×50 beam that was bolted on the strong floor, and sinusoidal motion was applied through an actuator attached to the frame at one end, and to a reaction frame at the other. Frequency of the imposed displacement varied in the range of 0.01 Hz (quasi-static conditions) to 4 Hz (dynamic conditions), and the amplitudes included 6.35 mm and 8.45 mm (0.25 in and 0.33 in). Lateral displacement, or drift of the frame (displacement of the beam-to-column joint), damper deformation (relative displacement between the two ends of the damper), damper force, and lateral force (force required to impose the displacement) were recorded. The lateral force included the resisting



Figure 3-10 View of Frame during Testing under Imposed Lateral Joint Displacement on Strong Floor (Rigid-Simple Connections) force of the frame (sum of damping force and restoring force) and the inertia force. The inertia force was negligible compared to the resisting force of the frame therefore no corrections were made (peak value of the inertia force in the tests at frequency of 4 Hz was about 4-percent of the lateral force, observed for simple-rigid connections of the frame; lower values of the ratio of peak inertia force to peak lateral force were observed for rigid-simple and rigid-rigid configurations). As mentioned earlier, the strong floor tests utilized damper 1 (see Figure 3-8).

3.4 Earthquake-Simulator Testing Program

Testing of the model structure on the earthquake simulator (see Figure 3-2) consisted of identification of dynamic characteristics using white noise excitations, and of seismic tests using records of actual ground motions (Sections 4.2 and 4.3).

The ground motion records were compressed in time by a factor of $\sqrt{2}$ in accordance with the model's length scale factor of 1/2. These records were also scaled in acceleration amplitude, ranging from 10-percent to 300-percent of the actual value. Seismic tests generally included only horizontal excitations; some tests were repeated with simultaneous application of horizontal and vertical ground motion components.

A list of the ground motions used in the earthquake-simulator testing and their characteristics are presented in Table 3-1. The table provides information on the absolute maxima of acceleration, velocity, and displacement of the prototype (original) record, and the maximum factor used to scale the original records in acceleration amplitude. For example, the El Centro motion was applied in increasing scales up to one and one half times (150-percent) the actual record, that is, with a peak acceleration of 0.52g.

Figure 3-11 compares the 5-percent damped acceleration response spectra of the actual (target) ground motions with the spectra of the motions simulated by the earthquake simulator. In general, the target motions were reproduced satisfactorily, however a close examination reveals higher spectral accelerations of the simulated motions, particularly in the vicinity of the natural period of the structure (~ 0.25 sec - 0.3 sec). The difference in spectral accelerations between target and simulated motions can be attributed mainly to simulator-structure interaction, which becomes more pronounced around the resonant frequency, at which the structure, not the simulator may become the driving component in the test.

NOTATION	RECORD	PEAK ACCEL. (g)	PEAK VEL. (mm/sec)	PEAK DISPL. (mm)	MAX SCALE FACTOR*
El Centro S00E	Imperial Valley, May 18, 1940, component S00E	0.348	334.5	108.7	150
Taft N21E	Kern County, July 21, 1952 component N21E	0.156	157.2	67.1	300
Pacoima S74W	San Fernando, February 9, 1971, component S74W	1.076	568.2	108.2	50
Pacoima S16E	San Fernando, February 9, 1971, component S16E	1.171	1132.3	365.3	50
Miyagi-Ken- Oki EW	Tohuku Univ., Sendai, Japan, June 12, 1978, component EW	0.164	141.0	50.8	300
Hachinohe NS	Tokachi-Oki earthquake, Japan, May 16, 1968, component NS	0.229	357.1	118.9	150
Mexico N90W	Mexico City, September 19, 1985 SCT Building, component N90W	0.171	605.0	212.0	100
Sylmar 90	Northridge, January 17, 1994, LA Olive View HospParking Lot, component 90	0.604	769.4	152.0	100
Newhall 90	Northridge, January 17, 1994, LA County Fire Station, component 90	0.583	748.4	176.0	50
Newhall 360	Northridge, January 17, 1994, LA County Fire Station, component 360 0.589 947.0 305.0		305.0	75	
Kobe EW	Hyogo-Ken Nanbu Earthquake, Japan, January 17, 1995, JMA-Kobe, component EW	0.629	742.0	191.0	50

Table 3-1Earthquake Motions Used in Earthquake-Simulator Testing and
Characteristics in Prototype Scale (all components are horizontal)

* Used in testing as a percentage of the actual record



Figure 3-11 Response Spectra in Model Scale of Actual (Target) Ground Motions and Motions Produced by Earthquake Simulator



Figure 3-11 (Continued) Response Spectra in Model Scale of Actual (Target) Ground Motions and Motions Produced by Earthquake Simulator



Figure 3-11 (Continued) Response Spectra in Model Scale of Actual (Target) Ground Motions and Motions Produced by Earthquake Simulator



Figure 3-11 (Continued) Response Spectra in Model Scale of Actual (Target) Ground Motions and Motions Produced by Earthquake Simulator



Figure 3-11 (Continued) Response Spectra in Model Scale of Actual (Target) Ground Motions and Motions Produced by Earthquake Simulator

3.5 Instrumentation of Model Structure for Earthquake-Simulator Testing

The data acquisition system consisted of accelerometers, displacement transducers, and load cells, which measured the response of the structure as well as the motion of the earthquake simulator. The instrumentation scheme was similar to that of the previously tested toggle-brace system (Hammel 1997). A list of monitored channels and their descriptions are presented in Table 3-2. Figures 3-12 and 3-13 illustrate the locations of these channels. All measured signals were filtered using a low-pass filter with a cutoff frequency of 25 Hz in the D/A and A/D input.

CHANNEL	INSTRUMENT	NOTATION	RESPONSE MEASURED ¹	UNITS
1	/	TIME	Time	sec.
2	Accelerometer	ABEH	Base Horizontal AccelE	g
3	Accelerometer	ABWH	Base Horizontal AccelW	g
4	Accelerometer	ABSEV	Base Vertical AccelSE	g
5	Accelerometer	ABSWV	Base Vertical AccelSW	g
6	Accelerometer	ABNEV	Base Vertical AccelNE	g
7	Accelerometer	ACTE	Column Top Horiz. AccelE	g
8	Accelerometer	ACJE	Column Joint Horiz. AccelE	g
9	Accelerometer	ACTW	Column Top Horiz. AccelW	g
10	Accelerometer	ACJW	Column Joint Horiz. AccelW	g
11	Accelerometer	ACTTN	Column Top Transverse AccelN	g
12	Accelerometer	ACTTS	Column Top Transverse AccelS	g
13	Accelerometer	ACTVE	Column Top Vertical AccelE	g
14	Accelerometer	ACTVW	Column Top Vertical AccelW	g
15	Accelerometer	ACJNE	Column Joint Horiz. AccelNE	g
16	Accelerometer	ACTNE	Column Top Horiz. AccelNE	g
17	Accelerometer	ATBH	Top Block Horiz. Accel.	g
18	Displ. Transducer	DBE	Base Horiz. DisplEast	in.
19	Displ. Transducer	DBW	Base Horiz. DisplWest	in.
20	Displ. Transducer	DTE	Top Horiz. DisplEast	in.
21	Displ. Transducer	DTW	Top Horiz. DisplWest	in.
22	Load Cell	Dp_Frc_E	Damper Force-East	kips
23	Load Cell	Dp_Frc_W	Damper Force-West	kips
24	Displ. Transducer	Dp_Dsp_E	Damper DisplEast	in.
25	Displ. Transducer	Dp_Dsp_W	Damper DisplWest	in.
26 ²	Accelerometer	ALAT	Table Horiz. Accel	g
27 ²	Displ. Transducer	DLAT	Table Horiz. Displ.	in.
28 ²	Accelerometer	AVRT	Table Vertical Accel.	g
29 ²	Displ. Transducer	DVRT	Table Vertical Displ.	in.

Table 3-2List of Channels Utilized in Earthquake-Simulator Testing (refer to Figures
3-12 and 3-13 for locations)

¹ E = East, W = West, N = North, S = South, SE = South East, SW = South West, NE = North East

² Channels Used to Control Earthquake Simulator









SECTION 4

TEST RESULTS

4.1 Results of Testing of Frame under Imposed Lateral Joint Displacement

Testing of the frame with the scissor-jack system under harmonic joint displacement was briefly described in the previous section. The experimental program consisted of a series of tests on a single frame with each of the two scissor-brace assemblies. The scissor-braces were labeled as brace 1 and brace 2 for convenience. All tests utilized the same damper, labeled as damper 1. As mentioned previously, testing included various beam-to-column connections, and a range of frequencies and amplitudes of the imposed harmonic displacement (Section 3.3). A view of the test setup is shown in Figure 3-10. This section presents some of the experimental results of this testing to illustrate the key characteristics of the behavior of the frame with the scissor-jack-damper system. Other selected test results are presented in Appendix B. Test results in the appendix include the following information:

- 1. Test number (label), brace information, beam-to-column connection type, information on the amplitude and frequency of imposed motion, and date and time of test,
- 2. Graph showing the relation between the lateral force and the lateral displacement (drift) of the frame,
- 3. Graph of the damper force versus the damper displacement (i.e., deformation between the two ends of the damper),
- 4. Graph of the damper displacement versus the lateral displacement of the frame (i.e., the magnification factor).

All plots assume the following sign convention: Drift towards the right, resulting increase in damper length, and corresponding forces in the frame and the damper are taken positive.

The behavior of the frame with the scissor-jack-damper system for rigid-simple, simple-rigid and rigid-rigid configurations is presented in Figures 4-1 to 4-3, under quasi-static (0.01 Hz) and dynamic (4 Hz) conditions, and amplitude of 8.45 mm. From these figures, the magnification factors were calculated using the damper displacement – lateral displacement loops. Under quasi-static conditions for which the damper force is negligible, the definition of the magnification factor as the ratio of the peak damper force to the peak lateral displacement can

easily be applied. At higher frequencies, the viscoelastic effects due to the finite stiffness of the damping assembly become more apparent (see Section 2.6, eq. 2-23), therefore, the calculation of the magnification factor is rather complicated. The values reported herein for 4 Hz represent the ratio of the peak damper displacement to the peak lateral displacement. This is an average estimate since the absolute maxima of the damper displacement and the lateral displacement do not occur simultaneously, that is, at the point of peak damper displacement, the drift is less than its peak value, and vice versa.

The following observations can be made in the results of Figures 4-1 to 4-3:

- The rigid-simple configuration, which is shown in Figures 3-1 and 3-10, is, as expected, the most effective (see Sections 2.3 and 2.5), in terms of the value of the magnification factor and the energy dissipated per cycle in the lateral force – lateral displacement loops (energy dissipated per cycle equals the area enclosed by the lateral force – lateral displacement loop in one cycle of motion.
- 2. Regardless of the beam-to-column joint configuration, the magnification factor attains its largest value under quasi-static conditions (0.01 Hz) when the damping force is practically zero, and decreases with increasing frequency. This behavior is primarily the result of frame deformations under the action of forces in the scissor-jack system. Referring to Figure 2-6 and eqns. (2-12) and (2-13), it can be observed that although the damper forces are low (force F_D), the resultant force on the frame (resultant of forces T acting on the beam equals $F_D / tan\theta$) is large, due to the magnifying mechanism. For the tested configuration, $\theta = 9^\circ$ so that $F_D / tan\theta = 6.3 \cdot F_D$, causing deflection of the beam.
- 3. Under quasi-static conditions, the magnification factor is higher than predicted by theory (exclusive of the effect of vertical deformations) for the rigid-simple configuration, and lower for the simple-rigid configuration. For the rigid-rigid configuration, the magnification factor is close to theoretical predictions. It must be noted that for the tested frame, eq. (2-1) takes the form,

$$u_D = f' \cdot u \tag{4-1}$$

where the magnification factor is



Figure 4-1 Recorded Response of Frame with Brace 2 for Rigid-Simple Beam-to-Column Connections



Figure 4-2 Recorded Response of Frame with Brace 2 for Simple-Rigid Beam-to-Column Connections



Figure 4-3 Recorded Response of Frame with Brace 2 for Rigid-Rigid Beam-to-Column Connections

$$f' \approx f \cdot (h/h_s) \tag{4-2}$$

and h/h_s is a factor accounting for the geometry in which the vertical projection h of the scissor-jack is less than the story height h_s , where drift *u* takes place (see Figure 3-1). In this case, $h/h_s = 0.838$, f = 2.16 (eq. 2-11), and $f' \approx 1.8$. Testing under quasistatic conditions revealed $f' \approx 2.7$ for negative drift (i.e., drift towards the left, damper undergoes compression) and $f' \approx 2.2$ otherwise, for the rigid-simple configuration. For simple-rigid connections, $f' \approx 1.7$ and $f' \approx 1.4$ for damper compression and extension, respectively. The rigid-rigid configuration resulted in $f' \approx 2.0$ for drift to the left, and in $f' \approx 1.8$ for drift to the right. The difference between the predicted and the observed magnification factors is mainly due to frame deformations, as explained in Sections 2.3 and 2.5. For example, with the beam-to-column connections configured as rigid-simple, part of the damper deformation is caused by vertical deflection of the beam (for either direction of drift) - a factor not accounted for in theoretical predictions. The opposite occurs in the case of simple-rigid configuration. The rigid-rigid configuration falls in between these two cases. Calculation of the magnification factor approximately accounting for the effect of vertical deformations (eq. 2-20) yields f = 2.75 and $f' \approx 2.3$ for $a \approx 0.1$ (representative of the rigid-simple beam-to-column configuration), and f = 1.57 and $f' \approx 1.3$ for $a \approx -0.1$ (representative of the simple-rigid beam-to-column configuration), which are in closer agreement with the experimentally obtained magnification factors in comparison with the predictions of (2-11).

In addition to the effects of frame deformations on the magnification factor, for drift towards the left, as the scissor-braces close, both angles ψ and θ decrease, causing an increase in the instantaneous magnification factor (the opposite occurs for drift towards the right, see Figures 2-7 and 2-8), which cannot be captured by predictions under the assumption of small deformations (eqns. 2-11 and 2-20). The asymmetry in damper deformation and the larger value of f' under negative drift are due to this behavior.

4. Under dynamic conditions for which considerable forces develop in the scissor-jack assembly, the magnification factor attains values that are significantly lower than those

under quasi-static conditions. For example at a frequency of 4 Hz, the rigid-simple configuration results in $f' \approx 1.5$ for positive drift, and $f' \approx 1.9$ for negative drift. In case of the rigid-rigid configuration, $f' \approx 1.2$ and $f' \approx 1.4$ for positive drift and negative drift, respectively. The simple-rigid configuration, on the other hand, yields $f' \approx 0.8$ and $f' \approx 1.1$ for damper extension and compression, respectively, rendering the scissor-jack system ineffective under dynamic conditions for this frame. This is also apparent from comparison of the lateral force versus lateral displacement graphs for frequencies of 1, 2, 3 and 4 Hz in Appendix B, which indicate the small amount of energy dissipated at higher frequencies. A similar observation can also be made for the rigid-rigid configuration. This behavior is the result of frame deformations due to structural system configuration, and deformations caused by the forces in the damping system.

The substantial reduction in the values of the magnification factor with respect to those under quasi-static conditions for all three configurations is due to deformations of the energy dissipation assembly (primarily beam deflections) caused by the damping forces that develop in the damper and in the scissor-braces (see Sections 2.3 and 2.6).

5. There is considerable increase in the effective stiffness of the frame when significant damping forces develop, as is evident in the hysteresis loops of lateral force versus lateral displacement graphs in Figures 4-1 to 4-3. The source of the additional stiffness was explained in Section 2.6. For example, a 60-percent increase in effective stiffness is observed for the rigid-simple configuration, which corresponds to about 25-percent increase in frequency. This is consistent with the earthquake-simulator testing results and the predictions of simplified (approximate) methods of analysis to be presented in the following sections.

4.2 Identification of Dynamic Characteristics

The dynamic characteristics of the model (as depicted in Figure 3-2 with mass) were identified by exciting the structure on the earthquake simulator (with and without the scissor-jack-damper system) with a 0 - 25 Hz stationary band-limited white noise excitation. The peak acceleration of the excitations included 0.05g and 0.10g for the bare structure (without the damping system),

and varied from 0.05g to 0.30g for the structure with the damping system. It must be noted that the white noise excitation with a peak acceleration of 0.30g is comparable to earthquake simulator motions in acceleration amplitude (see Table 3-1). Transfer functions were then constructed as the ratio of the Fourier transform of the acceleration at the concrete mass-tocolumn joint (column-top) to the Fourier transform of the base acceleration (obtained from east frame instruments ACTE and ABEH, see Figure 3-12). Since the rigid concrete mass is simply connected to the top of the frames (see Figure 3-1), the movement of the concrete mass-tocolumn joint is effectively identical to the movement of the center of mass of the structure. Accordingly, the transfer function calculated on the basis of the description above may be used to obtain the dynamic characteristics of the model structure when it is represented as a singledegree-of-freedom system.

The model structure was identified in two different configurations of the beam-to- column connections: rigid-simple, which amplifies the magnification factor and simple-rigid, which causes an undesirable reduction of the factor. Figure 4-4 presents the amplitudes of the transfer functions of the tested frame with and without the scissor-jack-damper system. The amplitudes of the transfer functions reveal a simple relation that is characteristic of single-degree-of freedom systems. Accordingly, the frequency and damping ratio on the basis of the assumption of linear elastic and linear viscous behavior may be easily determined from the location and magnitude of the primary peak. They are presented in Table 4-1 for each of the tested configurations.

An observation to be made in the results of Table 4-1 and Figure 4-4 is the significant difference in the added damping in the two configurations of the frames, of which the origin has been previously explained. Another important observation is the significant stiffening of the structure, marked by the increase in frequency. For the rigid-simple configuration, the increase in frequency from 3.2 Hz to 4.0 Hz is significant and consistent with the approximately 60-percent increase in stiffness of the frame observed in testing under imposed displacement (Section 4.1). This increase in frequency is the result of viscoelastic behavior caused by frame and energy dissipation assembly deformations under the action of the damping forces as explained in Section 2.

In interpretation of the transfer function amplitudes and associated damping ratios, it must be noted that results of the white noise tests are rather sensitive to the excitation history the structure


Figure 4-4 Amplitude of Transfer Function of Model Structure with Rigid-Simple and Simple-Rigid Connections

Table 4-1	Identified Dynamic Characteristics of Model Structure with and without
	Scissor-Jack-Damper System

Beam-to-Column Connections	Configuration	Fundamental Frequency (Hz)	Damping Ratio
Digid Simula	No Dampers	3.2	0.028
Rigid-Simple	Scissor-Jack-Damper	4.0	0.130
Simula Diaid	No Dampers	3.2	0.029
Simple-Rigid	Scissor-Jack-Damper	3.5	0.055

was subjected to, prior to being identified. That is, white noise tests performed after a number of high-amplitude seismic tests may give slightly lower damping ratios compared to those obtained before seismic tests. In relation to this behavior, Figure 4-5 shows the amplitudes of transfer functions of the rigid-simple structure with the scissor-jack-damper system for the case of 0.30g white noise, obtained before (duplicate from Figure 4-4) and after conducting seismic tests. The figure reveals a reduction in the damping ratio after seismic tests, as implied by higher amplitude of the transfer function. The same trend was observed for lower amplitudes of white noise. The difference is mainly due to elevated temperature of the dampers from immediate prior testing, reduction in friction (at connections, supports, etc.) as a result of repetitive testing and occasional disassembly of the model, and deterioration of damper behavior due to repeated testing. It must be added that identifications of the structure with the scissor-jack-damper system (bare frame) in both configurations were performed after conducting the seismic tests (as noted in all figures of transfer function amplitudes).

Amplitudes of transfer functions of the structure with the damping system for various levels of excitations are presented in Figure 4-6, to reveal further observations. As the figure suggests, the damping ratio increases significantly with increasing level of excitation. For example, for the



Figure 4-5 Amplitude of Transfer Function of Rigid-Simple Structure with Scissor-Jack-Damper System for 0.30g White Noise, prior to and after Seismic Tests

excitation with a peak acceleration of 0.05g, the damping ratios are approximately 8.5-percent and 3.5-percent for the rigid-simple and the simple-rigid configurations, respectively. The damping ratios are 13-percent and 5.5- percent for the rigid-simple and the simple-rigid configurations, respectively, for the white noise excitation with 0.30g peak acceleration. Contributing mechanisms for this behavior include the amplitude dependence of inherent damping, the ineffectiveness of fluid dampers at very small amplitude movements, mild inelastic action in the scissor-jack system, and dependence of the magnification factor of the scissor-jack system on the amplitude of motion due to small imperfections (such as slightly oversized holes



Figure 4-6 Amplitude of Transfer Function of Model Structure with Scissor-Jack-Damper System under Various Levels of Excitation

that allow for some rigid body motion, use of plates for connecting the scissor-braces to the frame rather than utilizing spherical bushings, etc.). This latter mechanism is the major contributor to the dependency of damping ratio since lower magnification factors were recorded in earthquake-simulator testing with weak excitation (Section 4.3).

The structure was also identified without the dampers only (inclusive of the braces) in both beam-to-column configurations. The resulting transfer functions confirmed the purely viscous behavior of the dampers.

4.3 Earthquake-Simulator Testing Results

A summary of the earthquake-simulator (seismic) testing results is presented in Table 4-2. Tests are tabulated in the order in which they were conducted (white noise tests are excluded). The table contains the following:

- a. Test number.
- b. Description of seismic excitation, which includes the excitation name, component, and acceleration amplitude scale. For example, EL CENTRO S00E 50% implies that the record was component S00E of the El Centro earthquake, scaled in amplitude of acceleration to 50-percent of the actual value.
- c. Peak values of the earthquake simulator displacement, velocity and acceleration records. The peak simulator displacement was obtained from displacement transducer DBE, the peak velocity was derived from numerical differentiation of the displacement record, and the peak acceleration was obtained from accelerometer ABEH (see Table 3-2, and Figures 3-12 and 3-13).
- d. Peak frame response in terms of drift (displacement of the beam-to-column joint with respect to the column base), beam-to-column joint acceleration, damper relative displacement, and damper force. The peak frame output values are given for the east and west side frames in order to identify any torsional motion of the structure.
- e. Information as to whether the structure is tested with or without the scissor-jackdamper system.
- f. Values of the magnification factor for east and west frames, determined as the ratio of the peak damper displacement to the peak drift.

Table 4-2	Peak Response of Moo	del Str	ucture i	n Eartl	hquak	e-Simu	lator]	lesting							
Toot Mo	T. vitetion	Eartho	quake Sim	ulator			Fr	ame Pea	ık Value	S			accession of the second	Jinzoff	004:00
1 651 100	Excitation	Displ.	Veloc.	Accel.	Dr Dr	ift m)	Acc (g	el.)	Damper (mi	· Displ. n)	Damper (k)	· Force V)	Damper	Fac	tor
		(mm)	(mm/sec)	(g)	East	West	East	West	East	West	East	West		East	West
ELRSBD025	EL CENTRO S00E 25%	10.3	55.6	0.103	1.5	1.6	0.117	0.127	2.2	2.2	2.10	2.38	YES	1.5	1.7
ELRSBD050	EL CENTRO S00E 50%	20.7	112.1	0.187	3.1	3.2	0.232	0.247	4.9	4.9	4.30	4.59	YES	1.7	1.9
ELRSBD100	EL CENTRO S00E 100%	41.5	219.7	0.340	6.3	6.6	0.446	0.472	10.7	10.8	8.72	8.60	YES	1.8	2.0
ELRSBD150	EL CENTRO S00E 150%	62.4	332.7	0.573	10.0	10.7	0.659	0.656	16.0	16.9	11.02	11.72	YES	1.9	2.0
ELRSBD150.2	EL CENTRO S00E 150%	62.4	335.3	0.582	9.7	10.6	0.619	0.663	16.3	16.9	12.31	11.89	YES	1.9	2.0
ELRSBD150.3	EL CENTRO S00E 150%	62.0	334.0	0.549	9.7	10.3	0.655	0.719	16.5	17.0	12.71	12.81	YES	1.8	2.0
EVRSBD050	EL CENTRO S00E H+V 50%	20.6	110.2	0.183	3.3	3.3	0.236	0.253	5.0	4.9	3.89	4.56	YES	1.6	1.8
EVRSBD100	EL CENTRO S00E H+V 100%	41.3	221.0	0.338	6.4	6.7	0.460	0.481	10.3	10.9	8.46	8.63	YES	1.8	1.8
EVRSBD150	EL CENTRO S00E H+V 150%	62.1	329.6	0.558	10.0	10.3	0.660	0.717	16.3	16.8	13.33	12.91	YES	1.8	2.0
TARSBD075	TAFT N21E 75%	16.2	78.7	0.123	3.3	3.2	0.252	0.240	4.6	4.5	3.77	4.72	YES	1.6	1.4
TARSBD100	TAFT N21E 100%	21.9	106.0	0.156	4.2	4.1	0.316	0.306	6.0	5.9	4.93	6.01	YES	1.6	1.5
TARSBD200	TAFT N21E 200%	43.6	207.6	0.316	7.5	7.6	0.557	0.563	11.7	11.7	8.88	10.65	YES	1.7	1.7
TARSBD300	TAFT N21E 300%	65.3	315.9	0.564	10.6	10.6	0.731	0.746	16.2	16.4	12.22	13.80	YES	1.8	1.7
TARSBD300.2	TAFT N21E 300%	65.3	315.0	0.553	10.6	10.7	0.726	0.752	16.0	16.4	12.08	13.80	YES	1.8	1.7
TARSBD300.3	TAFT N21E 300%	65.7	318.5	0.612	10.4	10.5	0.679	0.733	14.9	15.2	13.73	13.81	YES	1.7	1.7
TVRSBD200	TAFT N21E H+V 200%	43.8	205.7	0.330	7.9	8.0	0.554	0.576	11.2	12.9	9.87	10.35	YES	1.7	1.7
HARSBD050	HACHINOHE NS 50%	25.1	107.0	0.134	3.1	3.3	0.228	0.245	4.3	5.1	3.87	4.28	YES	1.3	1.5
HARSBD100	HACHINOHE NS 100%	50.0	212.1	0.252	5.7	6.2	0.418	0.445	8.4	9.5	7.46	7.48	YES	1.6	1.6
HARSBD150	HACHINOHE NS 150%	75.1	321.3	0.371	8.4	8.9	0.593	0.606	11.7	12.5	9.86	10.02	YES	1.6	1.7
MIRSBD100	MIYAGIKEN EW 100%	17.4	100.7	0.161	3.5	3.8	0.254	0.277	5.1	5.3	4.64	4.72	YES	1.4	1.6
MIRSBD200	MIYAGIKEN EW 200%	34.7	203.2	0.284	6.6	7.2	0.479	0.509	10.6	11.0	8.67	8.60	YES	1.6	1.6

Table 4-2 Peak Response of Model Structure in Earthquake-Simulator Te	stin
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		Displ.	Veloc.	Accel.	Dr (m	ift m)	Acc (g	sel. ()	Dampei (mi	: Displ. n)	Dampeı (kî	r Force N)	Damper	Fac	tor
		(mm)	(mm/sec)	(g)	East	West	East	West	East	West	East	West		East	West
	MIYAGIKEN EW 300%	52.4	30.8	0.456	9.6	10.7	0.662	0.704	15.7	16.2	11.81	11.30	YES	1.6	1.6
1	MEXICO CITY N90W 100%	101.1	417.5	0.199	2.8	3.6	0.189	0.195	5.8	7.0	2.48	2.31	ΧES	2.1	1.9
	PACOIMA S74W 25%	14.0	103.8	0.265	4.8	4.8	0.359	0.378	6.7	7.4	6.30	6.67	YES	1.4	1.5
	PACOIMA S74W 50%	27.9	219.4	0.486	9.0	9.1	0.672	0.705	15.0	16.4	12.26	12.29	ΧES	1.6	1.8
1	PACOIMA S16E 10%	14.8	62.6	0.092	2.3	2.3	0.172	0.178	2.6	3.1	2.76	2.87	ΧES	1.1	1.3
	PACOIMA S16E 25%	37.3	154.0	0.242	5.0	5.1	0.370	0.384	7.2	7.7	6.24	6.58	ΧES	1.4	1.5
	PACOIMA S16E 50%	74.4	307.6	0.505	9.4	9.6	0.655	0.685	15.2	15.8	11.66	11.96	ΧES	1.6	1.7
	SYLMAR 90 25%	25.4	105.1	0.149	2.9	3.0	0.208	0.232	3.6	4.2	3.55	3.76	YES	1.4	1.7
	SYLMAR 90 40%	40.7	168.0	0.223	4.5	4.5	0.317	0.345	5.9	7.1	5.43	5.66	ΧES	1.5	1.8
	SYLMAR 90 50%	50.8	209.9	0.276	5.4	5.4	0.380	0.412	7.6	8.9	69.9	6.92	ΧES	1.6	1.8
	SYLMAR 90 10%	10.0	40.6	0.067	1.3	1.3	0.096	0.098	1.3	1.5	1.36	1.54	ΧES	1.0	1.4
	SYLMAR 90 75%	76.2	317.5	0.413	7.8	7.9	0.539	0.582	12.3	13.7	9.84	9.90	XES	1.8	1.9
	SYLMAR 90 100%	101.0	423.5	0.548	10.2	10.2	0.683	0.730	16.7	18.2	13.10	12.96	YES	1.8	1.9
5	SYLMAR 90 100%	101.1	420.1	0.543	10.1	10.1	0.677	0.718	18.1	18.5	12.88	12.58	YES	2.0	1.9
	NEWHALL 360 25%	30.1	146.1	0.200	3.6	3.9	0.281	0.286	5.4	6.2	4.64	4.83	YES	1.5	1.6
	NEWHALL 360 40%	48.0	235.9	0.330	5.6	6.0	0.416	0.447	9.2	10.2	7.52	7.65	YES	1.6	1.7
	NEWHALL 360 50%	60.0	298.5	0.414	7.0	7.4	0.518	0.553	11.6	13.1	9.41	9.31	YES	1.7	1.7
	NEWHALL 360 75%	89.9	466.1	0.663	10.9	11.4	0.736	0.809	19.1	20.1	14.02	13.22	YES	1.8	1.8
	NEWHALL 90 25%	17.9	100.3	0.173	4.7	5.0	0.349	0.356	6.7	7.3	4.55	5.57	YES	1.4	1.5
	NEWHALL 90 40%	28.6	161.6	0.301	7.1	7.5	0.529	0.556	10.8	12.6	7.93	8.49	YES	1.5	1.7
	NEWHALL 90 50%	35.8	203.2	0.402	8.9	9.1	0.634	0.675	14.3	15.6	9.79	10.30	YES	1.6	1.7

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1 621 140	EXCITATION	Displ.	Veloc.	Accel.	Dri (mi	(fi n)	Acc (g	eel.)	Damper (mi	Displ. Displ.	Damper (kN	Force ()	Damper	Fac	tor
		(mm)	(mm/sec)	(g)	East	West	East	West	East	West	East	West		East	West
KORSBD025	KOBE EW 25%	17.8	144.1	0.188	4.9	5.1	0.371	0.370	6.7	8.0	6.55	6.49	YES	1.3	1.5
KORSBD040	KOBE EW 40%	28.6	232.7	0.269	7.5	<i>T.T</i>	0.541	0.571	11.1	12.7	9.95	9.56	YES	1.5	1.6
KORSBD050	KOBE EW 50%	35.7	292.1	0.327	9.1	9.3	0.643	0.689	14.5	15.9	11.85	11.33	YES	1.6	1.7
SYRSBD050.2*	SYLMAR 90 50%	50.0	209.2	0.262	5.5	5.4	0.388	0.415	7.3	8.5	5.96	6.98	YES	1.6	1.7
SYRSBD075.2*	SYLMAR 90 75%	74.9	313.7	0.382	7.8	7.8	0.542	0.578	11.6	13.1	8.63	9.79	YES	1.7	1.8
TARSBD200.2*	TAFT N21E 200%	43.1	202.9	0.315	7.9	7.9	0.565	0.593	11.7	12.3	9.81	11.05	YES	1.7	1.6
ELRSBD100.2*	EL CENTRO S00E 100%	40.7	215.9	0.320	6.8	7.0	0.469	0.486	10.8	11.4	7.54	8.40	YES	1.6	1.8
ELRSNN025	EL CENTRO S00E 25%	10.0	59.7	0.107	6.1	6.1	0.286	0.292	I	I	I	I	ON	I	I
ELRSNN050	EL CENTRO S00E 50%	20.3	115.6	0.201	13.3	13.0	0.563	0.574	I	Ι	I	I	ON	Ι	I
TARSNN100	TAFT N21E 100%	21.8	103.5	0.156	7.4	7.2	0.329	0.342	Ι	Ι	Ι	Ι	ON	Ι	Ι
$TARSNN100.2^{\#}$	TAFT N21E 100%	21.8	104.8	0.154	7.1	7.1	0.322	0.335	Ι	Ι	I	I	ON	Ι	Ι
ELRSNN050.2 [#]	EL CENTRO S00E 50%	20.2	115.6	0.200	14.5	14.1	0.604	0.607	I	Ι	I	I	ON	I	I
ELRSNN025.2 [#]	EL CENTRO S00E 25%	10.1	61.0	0.112	7.4	7.2	0.337	0.332	Ι	Ι	Ι	Ι	ON	Ι	Ι
MIRSNN100	MIYAGIKEN EW 100%	17.3	94.6	0.146	10.6	10.7	0.426	0.475	Ι	Ι	I	I	ON	Ι	I
MXRSNN100	MEXICO CITY N90W 100%	100.1	412.8	0.196	6.4	7.2	0.304	0.338	Ι	Ι	I	I	ON	Ι	I
SYRSNN025	SYLMAR 90 25%	24.9	115.6	0.165	8.6	8.5	0.377	0.393	Ι	Ι	I	I	ON	Ι	I
HARSNN050	HACHINOHE NS 50%	25.1	108.0	0.127	8.3	8.6	0.349	0.407	Ι	Ι	I	I	NO	Ι	Ι
N9RSNN025	NEWHALL 90 25%	17.6	105.4	0.171	7.2	7.1	0.318	0.312	Ι	Ι	I	I	NO	Ι	Ι
N3RSNN025	NEWHALL 360 25%	29.6	153.7	0.188	10.1	9.8	0.438	0.462	Ι	Ι	I	I	NO	Ι	Ι
* Tests conducted	d to check the behavior prior to ide	entification	of the struct	ure in simp	le-rigid co	onfiguratic	u								

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Lests conducted to check the behavior prior to identification of the structure in simple-rigid

Tests repeated due to misplaced accelerometer ABNEV

All tabulated peak values represent the maximum absolute value for either the positive or negative direction of drift.

It should be noted that seismic testing was conducted with the effective rigid-simple frame configuration only (identification tests included both rigid-simple and simple-rigid configurations). As mentioned earlier, the scissor-jack systems included damper 3 (and brace 1) on the east frame, and damper 2 (and brace 2) on the west frame (see Fig. 3-9). Also, the frame that was tested on the strong floor was used as the east frame in the earthquake simulator tests.

The results of Table 4-2 demonstrate that the scissor-jack system operates as an effective damping system. Drift is substantially reduced. As an example, consider the results of the El Centro motion: without the scissor-jack system, the structure (which is essentially elastic with damping ratio of 0.03) undergoes drift of 14.3 mm (average of east and west frames, test ELRSNN050.2). The damped structure undergoes less than half that drift when excited by the full-scale El Centro motion (test ELRSBD100). Interestingly in this case of elastic response, the peak-recorded acceleration is also smaller in the damped structure despite the stronger input. The responses of the undamped and damped structures in the Mexico City motion reveal similar observations (tests MXRSBD100 and MXRSNN100, respectively). For the same input, the damped structure undergoes 50-percent lesser drift and 40-percent lesser acceleration.

Measured values of the magnification factor for east and west frames vary in the range of 1.3 to 2.1 and are dependent on the excitation type and amplitude. As noted earlier, the magnification factor is also dependent on the direction of movement due to changes in the geometry of the scissor-jack system. In comparison to the measured values, the theoretical value (exclusive of the vertical deformation effects) is 1.8, whereas values measured in the testing of the frame in rigid-simple configuration under imposed harmonic displacement varied between 1.5 and 1.9. It must be noted that lower magnification factors of 1.0 and 1.1 were observed (tests PERSBD010 and SYRSBD010) as tabulated in Table 4-2, when the structure was subjected to earthquake motions of small amplitudes. While the lower magnification factors explain the reduction in damping ratio in low amplitude excitations (see previous discussion in conjunction with Figure 4-6), they also shed light on the origin of this phenomenon. As discussed earlier in Section 4.2, the lower magnification factors are likely caused by small imperfections in the scissor-jack system.

As mentioned previously, some of the tests were conducted with simultaneous application of horizontal and vertical components of the ground motions. The results of Table 4-2 show that the influence of the vertical component is generally trivial. This is in part because the gravity load on the structure was applied directly to the columns and not on the beams (see Figures 3-1 and 3-2), therefore vertical ground excitations could not cause significant vibration of the beams, which in turn could affect the scissor-jack system performance.

Table 4-2 also reveals that there is slight torsional response of the structure, apparent by the higher peak response parameters of the west frame. The reason for this torsional behavior may be due to differences in the stiffness and damping properties of the two frames and the scissor-jack-damper assemblies, although the frames and the damping systems were designed to be identical. In addition, the control of the earthquake simulator might have been imperfect, causing unwanted torsional motion, which may also have contributed to the torsional response of the structure.

Based on the results of Table 4-2, Figure 4-7 was developed, which presents a comparison of the performances of the structure without and with the damping system. The recorded peak drift ratio (peak drift divided by the height of the beam-to-column joint), normalized peak structural acceleration (peak acceleration of the beam-to-column joint divided by the peak simulator acceleration), and normalized peak damper force (peak damper force divided by the tributary weight per frame) are plotted against the peak simulator acceleration. The peak response quantities of drift, acceleration and damper force in Figure 4-7 are the greater of the east and west frame peak values from Table 4-2. Note the use of the peak simulator acceleration as representative of the intensity of the seismic excitation, given the low period of the tested model. The benefits offered by the damping system are clearly evident in this figure: lower drift and lower acceleration for a given intensity of seismic excitation. These benefits are typical of what damping systems may offer, thus Figure 4-7 demonstrates the equivalence of the scissor- jackdamper configuration to more conventional configurations. In addition, the normalized peak damper force plot suggests that the required peak damper force as portion of the tributary weight in the scissor-jack-damper configuration is smaller in comparison to other tested damper configurations (e.g., Constantinou and Symans 1992, Seleemah and Constantinou 1997).

Of interest is to discuss the effect of the stiffening of the structure on the reduction of response.



Figure 4-7 Peak Response of Model Structure as Function of Peak Earthquake Simulator Acceleration (Rigid-Simple Configuration)

The reduction in displacement response is certainly the combined result of stiffening of the structure and of increased damping. However, the reduction of acceleration response is primarily the result of increased damping. It should be noted that the model structure is stiff with a fundamental period that falls within the acceleration-sensitive region of the spectrum for all of the earthquake motions used in the testing (see Figure 3-11 and Table 4-1), for which, reductions in period (stiffening) do not result in reduction of acceleration.

Detailed results of each test are presented in Appendix C in the form of histories of drift and acceleration, and damper force versus damper deformation loops. In accordance with previous sign conventions, drift towards the right and associated damper extension and damper force are taken positive.

SECTION 5

ANALYTICAL PREDICTION OF RESPONSE-HISTORY

5.1 Introduction

Dynamic analysis of the tested structure was performed using the computer program SAP2000 (Computers and Structures, Inc. 2000 and 2003). Two different solution methods were used; nonlinear modal time-history analysis (also called fast nonlinear analysis, or FNA), and nonlinear direct-integration time-history analysis. The latter became available in the latest version (at the time of writing of this report) of SAP2000 (version 8). Initially, the model was created with version 7.4 and analyzed using FNA. While detailed information on the two procedures is available in the program manuals, a brief description of each will be presented herein.

The FNA is suitable and very efficient for analyzing structures that are primarily linear elastic, but may have a number of predefined nonlinear elements (that is, geometric nonlinearities are not taken into account). This analysis option is thus expected to replicate the dynamic behavior of the structure under the assumption of small deformations. The method involves solution of uncoupled modal equations and modal superposition with the use of Load Dependent Ritz (LDR) vectors, which include the effects of nonlinear forces. On the other hand, the nonlinear direct-integration time-history analysis utilizes direct integration of the full equations of motion, in which, material and geometric nonlinearities may be included.

5.2 Analytical Model

Due to symmetry of the structure, only one of the two frames was modeled. Figure 5-1 illustrates the 2-dimensional model used to simulate the behavior of the structure under earthquake excitation. Masses, calculated from the added concrete blocks and tributary lengths of the elements, were lumped at the joints, as shown in Figure 5-2. Joint coordinates including lumped masses, and a summary of member properties are listed in Tables 5-1 and 5-2, respectively. In Figure 5-1, single lines denote frame elements (beam, columns and braces), and triple lines represent rigid elements (frame elements with large section properties), which were



Figure 5-1 Schematic Illustrating Joints and Elements in SAP2000 Model of Frame with Rigid-Simple Connections (see Tables 5-1 and 5-2 for joint coordinates and member properties)



Figure 5-2 Schematic Illustrating Lumped Weights in SAP2000 Model of Frame (1 lb = 4.45 N)

used to model the behavior of the concrete blocks. Joints and frame elements are numbered as they appear in the sample input file, which is included in Appendix D.

The connections of the columns to the base plates were modeled as pins with rotational springs, to simulate their semi-rigid behavior. The connection at the interface between the beam and the right column was also modeled as a pin with a rotational spring. The values of these rotational springs were assigned such that the calculated fundamental frequency of the analytical model

Joint No	X (in)	Z (in)	Mass (kips×sec ² /in)	Joint No	X (in)	Z (in)	Mass (kips×sec ² /in)
1	0	0	_	13	76.54	63.13	_
2	0	75.88	4.61×10 ⁻⁴	14	77.52	39.52	7.94×10 ⁻⁵
3	0	91.94	1.34×10 ⁻³	15	87.59	43.18	7.94×10 ⁻⁵
4	70.99	75.88	2.29×10 ⁻⁴	16	89.02	18.77	—
5	96.04	75.88	_	17	92.12	19.90	_
6	96.04	75.88	_	18	94.13	9.56	3.97×10 ⁻⁵
7	100	91.94	1.34×10 ⁻³	19	0	117.74	9.71×10 ⁻³
8	100	75.88	3.05×10 ⁻⁴	20	50	117.74	1.94×10 ⁻²
9	100	9.56	_	21	100	117.74	9.71×10 ⁻³
10	100	0	—	22	77.52	39.52	—
11	70.99	73.14	3.97×10 ⁻⁵	23	87.59	43.18	_
12	73.17	61.90	_				

Table 5-1Joint Coordinates and Lumped Masses in SAP2000 Model (1 in = 25.4 mm)

matched that obtained from testing. Rotational springs were also introduced at the joints where the scissor-braces and the damper were connected. These connections were not true pins so that they exhibited a finite amount of fixity. (Rotational springs are not needed when spherical bushings are used – a situation most likely to occur in applications of the technology.) It must be noted that in version 8, it is also possible to use **frame releases** with **partial fixity** in lieu of the rotational springs at the beam-to-column interface and damper-brace connections.

The dampers were modeled as nonlinear viscous elements, using the Nllink element (Link element in versions 8 of SAP2000), damper property, with force-velocity relation given by $F_D = C_{No} \cdot \dot{u}_D^{\alpha}$ with parameters $C_{No} = 137.3 \text{ N-(sec/mm)}^{\alpha}$, and $\alpha = 0.76$ to simulate their behavior for a large range of damper velocities (see Figure 3-9). Beam-to-column joints were modeled using end offsets and rigid-end factors, to represent the rigid zone at the connections (shown with thick single lines in Figure 5-1).

As seen in Figure 5-1, no elements were explicitly defined between joints 4 and 11, and between joints 18 and 9. Instead, each pair of these joints was constrained in the two translational (X and Z), and one rotational (Y) degrees of freedom. **Constraints** in translational degrees of freedom

Element	Joint i	Joint j	Section	Area (in ²)	I_{y-y} (in ⁴)	Shear Area (in ²)
1	1	2	W8×24	7.08	82.8	1.94
2	2	3	W8×24	7.08	82.8	1.94
3	2	4	W8×21	6.16	75.3	2.07
4	4	5	W8×21	6.16	75.3	2.07
5	6	8	W8×24	7.08	82.8	1.94
6	7	8	W8×24	7.08	82.8	1.94
7	8	9	W8×24	7.08	82.8	1.94
8	9	10	W8×24	7.08	82.8	1.94
9	11	12	PLATE	1	0.0052	0.83
10	11	13	PLATE	1	0.0052	0.83
11	12	14	$TS2 \times 2 \times 1/4$	1.59	0.766	1
12	13	15	$TS2 \times 2 \times 1/4$	1.59	0.766	1
13	22	16	TS2×2×1/4 1.59 0.766 1		1	
14	23	17	TS2×2×1/4 1.59 0.766 1		1	
15	16	18	PLATE	1	0.0052	0.83
16	17	18	PLATE	1	0.0052	0.83
17	3	19	RIGID	100	1000	0 *
18	19	20	RIGID	100	1000	0 *
19	20	21	RIGID	100	1000	0 *
20	21	7	RIGID	100	1000	0 *
SPRING	1	_	Kr	$_{\rm ot} = 45000$ k	kips×in/radia	n
SPRING	10	_	Kr	$_{\rm ot} = 45000$ k	kips×in/radia	n
NLLINK **	14	15	C = 0.3	6 kips-(sec	$(in)^{\alpha}$, and α	= 0.76
NLLINK **	5	6	K	$r_{rot} = 1660 \text{ k}$	ips×in/radiar	1
NLLINK **	14	22	K	$r_{\rm rot} = 1000 \ {\rm k}$	ips×in/radiar	1
NLLINK **	15	23	K	$r_{\rm rot} = 1000 \ {\rm k}$	ips×in/radiar	1

TABLE 5-2Element Properties in SAP2000 Model (1 in = 25.4 mm, 1 kip = 4.45 kN)

* A zero shear area is interpreted, as being infinite in SAP2000, such that the corresponding shear deformation is zero.

** Nllink elements are referred to as Link in SAP2000 version 8.

were also used at coincident joint pairs (i.e., joints 5-6, 14-22, and 15-23), where zero-length link elements provided for rotational stiffness. Additional details regarding the model may be found in the input file provided in Appendix D.

It must be noted that the text input file in Appendix D was created for SAP2000 version 7.4 (and uses FNA method), and it must be translated to run in version 8. Versions earlier than 8 support text input files besides graphical input, whereas version 8 is more graphical user interface (GUI) oriented. However it is possible to edit the input once the model is created. All input files, prepared for version 7.4 (with FNA analysis) and 8.2.3 (with FNA and nonlinear direct-integration time-history analysis options) can also be provided electronically.

5.3 Dynamic Response-History Analysis Results

Sample comparisons of experimental results to analytical predictions using the FNA method are presented in Figures 5-3 to 5-8. The compared responses are histories of drift (displacement of joint 2 with respect to joint 1) and of absolute acceleration of joint 2, and damper forcedisplacement loops (experimental data were obtained from east frame instruments). As evident in the figures, the analytical model is well capable of capturing significant characteristics of the behavior of the system, such as the stiffening effect (evident by matching frequency contents of the experimental and analytical response-histories) and peak values of drift and acceleration. Also, the analysis tended to slightly overestimate the damper displacements and forces in some tests, and underestimated them in others (it is easier to observe the match/mismatch in the damper output by looking at histories of damper force and damper displacement, rather than loops). The discrepancies between the experimental and the analytically predicted damper force and damper displacement are mainly due to:

- Uncertainties in the exact geometry of the scissor-braces during testing (the damper output is sensitive to changes in the geometry of the scissor-braces as seen in Figure 2-7).
- 2. Changes in the temperature of the dampers during continuous testing can cause changes in their properties.
- 3. Deformations as a result of any minor slippage (or reduction in friction) in the joints are magnified in the scissor-braces, which can affect the damper output.



Figure 5-3 Comparison of Analytical (SAP2000, FNA) and Experimental Response of Model Structure with Rigid-Simple Beam-to-Column Connections for El Centro 100% Input



Figure 5-4 Comparison of Analytical (SAP2000, FNA) and Experimental Response of Model Structure with Rigid-Simple Beam-to-Column Connections for Taft 200% Input



Figure 5-5 Comparison of Analytical (SAP2000, FNA) and Experimental Response of Model Structure with Rigid-Simple Beam-to-Column Connections for Hachinohe 100% Input



Figure 5-6 Comparison of Analytical (SAP2000, FNA) and Experimental Response of Model Structure with Rigid-Simple Beam-to-Column Connections for Sylmar 100% Input



Figure 5-7 Comparison of Analytical (SAP2000, FNA) and Experimental Response of Model Structure with Rigid-Simple Beam-to-Column Connections for Newhall 90 50% Input



Figure 5-8 Comparison of Analytical (SAP2000, FNA) and Experimental Response of Model Structure with Rigid-Simple Beam-to-Column Connections for Kobe 50% Input

4. Dependence of magnification factor on amplitude of motion due to scissor-jack imperfections.

As mentioned earlier, analysis of the model using the FNA method is representative of its behavior under the small deformation theory. Results of analysis under the assumption of small deformations are compared to large deformation analysis results in Figure 5-9, where the latter are obtained by the nonlinear direct-integration time-history analysis method. As the figure suggests, the difference between the two sets of results is insignificant, which is in agreement with the analysis presented in Figure 2-8 (note, in the figure, that the two methods start diverging at approximately ± 15 mm of lateral displacement, whereas the largest drift observed in the earthquake simulator tests with the damping system was about 11 mm, see Table 4-2).

It should be noted that the nonlinear direct-integration time-history analysis lasted over 30 hours for a 30-second portion of the El Centro motion, on an Intel Pentium computer (with two CPU's of 3.05 GHz each, and a 1 GB RAM), with and without the large displacement effects, whereas the FNA took only about 2 minutes for the full-length motion.



Figure 5-9 Comparison of Analytical Response of Model Structure by SAP2000 for El Centro 100% Input (Test ELRSBD100) Using Small and Large Deformation Theories

SECTION 6 Simplified Analysis

6.1 Introduction

The effects of adding an energy dissipation system to a structure, accounting for the flexibility of the damping assembly were explained in Section 2.6. This section presents an approximate analysis method based on energy principles, and illustrates its application on the tested structure utilizing the information given in Section 2.6.

The addition of energy dissipation devices to a building results in a nonclassically damped structure even if the structure itself has classical damping. Exact methods to determine the dynamic properties (e.g., frequencies of free vibration, mode shapes, and damping ratios) of the damped structure are available, but may become involved since they necessitate complex eigenvalue analysis. The interested reader is referred to Veletsos and Ventura (1986) for a comprehensive treatment of the complex eigenvalue problem and Constantinou and Symans (1992) for an illustration of the exact analysis for the case of linear viscous and viscoelastic fluid dampers.

Also, as explained in Section 2.6, addition of energy dissipation devices to a building causes an increase in stiffness and a reduction in period (see eq. 2-23). This phenomenon is well understood for devices with viscoelastic behavior (e.g., see Soong and Dargush 1997, Constantinou et al. 1998, and Hanson and Soong 2001).

An alternative to the exact methods is the use of approximate methods of analysis based on energy principles. Such methods are very simple to apply and, accordingly, have been utilized in analysis and design provisions and guidelines (Federal Emergency Management Agency 1997, 2000, and 2003). The approximate methods of analysis typically provide results of acceptable accuracy when complete vertical distributions of damping devices are used.

The approximate energy method starts with the assumption that the frequencies and mode shapes of the nonclassically damped structure are identical to those of the undamped structure with the added effect of storage stiffness (k'), but not of damping (c'), from the energy dissipation assembly (see eq. 2-23). This assumption implies that the added damping is proportional – that

is, the undamped mode shapes f of the structure with added stiffnesses that are due to the damping devices diagonalize the structure's damping matrix; namely the product $f^T \cdot C \cdot f$, where C is the damping matrix, is diagonal. Thus, the frequencies and mode shapes can be determined from standard eigenvalue analysis. Figure 6-1 depicts a multi-story structural system with linear viscous dampers in scissor-jack assemblies. Assuming that the structure undergoes vibration in the k^{th} mode with period T_k (or with frequency of vibration in the k^{th} mode equal to w_k), the damping ratio at the j^{th} story may be expressed as (Constantinou and Symans 1992, Federal Emergency Management Agency 1997, 2000, and 2003)

$$\boldsymbol{b}_{k} = \frac{g}{4 \cdot \boldsymbol{p}} \cdot \frac{T_{k} \cdot \sum_{j} c_{j}'(\boldsymbol{w}_{k}) \cdot f_{j}^{2} \cdot \boldsymbol{f}_{rj}^{2}}{\sum_{i} W_{i} \cdot \boldsymbol{f}_{i}^{2}}$$
(6-1)

where \mathbf{f}_{ij} is the k^{th} modal interstory drift of the j^{th} story (equal to the relative modal displacement between the ends of damper system j in the horizontal direction); W_i = lumped weight at the i^{th} floor level; \mathbf{f}_i = modal displacement of floor level i in the k^{th} mode of vibration; and c'_j is the damping coefficient of damper system j inclusive of the effect of energy dissipation assembly flexibility, given by (2-23). Summation j extends over all stories and summation i extends over all lumped weights. It must be noted that the magnification factor f_j is equal to $\cos q_j$ $(q_j = \text{inclination angle of the dampers)$ for devices installed diagonally; is unity for the case of chevron brace configuration of Figure 2-4; and $\cos \mathbf{y}_j / \tan q_j$ (refer to Figure 2-4 for angles \mathbf{y}_j and q_j) for the scissor-jack system. For a single-story structure, $\mathbf{f}_{rj} = \mathbf{f}_i = 1$, i = j = 1 and (6-1) simplifies, for the case of rigid energy dissipation assembly, to (2-5).

It can be seen that the calculation of b_k using (6-1) takes into account the lateral interstory drift only (i.e., the horizontal displacement of each floor represents a degree of freedom in Figure 6-1). However, vertical displacements (due to column rotation, axial flexibility of supporting columns and frame deformations) may also affect the damper deformation, as explained in Sections 2.3 and 2.5. In such cases, it is useful to express b_k in its general form, which is given as (Constantinou and Symans 1992, Federal Emergency Management Agency 1997 and 2000)



Figure 6-1 Structural System with Linear Dampers

$$\boldsymbol{b}_{k} = \frac{g}{4 \cdot \boldsymbol{p}} \cdot \frac{T_{k} \cdot \sum_{j} c_{j}'(\boldsymbol{w}_{k}) \cdot \boldsymbol{f}_{rjD}^{2}}{\sum_{i} W_{i} \cdot \boldsymbol{f}_{i}^{2}}$$
(6-2)

where f_{rjD} is the k^{th} modal relative displacement between the ends of the device along the axis of the device at the j^{th} story, which implicitly accounts for the effect of vertical displacements. Within the context of approximate methods of analysis (as, for example, those described in Federal Emergency Management Agency 2000), the mode shapes may be assumed or calculated for the undamped structure without the effect of the storage stiffness resulting from the viscoelastic nature of the energy dissipation assembly. Such an approximation produces acceptable results when the distribution of damping devices is complete over the building height. In such cases, it is of interest to develop a simple approximate method of estimating the periods of vibration of the damped structure. In arriving at an approximate method, one recognizes that

$$T'_{k} = 2 \cdot \boldsymbol{p} \cdot \left[\frac{\sum_{i} W_{i} \cdot \boldsymbol{f}_{i}^{2}}{g \cdot \sum_{j} K_{j} \cdot \boldsymbol{f}_{ij}^{2}} \right]^{1/2}$$
(6-3)

where T'_k is the k^{th} mode period of the undamped structure (exclusive of the damping system) and K_j is the horizontal stiffness of story *j*. Similarly, the period T_k of the structure with the damping system may be written as

$$T_{k} = 2 \cdot \boldsymbol{p} \cdot \left[\frac{\sum_{i} W_{i} \cdot \boldsymbol{f}_{i}^{2}}{g \cdot \sum_{j} (K_{j} + c'_{j} \cdot \boldsymbol{t}_{j} \cdot \boldsymbol{w}_{k}^{2} \cdot \boldsymbol{f}_{j}^{2}) \cdot \boldsymbol{f}_{rj}^{2}} \right]^{1/2}$$
(6-4)

where t_j is the relaxation time, calculated as the ratio of the damping constant to the stiffness of the damping assembly at story *j* (see Section 2.6). Equations (6-1) to (6-4) may be combined to arrive at the following relation provided that the mode shape is the same for the undamped and the damped structure:

$$T_{k} = T_{k}^{\prime} \cdot \left[1 - \frac{4 \cdot \boldsymbol{p} \cdot \boldsymbol{t} \cdot \boldsymbol{b}_{k}}{T_{k}}\right]^{1/2}$$
(6-5)

where parameter t is assumed to be the same for all stories of the structure, or t is an average representative value for all stories. Equation (6-5) is implicit in period T_k and can be solved using an iterative procedure, or by the solution of a cubic equation. However, given the approximate nature of the calculation, the following equation, representing the result of the first iteration, may be used.

$$T_{k} \approx T_{k}^{\prime} \cdot \left[1 - \frac{4 \cdot \boldsymbol{p} \cdot \boldsymbol{t} \cdot \boldsymbol{b}_{k}}{T_{k}^{\prime}}\right]^{1/2}$$
(6- 6)

6.2 Simplified Analysis of Tested Structure

Response-history analysis represents the best means of calculating the seismic response of a structure with the scissor-jack system. Illustrated in Figure 6-2(a) is the complete structural representation of the tested model that was used in the response-history analysis reported in Section 5 (only one of the two frames of Figure 3-2 was analyzed due to symmetry), with the exception that the damper is linear viscous as described by (2-3). This representation may be simplified for ease in the dynamic analysis by replacing the scissor-jack assembly by an equivalent spring and dashpot system as shown in Figure 6-2(b). It must be noted that the quantity K_a represents the stiffness of the assembly only, which is typically very large. It is determined by the procedure illustrated in Figure 6-2(b). In the calculation of stiffness, the damper is considered "locked" so that it acts as a spring with stiffness equal to that of the oil column in the damper. This stiffness is given by $A_r^2 \cdot B/V$, where A_r is the piston rod area, V is the effective volume of fluid, and B is its bulk modulus. In the case of the dampers in the tested model, the stiffness of the locked damper was represented by a steel element having a diameter of 10 mm and length equal to that of the damper.

Simplified analysis is based on the premise that a linear elastic and proportional linear viscous representation of the structural system produces estimates of the seismic response that are of acceptable accuracy. A discussion on the subject may be found in Hanson and Soong (2001); several examples of application of simplified methods of analysis and evaluation of the accuracy of the methods may be found in Ramirez et al. (2001). Herein, it is of interest to predict the fundamental period and associated damping ratio of the tested model using simplified methods of analysis described by eqns. (6-1) and (6-5).

The tested model is simple in the sense that it essentially is a single-degree-of-freedom system undergoing elastic deformations. Yet, application of the simplified methods for predicting the period and damping ratio are complicated by the effect of the deformations of each frame under the action of the damping forces, which results in a substantial increase in stiffness. Prediction of this increase in stiffness is important in the application of simplified analysis.

Figure 6-2(c) illustrates the procedure for calculating stiffness parameter K_b . As in the case of the response-history analysis, half of the structure is analyzed (due to symmetry) with the lateral degree of freedom restrained, the scissor-jack assembly disconnected from the column and a





(a) Complete Structural Representation

(b) Representation with Scissor-Jack Replaced by Equivalent Spring/Dashpot



(c) Determination of Stiffness K_b

(e) SDOF Representation using Kelvin Element

Figure 6-2 Representation of Structure for Simplified Analysis

displacement *D* applied along the axis of the scissor-jack. The force *P* needed to produce displacement *D* is calculated and the stiffness parameter is computed as $K_b = P/D$. This stiffness is simply related to the added stiffness provided by the energy dissipation assembly. Stiffness K_b was calculated to be 34.5 kN/mm (per frame).

Shown in Figure 62(d) is a single-degree-of-freedom representation of the tested model, in which, the structure is considered in its entirety (i.e., two frames, two scissor-jacks). In this representation, K_F is the lateral stiffness of the frames exclusive of the energy dissipation system,

and K_s and C_s are the stiffness and damping constant, respectively, contributed by the two scissor-jack-damper systems inclusive of the effect of interaction between each frame and the scissor-jack-damper system connected to it. These parameters are given by

$$K_{s} = 2 \cdot K_{b} \cdot \cos^{2} \mathbf{y} \cdot \left(\frac{\mathbf{f}_{r1}}{\mathbf{f}_{1}}\right)^{2} \approx 2 \cdot K_{b} \cdot \cos^{2} \mathbf{y} \cdot \left(\frac{h}{H}\right)^{2}$$
(6-7)

$$C_{s} = 2 \cdot \frac{C_{o} \cdot \cos^{2} \mathbf{y}}{\tan^{2} \mathbf{q}} \cdot \left(\frac{\mathbf{f}_{r1}}{\mathbf{f}_{1}}\right)^{2} \approx 2 \cdot \frac{C_{o} \cdot \cos^{2} \mathbf{y}}{\tan^{2} \mathbf{q}} \cdot \left(\frac{h}{H}\right)^{2}$$
(6-8)

where f_{r1} is the relative modal horizontal displacement of the two ends of the scissor-braces and f_1 is the modal displacement of the center of mass of the concrete block. The mode shape can easily be determined from the analytical model of the structure developed for response-history analysis (Section 5). One can also recognize that for the model structure, $f_{r1}/f_1 \approx h/H$, as illustrated in Figure 6-3 (also see Figure 3-1). It must be noted, in Figure 6-3, that f_1 is the same as the modal displacement of the points of connection of the concrete block to the columns (pins).

Analysis of the single-degree-of-freedom representation of Figure 62(d) is itself complicated given the viscoelastic nature of the system. For example, calculation of the period and damping



Figure 6-3 Schematic of Tested Model Showing Modal Displacements

ratio requires complex eigenvalue analysis (Constantinou and Symans 1992). Simplified analysis would require a further step of replacing the Maxwell element in Figure 6-2(d) by an equivalent Kelvin element of stiffness k' and damping constant c' (Constantinou et al. 1998) as shown in Figure 6-2(e), where (see eq. 2-23)

$$k' = \frac{C_s \cdot \boldsymbol{t} \cdot \boldsymbol{w}^2}{1 + \boldsymbol{t}^2 \cdot \boldsymbol{w}^2} \tag{6-9}$$

$$c' = \frac{C_S}{1 + \boldsymbol{t}^2 \cdot \boldsymbol{w}^2} \tag{6-10}$$

$$\boldsymbol{t} = \frac{C_s}{K_s} = \frac{C_o}{\tan^2 \boldsymbol{q} \cdot K_b}$$
(6-11)

Parameter $w = 2 \cdot p / T_1$ is the frequency of vibration of the damped structure, and parameter t is the relaxation time, which also appears in eqns. (6-5) and (6-6).

On the basis of the representations shown in Figures 6-2(e) and 6-3, the period T_1 and damping ratio **b**₁ of the damped structure are:

$$T_1 = 2 \cdot \boldsymbol{p} \cdot \left[\frac{m}{K_F + k'}\right]^{1/2} = T_1' \cdot \left[1 - \frac{4 \cdot \boldsymbol{p} \cdot \boldsymbol{t} \cdot \boldsymbol{b}_1}{T_1}\right]^{1/2}$$
(6-12)

$$\boldsymbol{b}_{1} = \frac{\boldsymbol{c}' \cdot \boldsymbol{T}_{1}}{4 \cdot \boldsymbol{p} \cdot \boldsymbol{m}} = \frac{\boldsymbol{C}_{o} \cdot \cos^{2} \boldsymbol{y} \cdot \boldsymbol{T}_{1}}{2 \cdot \boldsymbol{p} \cdot \boldsymbol{m} \cdot (1 + \boldsymbol{t}^{2} \cdot \boldsymbol{w}^{2}) \cdot \tan^{2} \boldsymbol{q}} \cdot \left(\frac{h}{H}\right)^{2}$$
(6-13)

where T'_1 is the period of vibration of the structure exclusive of the energy dissipation system given by (see eq. 6-3)

$$T_1' = 2 \cdot \boldsymbol{p} \cdot \left[\frac{m}{K_F}\right]^{1/2} \tag{6-14}$$

It must be noted that eqns. (6-12) and (6-13) are identical to eqns. (6-5) and (6-1), respectively, for the model structure with the exception that $\mathbf{f}_{r1}/\mathbf{f}_1$ appears instead of h/H. Also, the quantities f^2 and f_{r1}/f_1^2 are implicit in the calculation of c' (see eqns. 6-8 and 6-10).

In order to perform calculations, $C_o = 40.0$ N-sec/mm was used. This value represents well the behavior of the dampers for velocities less than about 250 mm/sec (see Figure 3-9). Also, m = 14.5 N-sec/mm² (W = 142.3 kN). Parameters K_S and C_S are calculated using eqns. (6-7) and (6-8). Equation (6-11) then results in t = 0.046 sec. Based on the model identification (frequency of 3.2 Hz, see Table 4-1), period $T'_1 = 0.31$ sec. Additionally, $\mathbf{f}_{r1} / \mathbf{f}_1 \approx h / H = 0.68$ and eqns. (6-12) and (6-13) are iteratively solved to result in $T_1 = 0.27$ sec and $\mathbf{b}_1 = 0.12$. One can also calculate \mathbf{w} (thus T_1) by solving (from eqns. 6-12 and 6-9)

$$K_F + \frac{C_S \cdot \boldsymbol{t} \cdot \boldsymbol{w}}{1 + \boldsymbol{t}^2 \cdot \boldsymbol{w}^2} = m \cdot \boldsymbol{w}^2$$
(6-15)

where K_F is calculated using (6-14). Note that the period of 0.27 sec corresponds to a frequency of 3.7 Hz, which is less than the identified value of 4.0 Hz (see Table 4-1) but sufficiently close for practical purposes. Also, the calculated damping ratio of 0.12 represents the added damping supplied by the damping system. Since the inherent damping was of the order of 0.03, the total damping ratio is about 0.15. Identification of the structure showed a total damping ratio of 0.13 (see Table 4-1).

As an alternative to using (6-13) to predict T_1 and \boldsymbol{b}_1 , (6-2) can be used, which takes into account the vertical deflections due to frame deformations. For the tested structure, (6-2) can be written as

$$\boldsymbol{b}_{1} = \frac{\boldsymbol{C}_{o} \cdot \boldsymbol{T}_{1}}{2 \cdot \boldsymbol{p} \cdot \boldsymbol{m} \cdot (1 + \boldsymbol{t}^{2} \cdot \boldsymbol{w}^{2})} \cdot \left(\frac{\boldsymbol{f}_{r1D}}{\boldsymbol{f}_{1}}\right)^{2}$$
(6-16)

Equation (6-16) necessitates modal analysis for calculation of \mathbf{f}_{r1D} and \mathbf{f}_1 . Using the analytical model of the structure developed for response-history analysis (Section 5), $\mathbf{f}_{r1D} / \mathbf{f}_1 = 1.82$. Iterative solution of eqns. (6-12) and (6-16) result in $T_1 = 0.25$ sec (frequency of 4 Hz) and $\mathbf{b}_1 = 0.155$. The predicted frequency is the same with that obtained from identification tests, but the damping ratio is overestimated (see Table 4-1). In general however, simplified analysis predicts the dynamic properties of the model satisfactorily.

The peak dynamic response of the tested model can be estimated by use of response spectra, for the purpose of comparison to experimental results (tabulated in Table 42). As an example, Figure 6-4 presents acceleration and displacement spectra for the El Centro motion for various damping ratios, generated using the earthquake simulator motion. It must be noted that the acceleration spectra in Figure 6-4 represent the actual maximum accelerations, not the pseudo-accelerations, which were shown in Figure 3-11. The peak response obtained from Figure 6-4 corresponds to that of the center of mass of the concrete blocks, and must be multiplied by the factor $h_s / H = 0.81$ (see Figures 3-1 and 6-2a) in order to approximately calculate the peak response of the beam-to-column joint (for comparison with the peak frame values reported in Table 4-2).

Table 6-1 presents a summary of the peak dynamic response calculated based on Figure 6-4 together with the experimental response from Table 4-2. In estimating the maximum acceleration and the maximum displacement, T = 0.27 sec and damping ratio = 0.15 were used (based on simplified calculations where the damping ratio was calculated as 0.12 and 0.03 was added for the inherent damping). The peak damper displacement u_{Dmax} was calculated from

$$u_{D\,max} = f \cdot \frac{h}{H} \cdot u_{max} \tag{6-17}$$

where u_{max} is the maximum displacement from Figure 6-4 and *f* is given by (2-11). The peak damper force F_{Dmax} was calculated using

$$F_{Dmax} = C_o \cdot \frac{2 \cdot \boldsymbol{p}}{T} \cdot \boldsymbol{u}_{Dmax} \cdot CFV \tag{6-18}$$

The quantity $(2 \cdot \mathbf{p} / T) \cdot u_{Dmax}$ in (6-18) is the damper pseudo-velocity, and *CFV* is a correction factor so that the product of *CFV* and the pseudo-velocity is a good approximation to the peak damper velocity. Values of *CFV* have been presented in Ramirez et al. (2001). They are dependent on period and damping ratio. For period of 0.27 sec and damping ratio of about 0.15, *CFV* is equal to 0.70. It can be seen in Table 6-1 that the predicted response is in good agreement with the experimentally obtained peak response.


Figure 6-4 Response Spectra for El Centro S00E (100%) at Damping Ratios of 0.05, 0.10 and 0.15

TABLE 6-1Peak Response of Tested Structure Calculated by Simplified Analysis and
Comparison to Results of Earthquake-Simulator Tests (El Centro 100 %
Input)

Peak Response Quantity	Analytical	Experimental *
Drift (mm)	7.1	6.3
Joint Acceleration (g)	0.41	0.45
Peak Damper Displacement (mm)	12.9	10.7
Peak Damper Force (kN)	8.4	8.7

* East frame response, test ELRSBD100

SECTION 7

APPLICATION OF SCISSOR-JACK-DAMPER SYSTEM

The scissor-jack-damper system has been implemented in the design of a new, 3-story building structure in Cyprus, which is currently under construction. The building will serve as the headquarters of the Cyprus Olympic Committee. The scissor-jack-damper system was determined to be advantageous with respect to other economically feasible supplemental damping systems due to open-space requirements for the structure. Figure 7-1 illustrates the plan view of the 1st floor of this building, where "S" demarks the locations of the scissor-jack-damper systems. Also included in this figure is a cross-sectional view of the building (cut along the dashed line shown in the 1st floor plan view) illustrating the scissor-jack-damper installation. Details regarding the geometry and dimensions of the damper assembly are given in Figure 7-2. The structure utilizes 52 scissor-jack-damper assemblies equipped with linear fluid viscous damping devices.

Modeling and analysis of the building was performed using SAP2000. Response-history analyses were conducted with the 1940 El Centro S00E, 1986 Kalamata-Nomarhia NS (Greece) and 1952 Taft N21E ground motions, scaled at various amplitudes in order to represent the site response spectrum. Figure 7-3 presents the average of the pseudo-acceleration spectra of the scaled motions, superposed upon the site-specific design spectrum. The analytical model of the structure resembles that of the tested scissor-jack-damper system (Section 5). Figure 7-4 illustrates the SAP2000 model, which represents a portion of the structure. Sample results from response-history analysis, using the model shown in Figure 7-4 with and without the scissor-jack-damper system are presented in Figures 7-5 and 7-6, for the controlling-scaled (scale factor 100%) Taft N21E record. The effectiveness of the damping system is evident in the reduced displacement (and slightly reduced acceleration) response of the structure with the scissor-jack-damper system.







Figure 7-2 Details of Scissor-Jack-Damper System for Building in Cyprus (typical for 2nd and 3rd floors; dimensions in mm)



Figure 7-3 Response Spectra of Scaled Motions used in Response-History Analysis



Figure 7-4 SAP2000 Model for Response-History Analysis



Figure 7-5 Displacement Response of Structure from Response History Analysis for Taft N21E 100% Input (see Figure 7-4 for joint locations)



Figure 7-6 Acceleration Response of Structure from Response History Analysis for Taft N21E 100% Input (see Figure 7-4 for joint locations)

SECTION 8

SUMMARY AND CONCLUSIONS

Scissor-jack-damper energy dissipation system offers structural systems with small drifts an opportunity to benefit from supplemental damping. Similar to the toggle-brace-damper system that preceded its development, the scissor-jack-damper system utilizes shallow trusses to magnify the damper displacement for a given interstory drift, and to magnify the damper force output delivered to the structural frame. The system thus extends the applicability of damping devices to cases of small interstory drifts, such as stiff structures under seismic loading and structures subject to wind loading. Additionally, the scissor-jack damping system can be configured to allow for open space through its compactness and near-vertical installation, a feature that is often desired for architectural purposes.

This report presented a theoretical treatment of the scissor-jack-damper system and demonstrated its utility via experimental results. Analysis of the kinematics of the scissor-jack configuration was illustrated. The equations that described the behavior were reduced to the case of small rotations to yield simple expressions for the magnification factor, which is defined as the ratio of the damper deformation to the lateral interstory drift. The effect of frame deformations on the damper deformation and the magnification factor was also presented. In addition, the effects of the flexibility of the energy dissipation assembly were discussed.

The experimental study included testing of a half-scale steel model structure equipped with a scissor-jack-damper system on the strong floor under imposed cyclic displacement and on the earthquake simulator. The strong floor tests confirmed the theoretically predicted behavior. The effect of various beam-to-column connection configurations was also demonstrated. The earthquake-simulator testing indicated a significant increase in the damping ratio, accompanied with reduction in drift and acceleration responses, in comparison with the same structure without the energy dissipation system. The scissor-jack-damper system also stiffened the structure, marked by an increase in frequency. This viscoelastic behavior occurred as a result of frame and energy dissipation assembly deformations under the action of damping forces. Testing of the scissor-jack-damper system also pointed out important issues that can affect the performance of

the system, such as sensitivity of the system's behavior to repetitive testing and small imperfections.

The response of the model structure to seismic excitation was reproduced analytically by response-history analysis using SAP2000. Two different analysis methods, namely nonlinear modal time-history analysis and nonlinear direct-integration time-history analysis, were utilized. In addition to using a different solution procedure, the modal method predicted the dynamic response of the structure under the assumption of small deformations, whereas the direct-integration method allowed for large deformations. It was shown that analysis of the structure using the small deformation theory produces results that are identical to those of the large deformation theory and that the modal method is more efficient than the direct-integration method, which requires substantial amount of computational time. The analytical model produced response-history results of acceptable accuracy and satisfactorily captured the significant characteristics of the model such as the stiffening effect, and peak values of drift and acceleration.

In addition, application of simplified analysis methods for predicting the period and damping ratio of the model structure, based on information on the dynamic characteristics of the undamped structure and the geometry of the scissor-jack-damper system, was presented. Simplified analysis requires a proper presentation of the increase in stiffness of the structure due to the damping system. Results of this analysis were in close agreement with those of the experiments. Also, the peak dynamic response of the tested structure was estimated utilizing response spectra of ground motions, which matched closely with experimental results.

An important outcome of this study is that structures with the scissor-jack-damper system can conveniently be analyzed using code-oriented procedures such as those outlined in Federal Emergency Management Agency (1997, 2000, and 2003), with the appropriate modification for the magnification factor, which relates the lateral interstory drift to the damper deformation. This is also true for structures with toggle-brace-damper systems. The simplicity of both configurations lies in the fact that the magnification factors are related to simple geometry of the systems. In cases where vertical deflections may affect the damper deformation, these equations may still be utilized with the appropriate adjustment to account for the vertical motion effects.

The effectiveness of the scissor-jack-damper system was demonstrated on a linear elastic structure. This system can also be implemented in building structures that are expected to respond beyond the elastic limit. Simplified methods for the analysis of yielding structures with energy dissipation systems are presented in Federal Emergency Management Agency (1997, 2000, and 2003). Evaluation and application of these methods can be found in Ramirez et al. (2001) and Tsopelas et al. (1997). In a yielding structure with the scissor-jack-damper configuration, the frame deformations due to structural system configuration are likely to be different than before beyond the elastic limit. Changes in the boundary conditions at the beam-to-column connections are important due to their effect on the magnification factor. Such changes must be considered. In addition, since the connection details and sensitivity to imperfections are important in ensuring magnification and proper load transfer, the components and connections in the vicinity of the scissor-jack-damper system should be designed to remain elastic for all levels of earthquake shaking.

The scissor-jack-damper system is a viable solution for supplemental damping in stiff structural systems and for reducing wind-induced vibrations. The versatility of the system is enhanced by its compact size and geometry, which allow for configurations that offer open space for architectural components. Indeed, the decision to use the scissor-jack-damper system for supplemental damping for a building in Cyprus was in the large part related to architectural considerations. The system can also be installed in various arrangements depending on damping and open space requirements. Moreover, the scissor-jack configuration can be used in conjunction with other magnifying mechanisms, such as the coupled truss and damping system used in the Torre Mayor. As with all supplemental damping mechanisms, the scissor-jack-damper system is replaceable following a major event; the fact that the scissor-jack system permits the use of standard connection details makes the post-earthquake maintenance and replacement relatively simple.

SECTION 9

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APPENDIX A

DRAWINGS OF TESTED STRUCTURE

Notes: 1. Drawings are as provided to fabricator. 2. Refer to Hammel (1997) for information on existing frame connection details.

(1 foot = 304.8 mm, 1 in = 25.4 mm)













APPENDIX B

RESULTS OF TESTING OF FRAME UNDER IMPOSED LATERAL JOINT DISPLACEMENT











F2SR03 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_{O} =6.35 mm, f=0.5 Hz (03/19/99, 10:05:54)



F2SR04 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_{O} =6.35 mm, f=1 Hz (03/19/99, 10:20:14)



F2SR05 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_{0} =6.35 mm, f=2 Hz (03/19/99, 10:22:12)



F2SR06 : BRACE 2, SIMPLE-RIGID CONNECTIONS $U_{\rm O}\text{=}6.35$ mm, f=3 Hz (03/19/99, 10:08:07)



F2SR07 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_{0} =6.35 mm, f=4 Hz (03/19/99, 10:10:34)



F2SR08 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_0=8.45 mm, f=0.01 Hz (03/19/99, 10:37:20)



F2SR09 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_o=8.47 mm, f=0.05 Hz (03/19/99, 10:34:29)


F2SR10 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_{0} =8.45 mm, f=0.5 Hz (03/19/99, 10:44:25)



F2SR11 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_{0} =8.45 mm, f=1 Hz (03/19/99, 10:50:57)



F2SR12 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_0 =8.45 mm, f=2 Hz (03/19/99, 10:52:25)







F2SR14 : BRACE 2, SIMPLE-RIGID CONNECTIONS U_=8.45 mm, f=4 Hz (03/19/99, 10:48:53)







F2RS02 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_{0} =6.35 mm, f=0.05 Hz (03/19/99, 13:54:47)







F2RS04 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_{O} =6.35 mm, f=1 Hz (03/19/99, 14:02:34)



F2RS05 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_{O} =6.35 mm, f=2 Hz (03/19/99, 14:04:20)



F2RS06 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_0 =6.35 mm, f=3 Hz (03/19/99, 14:05:53)



F2RS07 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_0 =6.35 mm, f=4 Hz (03/19/99, 14:07:51)



F2RS08 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_{O} =8.45 mm, f=0.01 Hz (03/19/99, 14:10:06)



F2RS09 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_{0} =8.45 mm, f=0.05 Hz (03/19/99, 14:16:38)



F2RS10 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_{0} =8.45 mm, f=0.5 Hz (03/19/99, 14:18:52)







F2RS12 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_{0} =8.45 mm, f=2 Hz (03/19/99, 14:21:53)



F2RS13 : BRACE 2, RIGID-SIMPLE CONNECTIONS U_{0} =8.45 mm, f=3 Hz (03/19/99, 14:23:14)









F2RR11 : BRACE 2, RIGID-RIGID CONNECTIONS U_=8.45 mm, f=1 Hz (03/26/99, 14:43:03)













F1RS04 : BRACE 1, RIGID-SIMPLE CONNECTIONS U_{O} =6.35 mm, f=1 Hz (04/05/99, 13:56:13)



F1RS06 : BRACE 1, RIGID-SIMPLE CONNECTIONS U_0 =6.35 mm, f=3 Hz (04/05/99, 14:01:38)



F1RS07 : BRACE 1, RIGID-SIMPLE CONNECTIONS $U_{\rm O}\text{=}6.35$ mm, f=4 Hz (04/05/99, 14:04:30)





APPENDIX C

RESULTS OF EARTHQUAKE-SIMULATOR TESTING

(ALL TESTS PERFORMED WITH RIGID-SIMPLE BEAM-TO-COLUMN CONNECTIONS)









DAMPER DISPL. (mm)




DAMPER DISPL. (mm)

































DAMPER DISPL. (mm)








































N9RSBD050 : NEWHALL 90 50% (05/17/99)



APPENDIX D

INPUT FILE FOR RESPONSE-HISTORY ANALYSIS OF FRAME WITH SAP2000 (USING FNA METHOD)

; scissor elcen.s2k

; SAP2000 input file for earthquake-simulator testing using fast nonlinear analysis (FNA) method.

;This file can be imported to SAP2000 version 7, or translated to run in version 8.

; ELCENTRO100%

, SYSTEM

DOF=UX,UZ,RY LENGTH=IN FORCE=Kip PAGE=SECTIONS

JOINT

```
1 X=0 Y=0 Z=0
2 X=0 Y=0 Z=75.875
3 X=0 Y=0 Z=91.9375
4 X=70.98813 Y=0 Z=75.875
5 X=96.035 Y=0 Z=75.875
6 X=96.035 Y=0 Z=75.875
7 X=100 Y=0 Z=91.9375
8 X=100 Y=0 Z=75.875
9 X=100 Y=0 Z=9.5625
 10 X=100 Y=0 Z=0
 11 X=70.98813 Y=0 Z=73.14069
 12 X=73.17303 Y=0 Z=61.90036
 13 X=76.53955 Y=0 Z=63.12567
 14 X=77.52354 Y=0 Z=39.51905
 15 X=87.59333 Y=0 Z=43.18417
 16 X=89.02393 Y=0 Z=18.77181
 17 X=92.11961 Y=0 Z=19.89855
 18 X=94.12875 Y=0 Z=9.5625
 19 X=0 Y=0 Z=117.7375
20 X=50 Y=0 Z=117.7375
21 X=100 Y=0 Z=117.7375
22 X=77.52354 Y=0 Z=39.51905
23 X=87.59333 Y=0 Z=43.18417
RESTRAINT
ADD=1 DOF=U1,U3
ADD=10 DOF=U1,U3
CONSTRAINT
NAME=EQUAL1 TYPE=EQUAL DOF=UX,UZ,RY CSYS=0
 ADD=9
 ADD=18
NAME=EQUAL2 TYPE=EQUAL DOF=UX,UZ,RY CSYS=0
 ADD=4
 ADD=11
NAME=EQUAL3 TYPE=EQUAL DOF=UX,UZ CSYS=0
 ADD=14
 ADD=22
NAME=EQUAL4 TYPE=EQUAL DOF=UX,UZ CSYS=0
 ADD=15
```

```
ADD=23
NAME=TRANS1 TYPE=EQUAL DOF=UX,UZ CSYS=0
 ADD=5
 ADD=6
PATTERN
NAME=DEFAULT
SPRING
ADD=1 R2=45000
ADD=10 R2=45000
MASS
ADD=2 U1=4.605691E-04 U3=4.605691E-04
ADD=3 U1=1.336188E-03 U3=1.336188E-03
ADD=4 U1=2.291214E-04 U3=2.291214E-04
ADD=7 U1=1.336188E-03 U3=1.336188E-03
ADD=8 U1=3.050554E-04 U3=3.050554E-04
ADD=11 U1=3.970103E-05 U3=3.970103E-05
ADD=14 U1=7.940207E-05 U3=7.940207E-05
ADD=15 U1=7.940207E-05 U3=7.940207E-05
ADD=18 U1=3.970103E-05 U3=3.970103E-05
ADD=19 U1=9.709492E-03 U3=9.709492E-03
ADD=20 U1=1.941898E-02 U3=1.941898E-02
ADD=21 U1=9.709492E-03 U3=9.709492E-03
MATERIAL
NAME=STEEL IDES=N
 T=0 E=29000 U=.3 A=0
FRAME SECTION
NAME=W8X21 MAT=STEEL A=6.16 J=.28 I=75.3,9.77 AS=2.07,3.5133
S=18.18841,3.70778 Z=20.4,5.69 R=3.496288,1.25938 T=8.28,5.27,.4,.25,5.27,.4 SHN=W8X21
DSG=W
NAME=W8X24 MAT=STEEL A=7.08 J=.35 I=82.8,18.3 AS=1.9429,4.33
S=20.88272,5.635104 Z=23.2,8.57 R=3.419783,1.607714 T=7.93,6.495,.4,.245,6.495,.4
SHN=W8X24 DSG=W
NAME=PLATE MAT=STEEL SH=R T=.25,4 A=1 J=2.001302E-02 I=5.208333E-03,1.333333
AS=.8333333..8333333
NAME=TUBE MAT=STEEL A=1.59 J=1.36 I=.766,.766 AS=1,1 S=.766,.766 Z=1,1
R=.69409,.69409 T=2,2,.25,.25,0,0 SHN=TS2X2X1/4 DSG=B
NAME=RIGID MAT=STEEL A=100 J=0 I=1000,0 AS=0,0 T=1,1
NLPROP
NAME=DAMPER TYPE=Damper
 DOF=U1 KE=0 CE=0 K=1000 C=.36 CEXP=.76
NAME=SPRING1 TYPE=Damper
 DOF=R3 KE=1660 CE=0
NAME=SPRING3 TYPE=Damper
 DOF=R3 KE=1000 CE=0
NAME=SPRING2 TYPE=Damper
```

```
221
```

DOF=R3 KE=45000 CE=0

FRAME

```
1 J=1,2 SEC=W8X24 NSEG=2 ANG=0 JOFF=12.14 RIGID=.3
2 J=2.3 SEC=W8X24 NSEG=2 ANG=0 IOFF=12.14 RIGID=.3
3 J=2,4 SEC=W8X21 NSEG=2 ANG=0 IOFF=11.965 RIGID=.3
4 J=4,5 SEC=W8X21 NSEG=2 ANG=0
5 J=6.8 SEC=W8X24 NSEG=2 ANG=0 JOFF=3.92535 RIGID=1
6 J=7.8 SEC=W8X24 NSEG=2 ANG=0 JOFF=12.14 RIGID=.3
7 J=8,9 SEC=W8X24 NSEG=2 ANG=0 IOFF=12.14 RIGID=.3
8 J=9,10 SEC=W8X24 NSEG=2 ANG=0
9 J=11,12 SEC=PLATE NSEG=2 ANG=0
10 J=11,13 SEC=PLATE NSEG=2 ANG=0
11 J=12,14 SEC=TUBE NSEG=2 ANG=0
12 J=13,15 SEC=TUBE NSEG=2 ANG=0
13 J=22,16 SEC=TUBE NSEG=2 ANG=0
14 J=23,17 SEC=TUBE NSEG=2 ANG=0
15 J=16,18 SEC=PLATE NSEG=2 ANG=0
16 J=17,18 SEC=PLATE NSEG=2 ANG=0
17 J=3,19 SEC=RIGID NSEG=2 ANG=0 IREL=R3
18 J=19,20 SEC=RIGID NSEG=2 ANG=0
19 J=20,21 SEC=RIGID NSEG=2 ANG=0
20 J=21,7 SEC=RIGID NSEG=2 ANG=0 JREL=R3
NLLINK
```

1 J=14,15 NLP=DAMPER ANG=0 2 J=5,6 NLP=SPRING1 ANG=0 AXDIR=+Z 3 J=14,22 NLP=SPRING3 ANG=0 AXDIR=+Z 4 J=15,23 NLP=SPRING3 ANG=0 AXDIR=+Z

LOAD

NAME=LOAD1 CSYS=0

MODE

TYPE=RITZ N=10 ACC=UX ACC=UZ NLLINK=*

FUNCTION

NAME=ELCEN106 DT=0.01 NPL=1 PRINT=N FILE=ELRSD106.txt ; ground motion input in a separate file.

HISTORY

```
NAME=ELRSD106 TYPE=NON NSTEP=4009 DT=.01 DAMP=.02
ACC=U1 ANG=0 FUNC=ELCEN106 SF=386.22 AT=0
```

OUTPUT

END

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